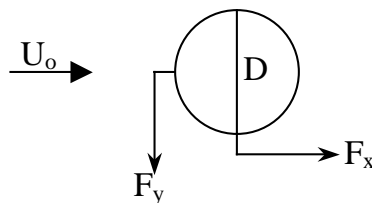


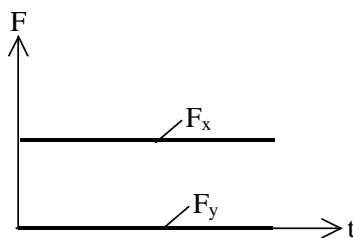
## 13.021 - Marine Hydrodynamics Lecture 15

### Vortex Shedding and Vortex Induced Vibrations

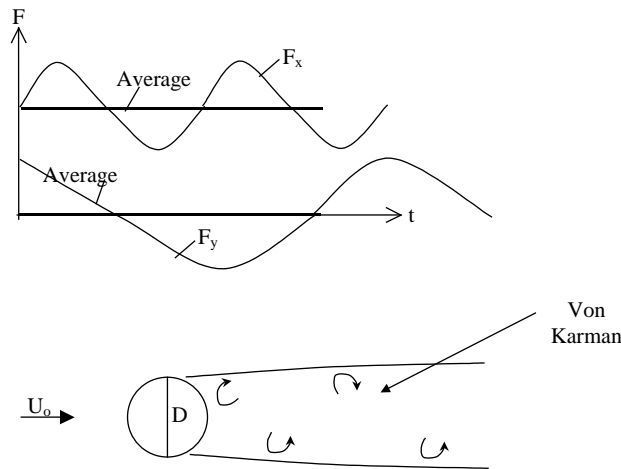
Consider a steady flow  $U_o$  on a bluff body with diameter  $D$ .



We would **expect** the average forces to be:



The **measured** oscillatory forces are:

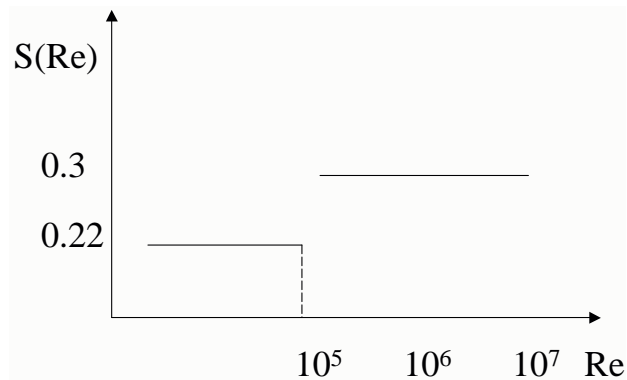


**Von Karman Street:** Unsteady non-symmetric wake of staggered array of vortices.

Frequency of vortex shedding  $f = \omega/2\pi$  is given by a non-dimensional number.

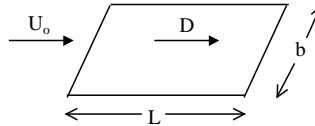
$$\frac{fD}{U_0} = S(Re)$$

where  $f$  is the Strouhal frequency,  $D$  is the body diameter and  $S$  is the Strouhal number. The Drag  $F_x$  has frequency  $2f$  and non-zero mean value, and the Lift  $F_y$  has frequency  $f$ , but zero mean value. For laminar flow  $S \sim 0.22$  for a cylinder, and for turbulent flow,  $S \sim 0.3$  for a cylinder.



$C_D$  and  $C_L$  are functions of the correlation length. For " $\infty$ " correlation length,  $C_L \sim O(1)$  for a fixed cylinder, comparable to  $C_D$ . For a moving cylinder, if the Strouhal frequency  $f$  is close to one of the cylinder natural frequencies, lock-in occurs. Therefore, if one natural frequency is close to the Strouhal Frequency  $f_S$ , we have large amplitude of motion  $\Rightarrow$  **Vortex induced vibration (VIV)**.

