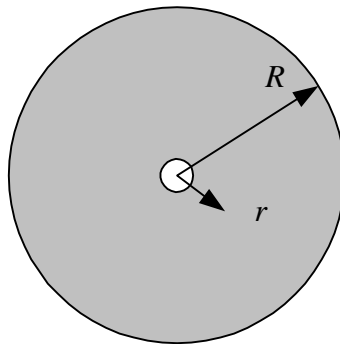
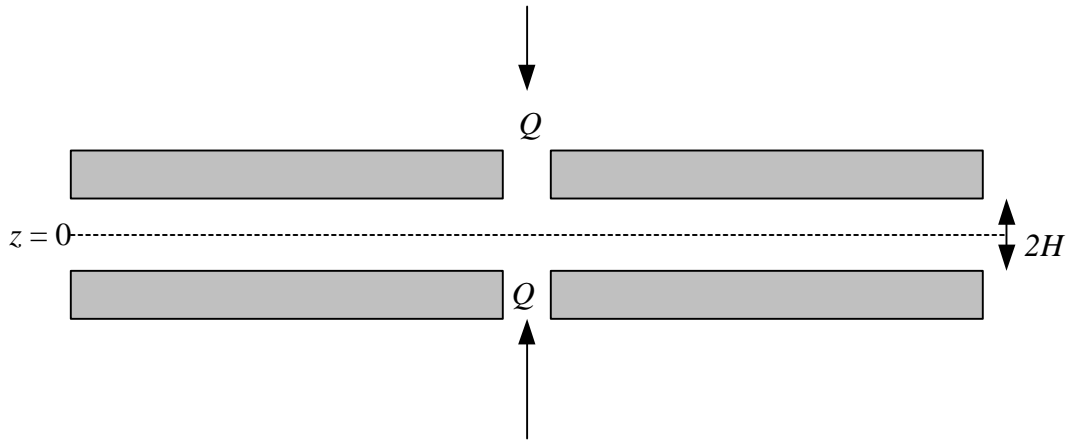


10.301- Review Problems for Exam III
Kevin D. Dorfman

Problem 1. Flow between two radial disks (from W.M. Deen)

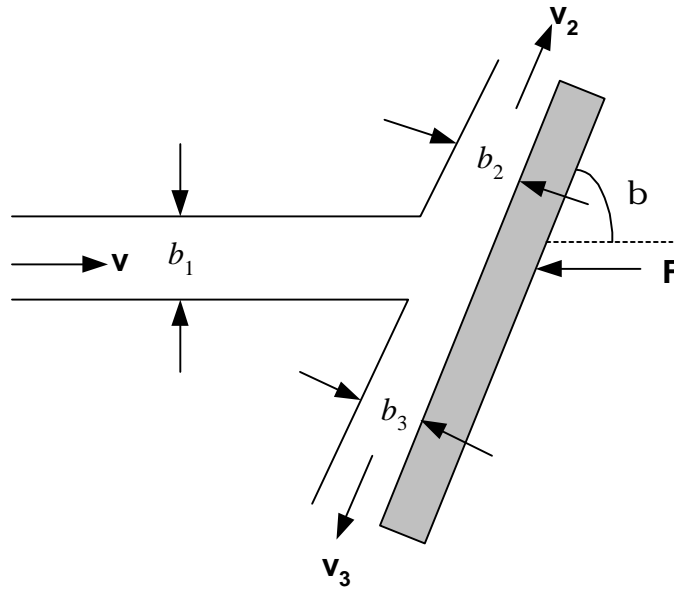
Consider the flow between two radial disks of radius R separated by a distance $2H$ with a concentric hole. Fluid flows through the hole on both sides at a volumetric flow rate Q , moving out towards the edges of the disk. Assume that the flow is steady and the fluid is Newtonian.



- (a) Relate the flow rate Q to the average fluid velocity $U(r)$.
- (b) Assume a velocity of the form $\mathbf{v} = v_r(r,z)\mathbf{i}_r = f(z)/r \mathbf{i}_r$ and a dynamic pressure field $P=P(r)$. Show that this satisfies continuity and the Navier-Stokes equation and compute the velocity field.
- (c) Eliminate the pressure from the expression in part (b), thereby expressing v_r in known quantities. What is the function $f(z)$?

Problem 2. Fluid Jet Impacting an Inclined Plate (K. Smith)

Consider a fluid jet of velocity \mathbf{v} impacting upon a plate inclined at an angle β with respect to the incoming jet. A force \mathbf{F} is applied normal to the plate to hold it in place. Assuming that viscous effects, gravity, and pressure are negligible compared to inertia, compute the widths (b_1 , b_2 , and b_3) and velocities (\mathbf{v}_2 and \mathbf{v}_3) of each jet. You may assume the process is uniform in the z -direction, so that the problem is essentially two-dimensional. Also, assume that the velocities are uniform across their cross-sections.



