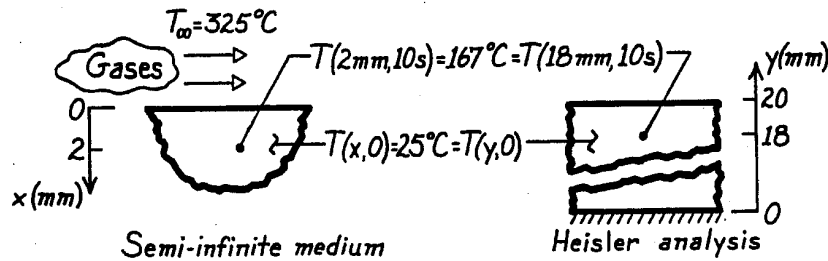


## PROBLEM 5.60

**KNOWN:** Temperature response of an in-depth thermocouple in a block, initially at a uniform temperature, embedded in the channel suddenly exposed to hot gases.

**FIND:** The surface convection heat flux assuming the block is a semi-infinite medium. Compare result with that obtained from a Heisler analysis.

**SCHEMATIC:**



**ASSUMPTIONS:** (1) Block behaves as semi-infinite medium, (2) Constant properties.

**PROPERTIES:** Block (given):  $k = 15 \text{ W/m}\cdot\text{K}$ ,  $\alpha = 2.0 \times 10^{-5} \text{ m}^2/\text{s}$ .

**ANALYSIS:** Treat the block as a semi-infinite medium, initially at a uniform temperature  $T(x, 0) = T_i$ , and suddenly exposed to a surface convective flux ( $T_\infty, h$ ) (Fig. 5.7, case 3). Evaluate,

$$\frac{T(x, t) - T_i}{T_\infty - T_i} = \frac{(167 - 25)^\circ\text{C}}{(325 - 25)^\circ\text{C}} = 0.4733$$

$$\frac{x}{2(\alpha t)^{1/2}} = \frac{0.002 \text{ m}}{2(2 \times 10^{-5} \text{ m}^2/\text{s} \times 10 \text{ s})^{1/2}} = 0.0707$$

and from Fig. 5.8, find

$$\frac{h(\alpha t)^{1/2}}{k} \approx 0.8$$

$$h = \frac{0.8 k}{(\alpha t)^{1/2}} = \frac{0.8 \times 15 \text{ W/m}\cdot\text{K}}{(2 \times 10^{-5} \text{ m}^2/\text{s} \times 10 \text{ s})^{1/2}} = 849 \text{ W/m}^2\cdot\text{K}.$$

From Figure 5.8, read that at  $x = 0$ ,

$$\frac{T(0, t) - T_i}{T_\infty - T_i} \approx 0.52 \quad \text{or} \quad T(0, 10 \text{ s}) = 181^\circ\text{C}.$$

Hence, the surface convective heat flux at  $t = 10 \text{ s}$  is

$$q''_x(0, 10 \text{ s}) = h[T_\infty - T(0, t)] \tag{1}$$

$$q''_x(0, 10 \text{ s}) = 849 \text{ W/m}^2\cdot\text{K}[325 - 181]^\circ\text{C} = 122.3 \text{ kW/m}^2.$$



Continued .....

### PROBLEM 5.60 (Cont.)

Treating the block as a plane wall, consider the coordinate system shown above using the approximate solution, Eq. 5.40,

$$\theta^* = C_1 \exp(-\zeta_1^2 Fo) \cos \zeta_1 y^* \quad (2)$$

where

$$\theta^* = \frac{T(y^*, 10s) - T_\infty}{T_1 - T_\infty} = \frac{(167 - 325)^\circ\text{C}}{(25 - 325)^\circ\text{C}} = 0.527$$

$$Fo = \frac{\alpha t}{L^2} = \frac{2 \times 10^{-5} \text{ m}^2/\text{s} \times 10\text{s}}{(0.020 \text{ m})^2} = 0.50$$

then

$$0.527 = C_1 \exp(-\zeta_1^2 \times 0.50) \cos(0.9 \zeta_1) \quad (3)$$

where  $y^* = 18/20 = 0.9$  and  $(C_1, \zeta_1) = f(Bi)$ . Since  $Bi = hL/k$  and  $h$  is presently unknown, an iterative solution is necessary. Let  $h = 850 \text{ W/m}^2 \cdot \text{K}$  for which

$$Bi = 850 \text{ W/m}^2 \cdot \text{K} \times 0.020 \text{ m} / 15 \text{ W/m} \cdot \text{K} = 1.133.$$

