

16.810 (16.682)

Engineering Design and Rapid Prototyping

Design Optimization

- Structural Design Optimization

16.810

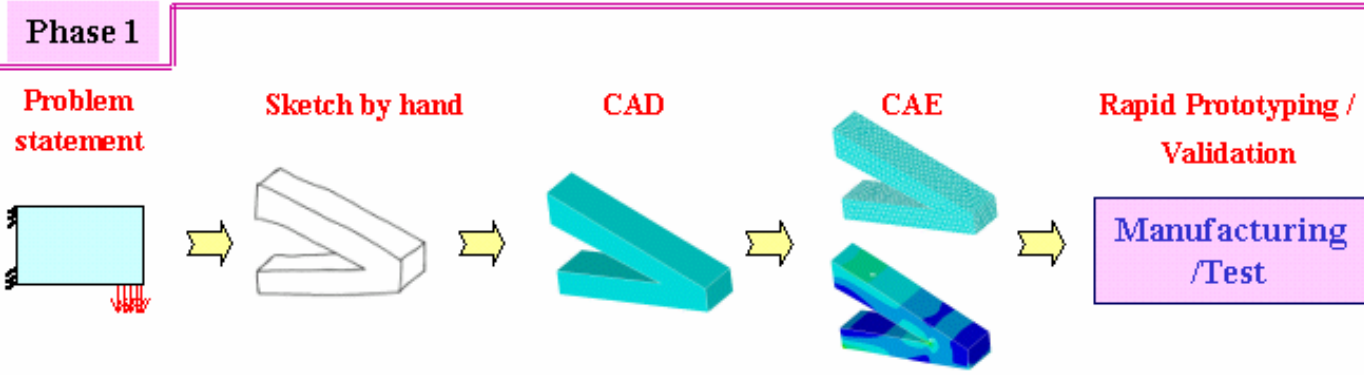
Instructor(s)

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deweck@mit.edu

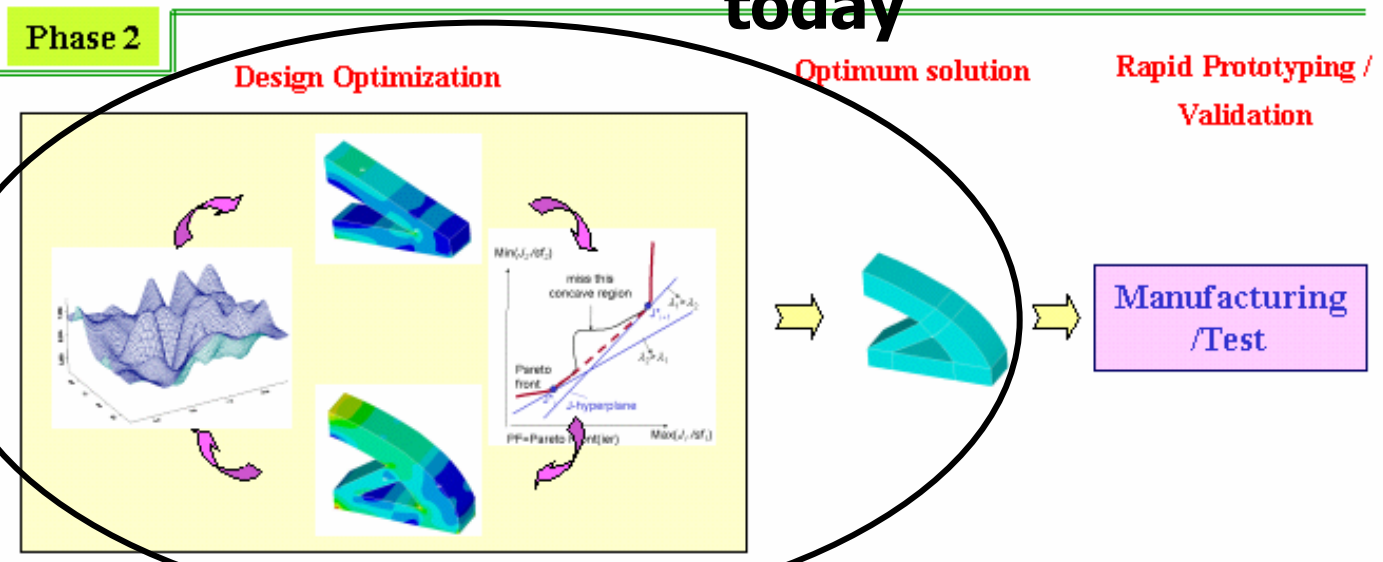
Dr. Il Yong Kim
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January 23, 2004