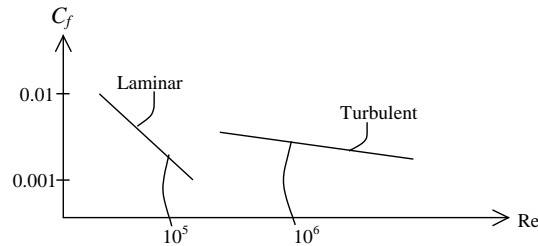
## 4.2 – Drag on a very streamlined body: Flat Plate



$$\frac{D}{\frac{1}{2}\rho U^2 \underbrace{(Lb)}_{S \text{ (wetted area) one side of plate}}} = C_f(Re, L/b)$$

Unlike a bluff body,  $C_f$  is a strong function of  $Re$  since  $D$  is proportional to  $\nu$ .  $\left(\tau = \nu \frac{\partial u}{\partial y}\right)$

### Flat Plate Drag



- $Re$  depends on plate smoothness, ambient turbulence, ...

- In general,  $C_f$ 's are much smaller than  $C_D$ 's (a factor of 10 : 100). Therefore, designing streamlined bodies allows minimal separation and form drag (at the expense of friction drag).
- In general, for streamlined bodies

$C_{force}$  is a combination of  $C_D(Re)$  and  $C_f(Re)$

where  $C_D$  is a function of the regime and  $C_f$  is a function of  $Re_L$  continuously.

### Governing equations:

- Navier-Stokes':

$$\frac{\partial \vec{v}}{\partial t} + (\vec{v} \cdot \nabla) \vec{v} = -\frac{1}{\rho} \nabla p + \nu \nabla^2 \vec{v} + \frac{1}{\rho} \vec{f}$$

- Conservation of mass:

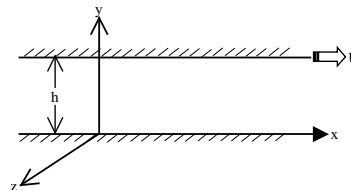
$$\nabla \cdot \vec{v} = 0$$

- Boundary conditions on solid boundaries “no-slip”:

$$\vec{v} = \vec{U}$$

Equations very difficult to solve, analytic solution only for a few very special cases (usually when  $(\vec{v} \cdot \nabla) \vec{v} = 0 \dots$ )

### 4.3 Steady Laminar Flow Between 2 Infinite Parallel Walls - Plane Couette Flow



Assume steady flow ( $\frac{\partial}{\partial t} = 0$ ). For the horizontal dimensions  $(x, z) \gg h$ , we assume flow independent of  $x$  and  $z$ , i.e.,  $\frac{\partial \vec{v}}{\partial x}, \frac{\partial \vec{v}}{\partial z} = 0$ , so  $\vec{v} = \vec{v}(y)$ .

- Kinematic boundary conditions (k.b.c.):

$$\vec{v} = (0, 0, 0) \text{ on } y = 0$$

$$\vec{v} = (U, 0, 0) \text{ on } y = h$$

- Conservation of mass:

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} + \frac{\partial w}{\partial z} = 0 \rightarrow \frac{\partial v}{\partial y} = 0 \rightarrow v = v(x, z),$$

but  $v = 0$  on  $y = 0, h$  (k.b.c.), therefore  $v = 0$ .

- Navier-Stokes equation for steady flow ( $\frac{\partial}{\partial t} = 0$ ), no  $\vec{f}$  and  $\frac{\partial \vec{v}}{\partial x} = \frac{\partial \vec{v}}{\partial z} = v = 0$ :

$$u : \nu \frac{\partial^2 u}{\partial y^2} = \frac{1}{\rho} \frac{\partial p}{\partial x}$$

$$v : \frac{\partial p}{\partial y} = 0 \rightarrow p = p(x, z)$$

$$w : \nu \frac{\partial^2 w}{\partial y^2} = \frac{1}{\rho} \frac{\partial p}{\partial z}$$

We assume that  $p = p(x)$ , i.e.  $\frac{\partial p}{\partial z} \equiv 0$ , then  $\nu \frac{\partial^2 w}{\partial y^2} = 0 \rightarrow w = a + by$ . But k.b.c.:  $w = 0$  on  $y = 0, h$ . Therefore,  $w \equiv 0$ .