Find values for  $C_1$  and  $\zeta_1$  from Table 5.1 to see if Eq. (3) is satisfied. Here are several tabulated results where RHS denotes the RHS of Eq. (3):

Bi	$\zeta_1$ (rad)	$C_1$	RHS
1.0	0.8603	1.1191	0.553
0.9	0.8274	1.1107	0.580
2.0	1.0769	1.1795	0.374

By linear interpolation, find  $Bi \approx 1.145$  or  $h = 859 \text{ W/m}^2 \cdot \text{K}$ . The surface temperature, from Eq. 5.40 with  $y^* = 1$  is

$$\theta_o^* = C_1 \exp(-\zeta_1^2 Fo) \cos(1 \times \zeta_1)$$

$$\theta_o^* = \frac{T(0, 10s) - 325}{25 - 325} = 1.1279 \exp(-0.8917^2 \times 0.5) \cos(0.8917) = 0.4760$$

$$T(20\text{mm}, 10s) = 182^\circ\text{C}$$

where  $C_1 = 1.1279$  and  $\zeta_1 = 0.8917$  determined from interpolation in Table 5.1 with  $Bi = 1.145$ . Hence the convection heat flux is

$$q''_x(20\text{mm}, 10s) = h[T_\infty - T(20\text{mm}, 10s)]$$

$$q''_x = 859 \text{ W/m}^2 (325 - 182)^\circ\text{C} = 122.8 \text{ kW/m}^2.$$



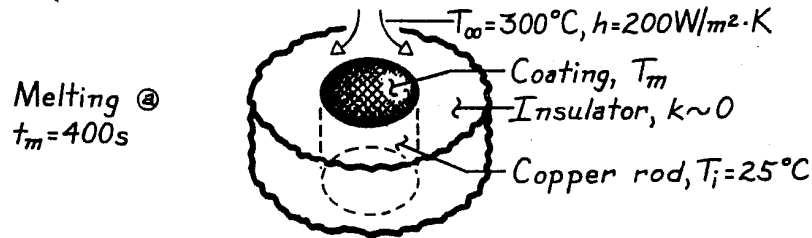
**COMMENTS:** For these conditions, the block behavior does approximate that of a semi-infinite medium. This method is used to determine surface convective heat flux in many applications. The purpose of imbedding the thermocouple is to provide protection from the gas stream.

### PROBLEM 5.63

**KNOWN:** Procedure for measuring convection heat transfer coefficient, which involves melting of a surface coating.

**FIND:** Melting point of coating for prescribed conditions.

**SCHEMATIC:**



**ASSUMPTIONS:** (1) One-dimensional conduction in solid rod (negligible losses to insulation), (2) Rod approximation as semi-infinite medium, (3) Negligible surface radiation, (4) Constant properties, (5) Negligible thermal resistance of coating.

**PROPERTIES:** Copper rod (Given):  $k = 400 \text{ W/m} \cdot \text{K}$ ,  $\alpha = 10^{-4} \text{ m}^2/\text{s}$ .

**ANALYSIS:** Problem corresponds to transient conduction in a semi-infinite solid. Thermal response is given by

$$\frac{T(x,t) - T_i}{T_\infty - T_i} = \text{erfc}\left(\frac{x}{2(\alpha t)^{1/2}}\right) - \left[\exp\left(\frac{hx}{k} + \frac{h^2 \alpha t}{k^2}\right)\right] \left[\text{erfc}\left(\frac{x}{2(\alpha t)^{1/2}} + \frac{h(\alpha t)^{1/2}}{k}\right)\right].$$