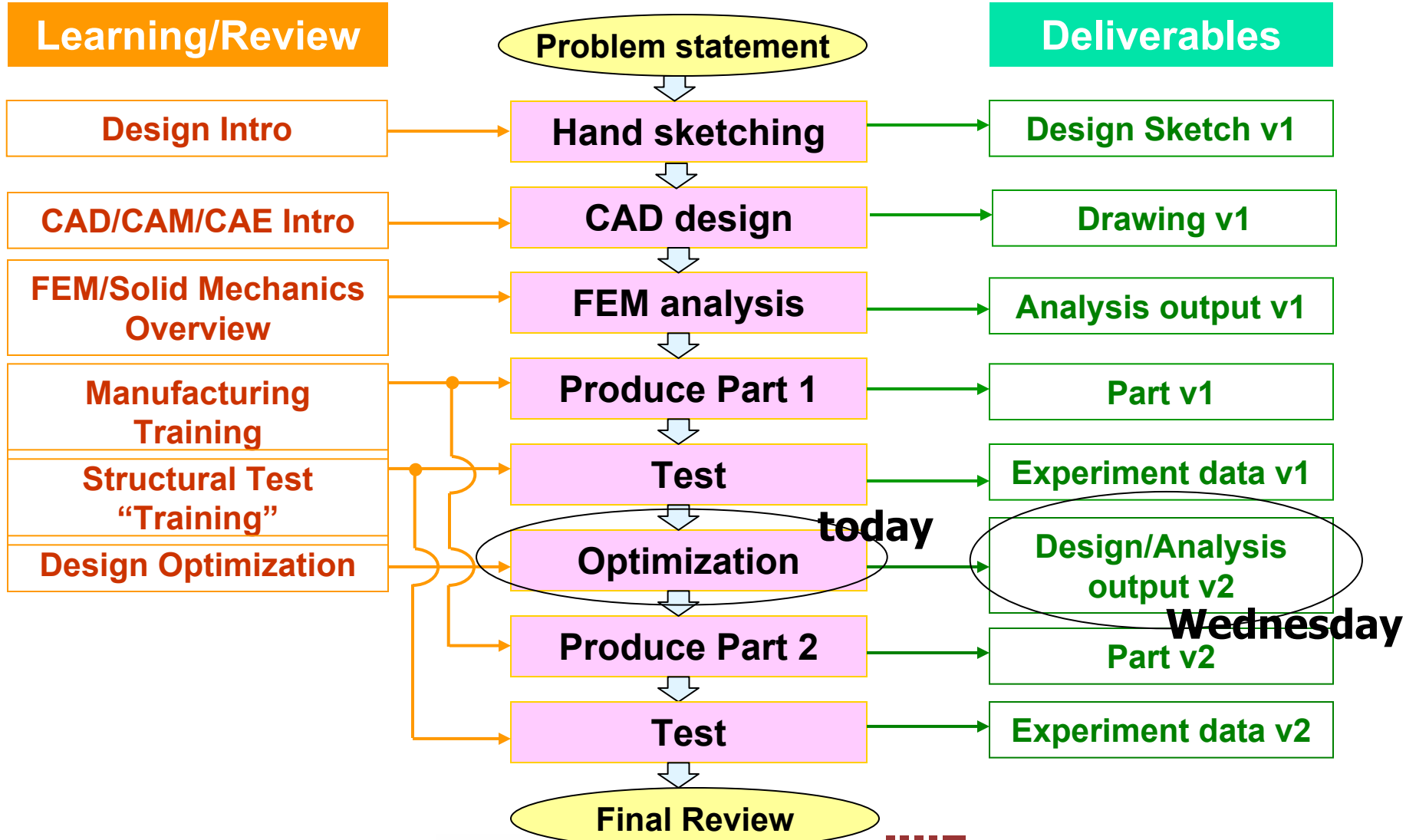
Course Concept



today



Course Flow Diagram



What Is Design Optimization?

