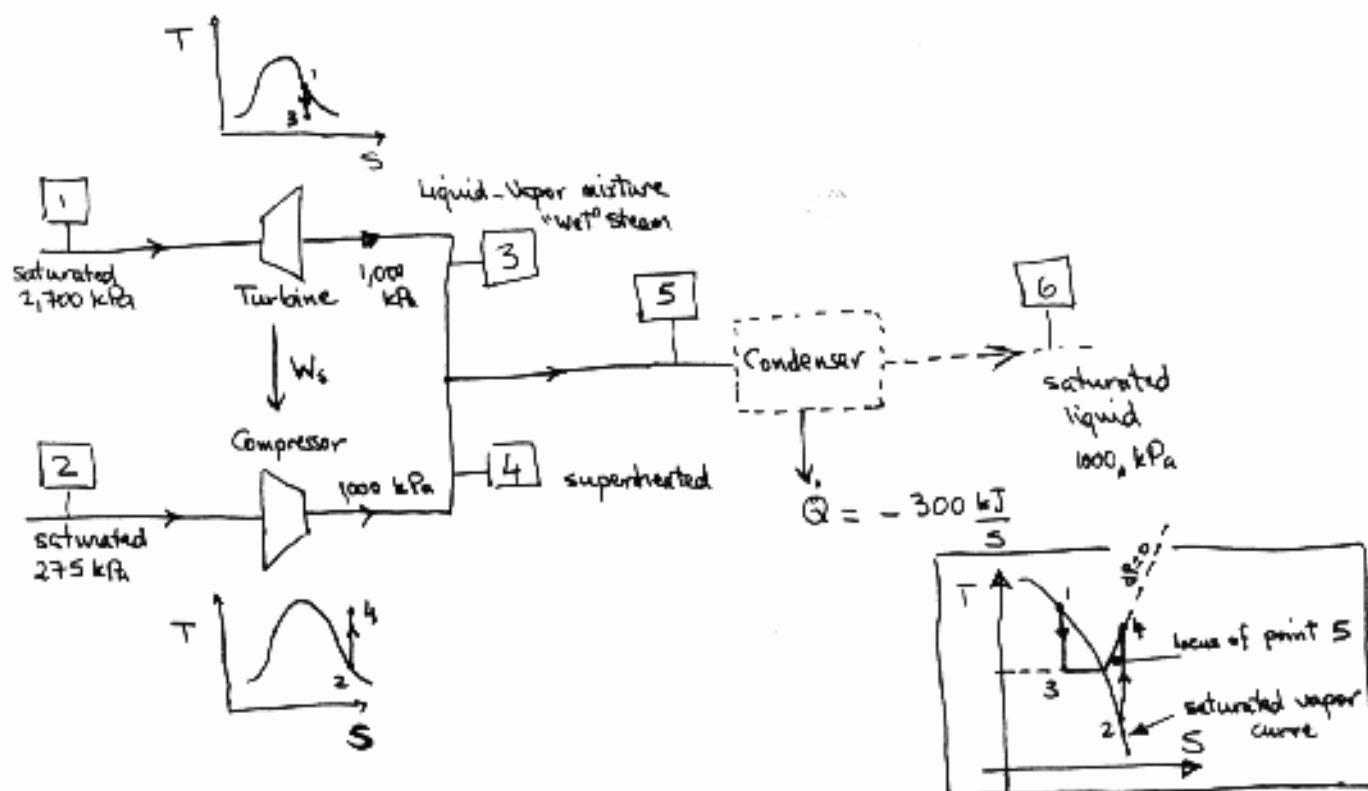


Problem 17

10.213



Stream 4 is a superheated steam and stream 3 is a wet steam (i.e. contains some liquid). Stream 5 can either be superheated, saturated or wet. That depends on the relative amounts of streams 3 and 4.