Finally:  $v = w = 0, u = u(y), p = p(x)$

$$\frac{d^2 u}{dy^2} = \frac{1}{\mu} \frac{dp}{dx} \text{ where } \mu = \rho \nu$$

- Solution:

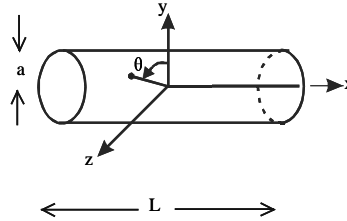
$$u = \frac{1}{2} y^2 \frac{1}{\mu} \frac{dp}{dx} + C_1 + C_2 y$$

k.b.c.:  $C_1 = 0$  and  $C_2 = \left( U - \frac{1}{2} h^2 \frac{1}{\mu} \frac{dp}{dx} \right)$  since  $u(0) = 0$  and  $u(h) = U$ . Finally,

$$u = \frac{1}{2\mu} (y - h) y \frac{dp}{dx} + \frac{Uy}{h} \text{ (plane) Couette flow}$$

#### 4.4 - Steady Laminar Flow in a pipe - Poiseuille Flow.

Assume steady, and for  $L \gg a$ ,  $\frac{\partial \vec{v}}{\partial x} = \frac{\partial \vec{v}}{\partial \theta} \equiv 0 \rightarrow \vec{v} = \vec{v}(r), r^2 = y^2 + z^2$ .



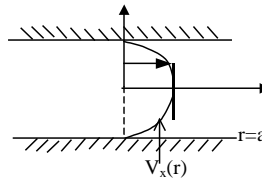
$$\vec{v} = (v_x, v_r, v_\theta)$$

Can show:

$$v_r = v_\theta = 0, \quad v_x = v_x(r), \quad p = p(x)$$

$$\frac{1}{\rho} \frac{dp}{dx} = \nu \underbrace{\left( \frac{1}{r} \frac{d}{dr} \left( r \frac{dv_x}{dr} \right) \right)}_{\substack{\text{r component of } \nabla^2 \\ \text{in cylindrical coordinates}}}$$

K.b.c.:  $v_x(a) = 0$  (no slip) and  $\frac{dv_x}{dr}(0) = 0$  (symmetry).



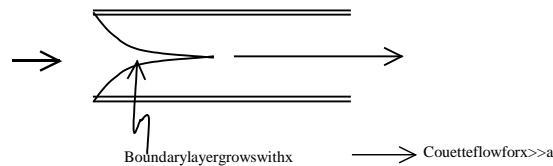
Solution:

$$v_x(r) = \frac{1}{4\mu} \left( -\frac{dp}{dx} \right) (a^2 - r^2) \text{ Poiseuille flow}$$

## 4.5 Unsteady Flow ( boundary layer growth) over an infinite flat plate

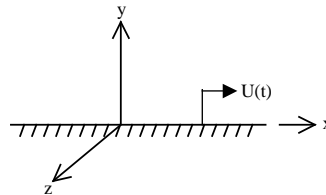
For steady  $\left( \begin{array}{l} \text{Couette} \\ \text{Poiseuille} \end{array} \right)$  flow, vorticity, viscosity effects diffuse to all  $\left( \begin{array}{l} h \\ a \end{array} \right)$

1. limit x



2. limit t (§4.5)

Consider the simplest example of an infinite plate in unsteady motion:



Assuming  $\nabla p = 0$ , we have  $\nabla \frac{\partial \vec{v}}{\partial x}, \frac{\partial \vec{v}}{\partial z} = 0$ , so  $\vec{v} = \vec{v}(y, t)$