Selecting the “best” design within the available means

1. What is our criterion for “best” design? Objective function

**2. What are the available means? Constraints
(design requirements)**

3. How do we describe different designs? Design Variables

Minimize $f(\mathbf{x})$

Subject to $g(\mathbf{x}) \leq 0$

$h(\mathbf{x}) = 0$

- Design requirements

2. Requirements

<u>Manufacturing Cost</u> (C):	$C \leq 3.6\$ / \text{part}$
<u>Performance</u> (δ_1, δ_2, f_1):	<u>Displacement</u> $\delta_1 \leq 0.078 \text{ mm}$ <u>Displacement</u> $\delta_2 \leq 0.012 \text{ mm}$ <u>First natural frequency</u> $f_1 \geq 195 \text{ Hz}$
<u>Mass</u> (m):	$m \leq 0.27 \text{ lbs}$
<u>Surface Quality</u> (Q):	$Q \geq 2$
<u>Load Case</u> (F):	$F1 = 50 \text{ lbs} / F2 = 50 \text{ lbs} / F3 = 100 \text{ lbs}$

Inequality constraints

Equality constraints

The part has to conform to the interface requirements and geometrical boundary conditions shown on page 2 of this document. This requirement cannot be waived.

3. Priorities

Low manufacturing cost is the first priority for this product. Next, the customer cares about light-weighting (low mass) and thirdly, structural performance should be as high as possible. These priorities are shown in the Ishii-matrix below:

Attribute	Constrain	Optimize	Accept
Cost	■		
Performance			■
Mass		■	

- A criterion for best design (or goodness of a design)

2. Requirements

Manufacturing Cost (C): $C \leq 3.6\$ / \text{part}$

Performance (δ_1, δ_2, f_1):
Displacement $\delta_1 \leq 0.078$ mm
Displacement $\delta_2 \leq 0.012$ mm
First natural frequency $f_1 \geq 195$ Hz

Mass (m): $m \leq 0.27$ lbs

Surface Quality (Q): $Q \geq 2$

Load Case (F): $F1 = 50$ lbs / $F2 = 50$ lbs / $F3 = 100$ lbs

The part has to conform to the interface requirements and geometrical boundary conditions shown on page 2 of this document. This requirement cannot be waived.

Objective function

3. Priorities

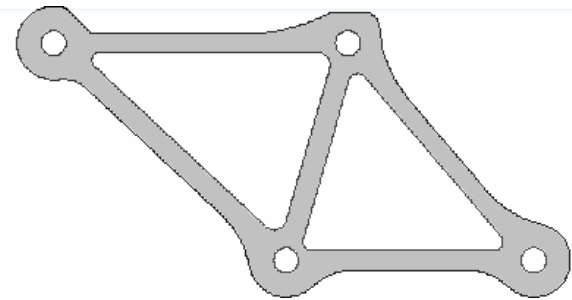
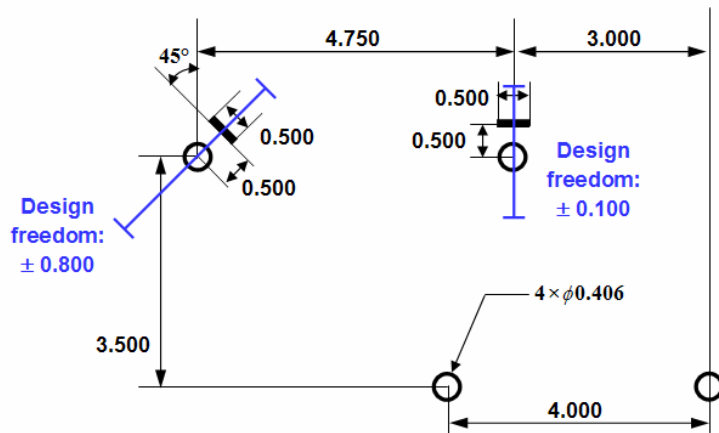
Low manufacturing cost is the first priority for this product. Next, the customer cares about light-weighting (low mass) and thirdly, structural performance should be as high as possible. These priorities are shown in the Ishii-matrix below:

Attribute	Constrain	Optimize	Accept
Cost	■		
Performance			■
Mass		■	■

Parameters that are chosen to describe the design of a system



Design variables are “controlled” by the designers



The position of upper holes along the design freedom line

For computational design optimization,



Objective function and constraints must be expressed as a function of design variables (or design vector \mathbf{X})

Objective function: $f(\mathbf{x})$

Constraints: $g(\mathbf{x}), h(\mathbf{x})$

Cost = $f(\text{design})$

Displacement = $f(\text{design})$

Natural frequency = $f(\text{design})$

Mass = $f(\text{design})$

What is "f" for each case?

Minimize $f(\mathbf{x})$

Subject to $g(\mathbf{x}) \leq 0$

$h(\mathbf{x}) = 0$

$f(\mathbf{x})$: Objective function to be minimized

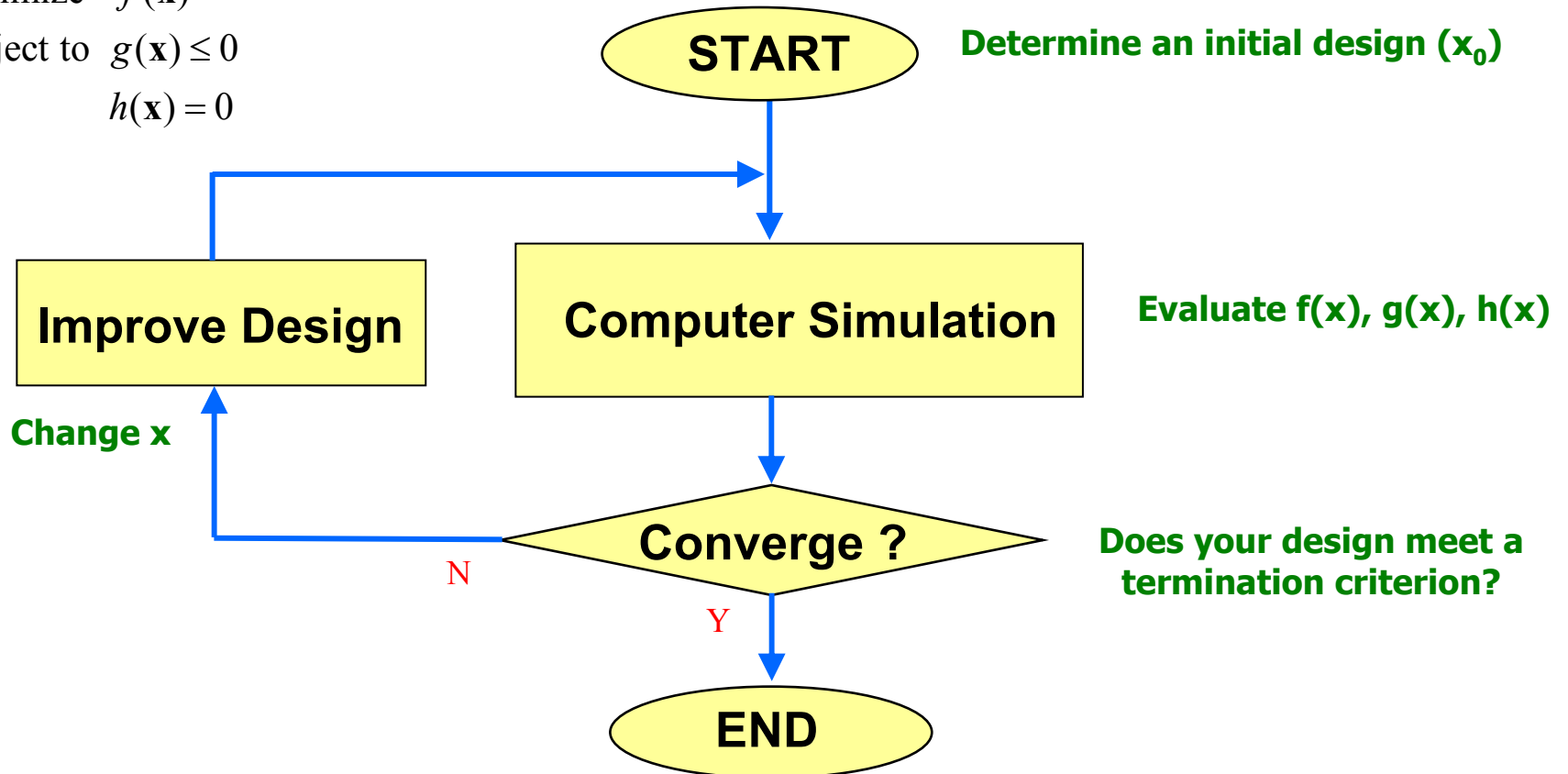
$g(\mathbf{x})$: Inequality constraints

$h(\mathbf{x})$: Equality constraints