- a) To solve for \dot{m}_1 and \dot{m}_2 we start with a mass balance about the overall system.

$$\dot{m}_1 + \dot{m}_2 = \dot{m}_5 \quad \text{--- (1)}$$

Energy balance

$$\dot{m}_1 H_1 + \dot{m}_2 H_2 = \dot{m}_5 H_5 \quad \text{--- (2)}$$

Since the process is reversible, the entropy generation is zero.

$$\dot{m}_1 S_1 + \dot{m}_2 S_2 = \dot{m}_5 S_5 \quad \text{--- (3)}$$

We can easily get the values of H_1, H_2, S_1 and S_2 from the steam tables. This leaves us with 5 unknowns $\dot{m}_1, \dot{m}_2, \dot{m}_3, H_5$ and S_5 . So we need two more equations to solve the system. We can get these two equations from the fact that upon condensation of stream 5, 300 kJ/s is released. Therefore,

$$\dot{m}_5 (H_5 - H_6) = 300 \text{ kJ/s} \quad \text{--- (4)}$$

Assuming that point 5 is either a wet steam or saturated steam, condensation will occur at constant temperature (we will need to check that assumption later).

Therefore,

$$\Delta S_{\text{condensation}} = \frac{Q_{\text{condensation}}}{T_{\text{sat}}}$$

$$S_5 - S_6 = \frac{H_5 - H_6}{T_{\text{sat}}} \quad \text{--- (5)}$$

Now we have the complete set of equations.

$$\text{From (4), } H_5 = H_6 + \frac{300}{\dot{m}_5} \quad \text{--- (6)}$$

$$\text{From (5) \& (4), } S_5 - S_6 = \frac{300}{\dot{m}_5 T_{\text{sat}}} \Rightarrow S_5 = S_6 + \frac{300}{\dot{m}_5 T_{\text{sat}}} \quad \text{--- (7)}$$

Substituting (6) in (2),

$$\dot{m}_1 H_1 + \dot{m}_2 H_2 = \dot{m}_5 H_6 + 300 \quad \text{--- (8)}$$

Substituting (7) in (3)

$$\dot{m}_1 S_1 + \dot{m}_2 S_2 = \dot{m}_5 S_6 + \frac{300}{T_{\text{sat}}} \quad \text{--- (9)}$$

From (1)

$$\dot{m}_2 = \dot{m}_5 - \dot{m}_1 \quad \text{--- (10)}$$

Substituting (10) in (8)

$$\dot{m}_1 H_1 + (\dot{m}_5 - \dot{m}_1) H_2 = \dot{m}_5 H_6 + 300 \quad \text{--- (11)}$$

same for entropy

$$\dot{m}_1 S_1 + (\dot{m}_5 - \dot{m}_1) S_2 = \dot{m}_5 S_6 + \frac{300}{T_{\text{sat}}} \quad \text{--- (12)}$$

Solving for m_1 in both (11) and (12). From (11)

$$(H_1 - H_2) m_1 = m_5 (H_6 - H_2) + 300$$

$$m_1 = m_5 \left(\frac{H_6 - H_2}{H_1 - H_2} \right) + \frac{300}{H_1 - H_2} \quad \text{--- (13)}$$

From (12)

$$(S_1 - S_2) m_1 = m_5 (S_6 - S_2) + \frac{300}{T_{\text{sat}}} \quad \text{--- (14)}$$

Substituting for m_1 from (13),

$$m_5 \left(\frac{H_6 - H_2}{H_1 - H_2} \right) + \frac{300}{H_1 - H_2} = m_5 \left(\frac{S_6 - S_2}{S_1 - S_2} \right) + \frac{300}{T_{\text{sat}}(S_1 - S_2)}$$

Solving for m_5 ,

$$m_5 = \frac{\left[\frac{300}{T_{\text{sat}}(S_1 - S_2)} - \frac{300}{H_1 - H_2} \right]}{\left[\left(\frac{H_6 - H_2}{H_1 - H_2} \right) - \left(\frac{S_6 - S_2}{S_1 - S_2} \right) \right]} \quad \text{--- (15)}$$

Getting the values from the steam tables (F.1).

Stream 1 [saturated steam @ 2700 kPa] {use closest value from table}

$$H_1 = 2801.7 \text{ kJ/kg}$$

$$S_1 = 6.2249 \text{ kJ/kg K}$$

Stream 2 [saturated steam @ 275 kPa]

$$H_2 = 2719.9 \text{ kJ/kg}$$

$$S_2 = 7.0261 \text{ kJ/kg K}$$

Stream 6 [saturated liquid @ 1000 kPa]

$$H_6 = 763.1 \text{ kJ/kg}$$

$$S_6 = 2.1393 \text{ kJ/kg K}$$

$$T_{\text{sat}} = 453.15 \text{ K}$$

Substituting in (15),

$$m_5 = 0.1497 \text{ kg/s}$$

Substituting in (13)

$$m_1 = 0.0864 \text{ kg/s}$$

Therefore,

$$m_2 = 0.0633 \text{ kg/s}$$

We still need to check if point 5 lies within the liquid-vapor dome.