Problem 1

D. P. Pina

a) Relate $U(r)$ to Q .

$$2Q = [2\pi r (2H)] U(r)$$

\uparrow total flow rate \uparrow cross-sectional area \uparrow mean velocity

$$U(r) = \frac{Q}{2\pi Hr}$$

b) $\underline{v} = v_r(r, z) \underline{i}_r = f(z)/r \underline{i}_r$; $P = P(r)$

i) continuity: $\frac{\partial}{\partial r}(r v_r) = 0 \Rightarrow v_r = \frac{f(z)}{r} \checkmark$

ii) θ -momentum: $\frac{\partial P}{\partial \theta} = 0 \Rightarrow P = P(r, z)$

(iii) z -momentum: $\frac{\partial P}{\partial z} = 0 \Rightarrow P = P(r) \checkmark$

(iv) r -momentum:

$$0 = -\frac{\partial P}{\partial r} + \mu \left[\frac{\partial}{\partial r} \left(\frac{1}{r} \frac{\partial}{\partial r} (r v_r) \right) + \frac{\partial^2 v_r}{\partial z^2} \right]$$

\uparrow
= $f(z)$

$$\frac{\partial^2 v_r}{\partial z^2} = \frac{1}{\mu} \frac{dP}{dr}$$

 \uparrow
not a function of z .

②

K. Dorfman

Integrate $\frac{\partial v_r}{\partial z} = \frac{z}{\mu} \frac{dP}{dr} + C_1(r)$
 $\uparrow = 0$ by symmetry @ centerline

Integrate: $v_r = \frac{z^2}{2\mu} \frac{dP}{dr} + C_2(r)$

No slip at $z=H$.

$$0 = \frac{H^2}{2\mu} \frac{dP}{dr} + C_2(r) \Rightarrow C_2(r) = -\frac{H^2}{2\mu} \frac{dP}{dr}$$

Consequently,

$$v_r = \frac{-H^2}{2\mu} \frac{dP}{dr} \left[1 - \left(\frac{z}{H}\right)^2 \right]$$

we still do not know this!

c) Find the function $f(z)$ by relating to $U(r)$

$$U(r) = \frac{1}{2H} \int_{-H}^H v_r dz = \frac{1}{2H} \left[\frac{-H^2}{2\mu} \frac{dP}{dr} \right] \int_{-H}^H \left(1 - \frac{z^2}{H^2} \right) dz$$

$$U(r) = \frac{-H^2}{2\mu} \frac{dP}{dr} \left(\frac{2}{3} \right) = \frac{Q}{2\pi H r}$$

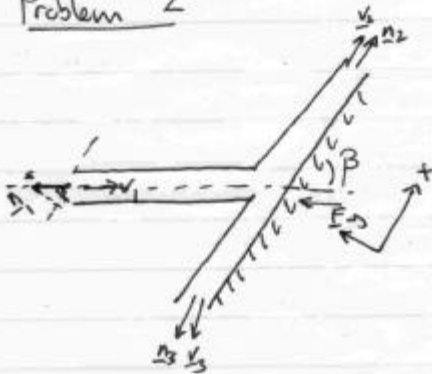
$$\text{So } \frac{-H^2}{2\mu} \frac{dP}{dr} = \frac{3Q}{4\pi H r}$$

and $v_r = \frac{1}{r} \left[\frac{3Q}{4\pi H} \left(1 - \frac{z^2}{H^2} \right) \right]$

\uparrow
 $f(z)$

Problem 2

K. Dorfman



1. $-\underline{n} \cdot \underline{v}_1 = v_1 = V$
2. $-\underline{n} \cdot \underline{v}_2 = -V_2$
3. $-\underline{n} \cdot \underline{v}_3 = -V_3$

Conservation of mass: $0 = \int_{\Sigma} (-\underline{n} \cdot \underline{v}) dS$

$$0 = v_1 b_1 - v_2 b_2 - v_3 b_3$$

Since we are neglecting viscous effects, ^{the pressure an atmosphere at the edges} pressure, and gravity, then Bernoulli's equation says that $\frac{1}{2} \rho v^2$ is a constant on a streamline. Thus:

$$v_1 = v_2 = v_3 \quad (\text{magnitudes are equal})$$

Substitute This into The mass balance and we get

$$b_1 = b_2 + b_3$$

To use conservation of momentum, we need The vector forms of The velocities and The force:

$$\begin{aligned} \underline{v}_1 &= V (\cos \beta \underline{i}_x - \sin \beta \underline{i}_y) \\ \underline{v}_2 &= V \underline{i}_x \\ \underline{v}_3 &= -V \underline{i}_x \\ \underline{F} &= F \underline{i}_y \end{aligned}$$

④

K. Dorfman

Neglecting the indicated forces, the linear momentum equation adopts the form:

$$\begin{aligned} 0 &= \int_{dx} (-\eta \cdot v) \rho v \, dS + \underline{F} \\ &= \int_1 v \rho v_1 \, dS + \int_2 (-v) \rho v_2 \, dS + \int_3 (-v) \rho v_3 \, dS + F_{iy} \end{aligned}$$

$$0_{ix} + 0_{iy} = \rho v^2 b_1 (\cos \beta i_x - \sin \beta i_y) + (-v) \rho v b_2 i_x + (-v) \rho (-v i_x) b_3 + F_{iy}$$

For the y-component:

$$0 = -\rho v^2 b_1 \sin \beta + F_y \Rightarrow \boxed{b_1 = \frac{F_y}{\rho v^2 \sin \beta}}$$

For the x-component:

$$0 = \rho v^2 \cos \beta b_1 - \rho v^2 b_2 + \rho v^2 b_3 = 0$$

$$0 = b_1 \cos \beta - b_2 + b_3$$

$$0 = b_1 \cos \beta - b_2 + b_1 - b_2 \quad (\text{from } b_3 = b_1 - b_2 \text{ from c. mass})$$

$$\boxed{b_2 = \frac{b_1 (1 + \cos \beta)}{2}}$$

$$\boxed{b_3 = b_1 - b_2 = \frac{b_1 (1 - \cos \beta)}{2}}$$