For  $x = 0$ ,  $\text{erfc}(0) = 1$  and  $T(x,t) = T(0,t) = T_s$ . Hence

$$\frac{T_s - T_i}{T_\infty - T_i} = 1 - \exp\left(\frac{h^2 \alpha t}{k^2}\right) \text{erfc}\left(\frac{h(\alpha t)^{1/2}}{k}\right)$$

with

$$\frac{h(\alpha t_m)^{1/2}}{k} = \frac{200 \text{ W/m}^2 \cdot \text{K} (10^{-4} \text{ m}^2/\text{s} \times 400 \text{ s})^{1/2}}{400 \text{ W/m} \cdot \text{K}} = 0.1$$

$$T_s = T_m = T_i + (T_\infty - T_i)[1 - \exp(0.01) \text{erfc}(0.1)]$$

$$T_s = 25^\circ\text{C} + 275^\circ\text{C}[1 - 1.01 \times 0.888] = 53.5^\circ\text{C}.$$

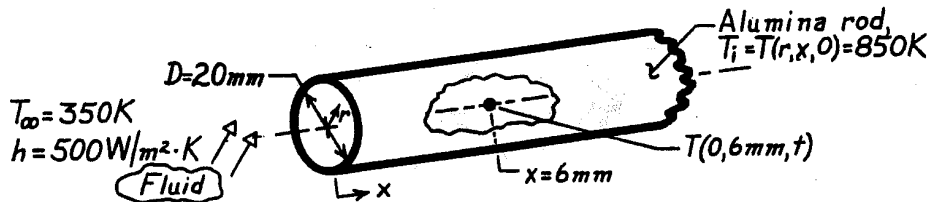
**COMMENTS:** Use of the procedure to evaluate  $h$  from measurement of  $t_m$  necessitates iterative calculations.

## PROBLEM 5.77

**KNOWN:** A long alumina rod, initially at a uniform temperature of 850K, is suddenly exposed to a cooler fluid.

**FIND:** Temperature of the rod after 30s, at an exposed end,  $T(0,0,t)$ , and at an axial distance 6mm from the end,  $T(0, 6\text{mm}, t)$ .

**SCHEMATIC:**



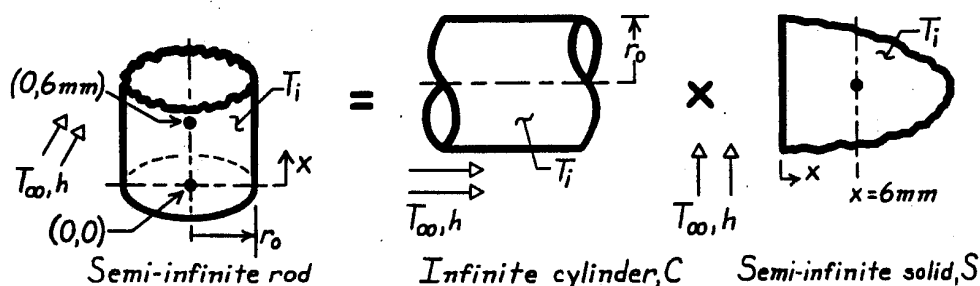
**ASSUMPTIONS:** (1) Two-dimensional conduction in  $(r,x)$  directions, (2) Constant properties, (3) Convection coefficient is same on end and cylindrical surfaces.

**PROPERTIES:** Table A-2, Alumina, polycrystalline aluminum oxide (assume  $\bar{T} \approx (850+600)\text{K}/2 = 725\text{K}$ ):  $\rho = 3970 \text{ kg/m}^3$ ,  $c = 1154 \text{ J/kg}\cdot\text{K}$ ,  $k = 12.4 \text{ W/m}\cdot\text{K}$ .

**ANALYSIS:** First, check if system behaves as a lumped capacitance. Find

$$Bi = \frac{hL_c}{k} = \frac{h(r_o/2)}{k} = \frac{500 \text{ W/m}^2\cdot\text{K} (0.010\text{m}/2)}{12.4 \text{ W/m}\cdot\text{K}} = 0.202.$$

Since  $Bi > 0.1$ , rod does not behave as spacewise isothermal object. Hence, treat rod as a semi-infinite cylinder, the multi-dimensional system Case (f), Fig. 5.11.