Can show that  $v = w = 0$  and  $u = u(y, t)$ .

$$\frac{\partial u}{\partial t} + u \underbrace{\frac{\partial u}{\partial x}}_{=0} + \underbrace{v}_{=0} \frac{\partial u}{\partial y} + w \underbrace{\frac{\partial u}{\partial z}}_{=0} = -\frac{1}{\rho} \underbrace{\frac{\partial p}{\partial x}}_{=0} + \nu \left( \underbrace{\frac{\partial^2 u}{\partial x^2}}_{=0} + \frac{\partial^2 u}{\partial y^2} + \underbrace{\frac{\partial^2 u}{\partial z^2}}_{=0} \right)$$

Finally:

$$\frac{\partial u}{\partial t} = \nu \frac{\partial^2 u}{\partial y^2} \underbrace{\text{“heat”}}_{\substack{\text{momentum} \\ \text{(velocity)}}} \text{ diffusion equation} \quad (1)$$

B.c.:  $u(0, t) = U(t), t > 0$ ;  $u$  bounded ( $\rightarrow 0$ ) as  $y \rightarrow \infty$  + suitable initial condition.

#### 4.5.1 Sinusoidally Oscillating Plate

$U(t) = U_o \cos \omega t = \text{Real} \{U_o e^{i\omega t}\}$   $e^{i\alpha} = \cos \alpha + i \sin \alpha$  where  $\alpha$  is real. Let  $u(y, t) = \text{Real} \{f(y) e^{i\omega t}\}$  where  $f(y)$  is an unknown complex (magnitude & phase) amplitude. Then (1):

$$i\omega f = \nu \frac{d^2 f}{dy^2} \leftarrow 2^{nd} \text{ order ODE for } f(y)$$

General Solution:

$$f(y) = C_1 e^{(1+i)(\sqrt{\omega/2\nu})y} + C_2 e^{-(1+i)(\sqrt{\omega/2\nu})y}$$

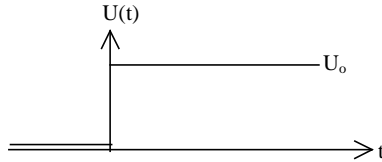
B.c.:  $u \rightarrow$  bounded as  $y \rightarrow \infty, C_1 = 0$ .  $u \rightarrow U(t)$  as  $y = 0, C_2 = U_o$ .

Finally:

$$u(y, t) = U_o e^{-(\sqrt{\omega/2\nu})y} \cos \left( -\sqrt{\frac{\omega}{2\nu}}y + \omega t \right) \text{ Stokes' (Oscillatory) b.l.}$$



## 4.5.2 Impulsively Started Plate



$$u(y, t) : \frac{\partial u}{\partial t} = \nu \frac{\partial^2 u}{\partial y^2}$$

B.c.:

$$\left. \begin{array}{l} u(0, t) = U_0 \\ u(\infty, t) = 0 \end{array} \right\} \text{ for } t > 0, \text{ i.e. } u(y, 0) = 0$$

Problem has no explicit time scale, can use dimensional analysis to solve in terms of a similarity parameter:

$$\frac{u}{U_0} = f(y, t, \nu) = \underbrace{f\left(\frac{y}{2\sqrt{\nu t}}\right)}_{\substack{\equiv \eta \text{ similarity} \\ \text{parameter}}}; \text{ i.e. } \frac{u}{U_0} = \underbrace{f(\eta)}_{\substack{\text{Self similar} \\ \text{solution}}}$$

Solution:

$$\underbrace{\frac{u}{U_0}}_{\substack{\text{Impulsively} \\ \text{started} \\ \text{flat-plate} \\ \text{boundary} \\ \text{layer} \\ \text{solution}}} = \underbrace{\text{erfc}(\eta)}_{\substack{\text{Complementary} \\ \text{error function}}} = 1 - \text{erf}(\eta) = 1 - \frac{2}{\sqrt{\pi}} \int_0^\eta e^{-\alpha^2} d\alpha$$