Using eqn (6), we get

$$H_5 = 2767.1 \text{ kJ/kg}$$

less than the saturated vapor enthalpy @ 1000 kPa of 2776.3 kJ/kg

Therefore, the above calculation is valid.

- b) Since the process is now irreversible, we can not use equation (2). Equations (1) and (2) are still valid of course. We also know that the work generated by the turbine is used by the compressor. Therefore

$$-m_1 (H_3 - H_1) = m_2 (H_4 - H_2) \quad \text{--- (16)}$$

We can use the thermodynamic efficiencies given to get H_3 and H_4 . For the turbine, $\eta = 0.78 = \left(\frac{W_{\text{reversible}}}{W_{\text{actual}}} \right)^{-1} = \frac{W_{\text{actual}}}{W_{\text{reversible}}} = \frac{H_3 - H_1}{H_{3'} - H_1}$

$H_{3'}$ is based on an isentropic process. $S_{3'} = S_1 = 6.2249 \text{ kJ/kg K}$

Point 3' is in the liq-vap. dome

mass fraction of vapor $\rightarrow X_{3'} = \frac{s_{3'} - s_g}{s_v - s_g} = \frac{6.2249 - 2.1393}{4.4426} = 0.9196$ from steam table @ 1000 kPa

$$H_{3'} = H_g + X_{3'} (H_v - H_g) = 763.1 + 0.9196 (2013.1) = 2614.4 \text{ kJ/kg}$$

Therefore, $H_3 = H_1 + \eta_{\text{turbine}} (H_{3'} - H_1) = 2801.7 + 0.78 (2614.4 - 2801.7) = \underline{\underline{2655.6 \text{ kJ/kg}}}$

We do the same thing for the compressor to get H_4 .

$$\eta = 0.75 = \frac{W_{\text{reversible}}}{W_{\text{actual}}} = \frac{H_4 - H_1}{H_4 - H_2}$$

H_4' is based on an isentropic process $S_{4'} = S_2 = 7.0261 \text{ kJ/kg K}$

Point 4' is in the superheated region. Using table F.2 @ 1000 kPa (p. 683)

$$H_{4'} = 2965.2 + \left(\frac{7.0261 - 6.968}{7.0485 - 6.968} \right) (3009 - 2965.2) = 2996.8 \text{ kJ/kg}$$

Therefore, $H_4 = H_1 + \frac{H_4' - H_1}{2} = 2807.7 + \frac{2996.8 - 2801.7}{0.75}$

$$H_4 = \underline{\underline{3061.8 \text{ kJ/kg}}}$$

Now the equations that we can use are

$$\dot{m}_1 + \dot{m}_2 = \dot{m}_5 \quad \text{--- (1)}$$

$$\dot{m}_1 H_1 + \dot{m}_2 H_2 = \dot{m}_5 H_5 \quad \text{--- (2)}$$

$$\dot{m}_5 (H_5 - H_6) = 300 \quad \text{--- (3)}$$

$$-\dot{m}_1 (H_3 - H_1) = \dot{m}_2 (H_4 - H_2) \quad \text{--- (16)}$$

We need to solve these 4 equations to get 4 unknowns \dot{m}_1 , \dot{m}_2 , \dot{m}_5 and H_5 .

$$H_5 = H_6 + \frac{300}{\dot{m}_5}$$

$$\dot{m}_1 H_1 + \dot{m}_2 H_2 = \dot{m}_5 H_6 + 300$$

From (16), $\dot{m}_2 = -\dot{m}_1 \frac{(H_3 - H_1)}{(H_4 - H_2)} \quad \text{--- (17)}$

From (1) $\dot{m}_1 + (-\dot{m}_1) \frac{H_3 - H_1}{H_4 - H_2} = \dot{m}_5$

Therefore, $\dot{m}_1 H_1 - \dot{m}_1 \left(\frac{H_3 - H_1}{H_4 - H_2} \right) H_2 = \dot{m}_1 H_6 \left[1 - \left(\frac{H_3 - H_1}{H_4 - H_2} \right) \right] + 300$