The product solution can be written as

$$\theta^*(r,x,t) = \frac{\theta(r,x,t)}{\theta_i} = \frac{\theta(r,t)}{\theta_i} \times \frac{\theta(x,t)}{\theta_i} = C(r^*,t^*) \times S(x^*,t^*)$$

*Infinite cylinder,  $C(r^*,t^*)$ .* Using the Heisler charts with  $r^* = r/r_o = 0$  and

$$Bi^{-1} = \left[ \frac{h r_o}{k} \right]^{-1} = \left[ \frac{500 \text{ W/m}^2\cdot\text{K} \times 0.01\text{m}}{12.4 \text{ W/m}\cdot\text{K}} \right]^{-1} = 2.48.$$

Evaluate  $\alpha = k/\rho c = 2.71 \times 10^{-6} \text{ m}^2/\text{s}$ , find  $Fo = \alpha t/r_o^2 = 2.71 \times 10^{-6} \text{ m}^2/\text{s} \times 30\text{s}/(0.01\text{m})^2 = 0.812$ . From the Heisler chart, Fig. C.4, with  $Bi^{-1} = 2.48$  and  $Fo = 0.812$ , read  $C(0,t^*) = \theta(0,t)/\theta_i = 0.61$ .

Continued .....

### PROBLEM 5.77 (Cont.)

*Semi-infinite medium,  $S(x^*, t^*)$ .* Recognize this as Case (3), Fig. 5.7. From Eq. 5.60, note that the LHS needs to be transformed as follows,

$$\frac{T - T_i}{T_\infty - T_i} = 1 - \frac{T - T_\infty}{T_i - T_\infty} \quad S(x, t) = \frac{T - T_\infty}{T_i - T_\infty}.$$

Thus,

$$S(x, t) = 1 - \left\{ \operatorname{erfc} \left[ \frac{x}{2(\alpha t)^{1/2}} \right] - \left[ \exp \left[ \frac{hx}{k} + \frac{h^2 \alpha t}{k^2} \right] \right] \left[ \operatorname{erfc} \left[ \frac{x}{2(\alpha t)^{1/2}} + \frac{h(\alpha t)^{1/2}}{k} \right] \right] \right\}.$$

Evaluating this expression at the surface ( $x=0$ ) and 6mm from the exposed end, find

$$S(0, 30s) = 1 - \left\{ \operatorname{erfc}(0) - \left[ \exp \left[ 0 + \frac{(500 \text{ W/m}^2 \cdot \text{K})^2 \cdot 2.71 \times 10^{-6} \text{ m}^2/\text{s} \times 30s}{(12.4 \text{ W/m} \cdot \text{K})^2} \right] \right] \left[ \operatorname{erfc} \left[ 0 + \frac{500 \text{ W/m}^2 \cdot \text{K} \cdot (2.71 \times 10^{-6} \text{ m}^2/\text{s} \times 30s)^{1/2}}{12.4 \text{ W/m} \cdot \text{K}} \right] \right] \right\}$$

$$S(0, 30s) = 1 - \left\{ 1 - [\exp(0.1322)] [\operatorname{erfc}(0.3636)] \right\} = 0.693.$$

Note that Table B.2 was used to evaluate the complementary error function,  $\operatorname{erfc}(w)$ .

$$S(6\text{mm}, 30s) = 1 - \left\{ \operatorname{erfc} \left[ \frac{0.006\text{m}}{2(2.71 \times 10^{-6} \text{ m}^2/\text{s} \times 30s)^{1/2}} \right] - \left[ \exp \left[ \frac{500 \text{ W/m}^2 \cdot \text{K} \times 0.006\text{m}}{12.4 \text{ W/m} \cdot \text{K}} + 0.1322 \right] \right] [\operatorname{erfc}(0.3327 + 0.3636)] \right\} = 0.835.$$

The product solution can now be evaluated for each location. At (0, 0),

$$\theta^*(0, 0, t) = \frac{T(0, 0, 30s) - T_\infty}{T_i - T_\infty} = C(0, t^*) \times S(0, t^*) = 0.61 \times 0.693 = 0.423.$$

Hence,

$$T(0, 0, 30s) = T_\infty + 0.423(T_i - T_\infty) = 350\text{K} + 0.423(850 - 350)\text{K} = 561\text{K}.$$

At (0, 6mm),

$$\theta^*(0, 6\text{mm}, t) = C(0, t^*) \times S(6\text{mm}, t^*) = 0.61 \times 0.835 = 0.509$$

$$T(0, 6\text{mm}, 30s) = 604\text{K}.$$

**COMMENTS:** Note that the temperature at which the properties were evaluated was a good estimate.