Solving for \dot{m}_1 ,

$$\dot{m}_1 = \frac{300}{\left[H_1 - H_6 + \left(\frac{H_3 - H_1}{H_4 - H_2} \right) (H_6 - H_2) \right]}$$

Substituting, we get

$$\dot{m}_1 = 0.1043 \text{ kg/s}$$

and from (17)

$$\dot{m}_2 = 0.0446 \text{ kg/s}$$

Thermodynamic Analysis

Assume $T_0 = 300 \text{ K}$

Using Eqn 16.2,

$$\dot{W}_{\text{ideal}} = \Delta(\dot{H})_{fs} - T_0 \Delta(\dot{S})_{fs}$$

$$\dot{W}_{\text{ideal}} = + T_0 (m_1 S_1 + m_2 S_2 - m_5 S_5)$$

To get S_5 , we need H_5 .

$$H_5 = H_6 + \frac{200}{m_5} = 763.1 + \frac{300}{(0.1043 + 0.0446)} =$$

$$H_5 = 2777.9 \frac{\text{kJ}}{\text{kg}}$$

Point 5 is slightly superheated ($H_5 > H_{\text{sat, vap}} @ 1000 \text{ kPa}$)

By interpolation, (page 683)

$$S_5 = 6.5828 + \frac{2777.9 - 2776.2}{2826.8 - 2776.2} (6.6922 - 6.5828) = 6.5864 \frac{\text{kJ}}{\text{kg K}}$$

$$m_5 = 0.1489 \frac{\text{kg}}{\text{s}}$$

$$\therefore \dot{W}_{\text{ideal}} = +300 [0.1043 \times 6.2249 + 0.0446 \times 7.0261 - 0.1489 \times 6.5864]$$

$$\dot{W}_{\text{ideal}} = -5.428 \frac{\text{kJ}}{\text{s}}$$

That means that we could have extracted $5.428 \frac{\text{kJ}}{\text{s}}$ from the system, if the process was ideal. Now we need to analyze each unit to calculate the work lost.

Turbine $\dot{W}_{\text{lost}} = + m_1 T_0 (S_3 - S_1)$

To get S_3 , we use the quality x_3 . $x_3 = \frac{H_3 - H_e}{H_v - H_e} = \frac{2655.6 - 763.1}{2013.1} = 0.94$

$$S_3 = S_e + x_3 (S_v - S_e) = 2.1393 + 0.94 (4.4426) = 6.3153 \frac{\text{kJ}}{\text{kg K}} = S_3$$

$$\dot{W}_{\text{lost, turbine}} = + 0.1043 \times 300 (6.3153 - 6.2249) = +2.83 \frac{\text{kJ}}{\text{s}}$$

Compressor

$$\dot{W}_{\text{lost}} = + \dot{m}_2 T_0 (S_4 - S_2)$$

To get S_4 , we interpolate based on the value of $H_4 = 3061.8 \text{ kJ/kg}$

$$S_4 = 7.1009 + \left(\frac{3061.8 - 3050.8}{3104.4 - 3050.8} \right) (7.1924 - 7.1009) = 7.1197 \frac{\text{kJ}}{\text{kgK}}$$

$$\dot{W}_{\text{lost, compressor}} = + 0.0446 \times 300 (7.1197 - 7.0261) = + 1.252 \frac{\text{kJ}}{\text{s}}$$

Mixing Point

$$\dot{W}_{\text{lost}} = + T_0 (\dot{m}_5 S_5 - \dot{m}_1 S_3 - \dot{m}_2 S_4)$$

$$= + 300 (0.1489 \times 6.5864 - 0.1043 \times 6.3153 - 0.0446 \times 7.1197)$$

$$\dot{W}_{\text{lost, mixing}} = + 1.347 \frac{\text{kJ}}{\text{s}}$$

	kW	Percent of $ W_{\text{ideal}} $
\dot{W}_{act} (whole system)	0	0
\dot{W}_{lost} (Turbine)	+ 2.83	52
\dot{W}_{lost} (compressor)	+ 1.252	23
\dot{W}_{lost} (mixing)	+ 1.347	25
$\dot{W}_{\text{rev}} = \dot{W}_{\text{act}} - \dot{W}_{\text{lost}}$	- 5.43	100