\mathbf{x} : Design variables

Optimization Procedure

Minimize $f(\mathbf{x})$
Subject to $g(\mathbf{x}) \leq 0$
 $h(\mathbf{x}) = 0$



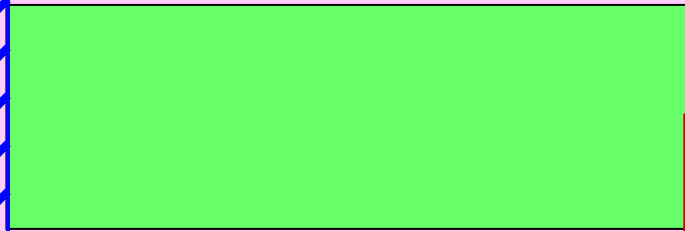
Selecting the best “**structural**” design

- Size Optimization
- Shape Optimization
- Topology Optimization

$$\begin{aligned} &\text{minimize } f(\mathbf{x}) \\ &\text{subject to } g(\mathbf{x}) \leq 0 \\ &\quad h(\mathbf{x}) = 0 \end{aligned}$$



BC's are given



Loads are given

1. To make the structure strong
e.g. Minimize displacement at the tip

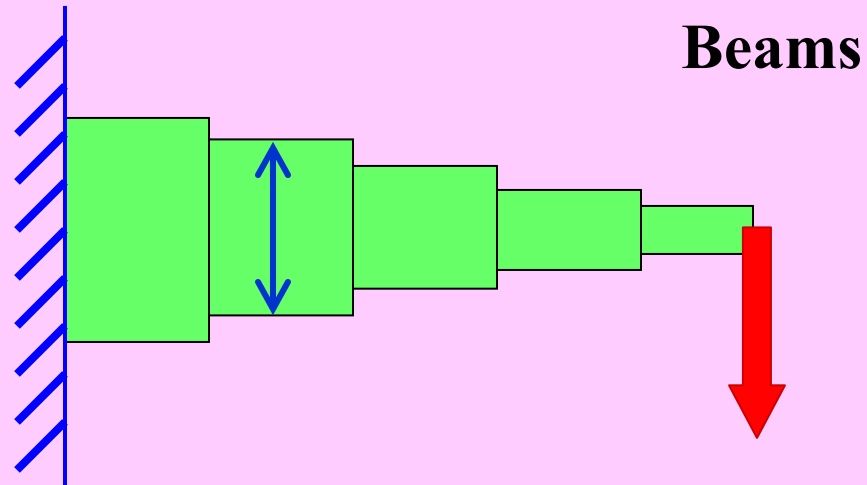
➔ *Min. $f(\mathbf{x})$*

2. Total mass $\leq M_c$

➔ $g(\mathbf{x}) \leq 0$

Size Optimization

$$\begin{aligned} &\text{minimize } f(\mathbf{x}) \\ &\text{subject to } g(\mathbf{x}) \leq 0 \\ &\quad h(\mathbf{x}) = 0 \end{aligned}$$



Design variables (\mathbf{x})

\mathbf{x} : thickness of each beam

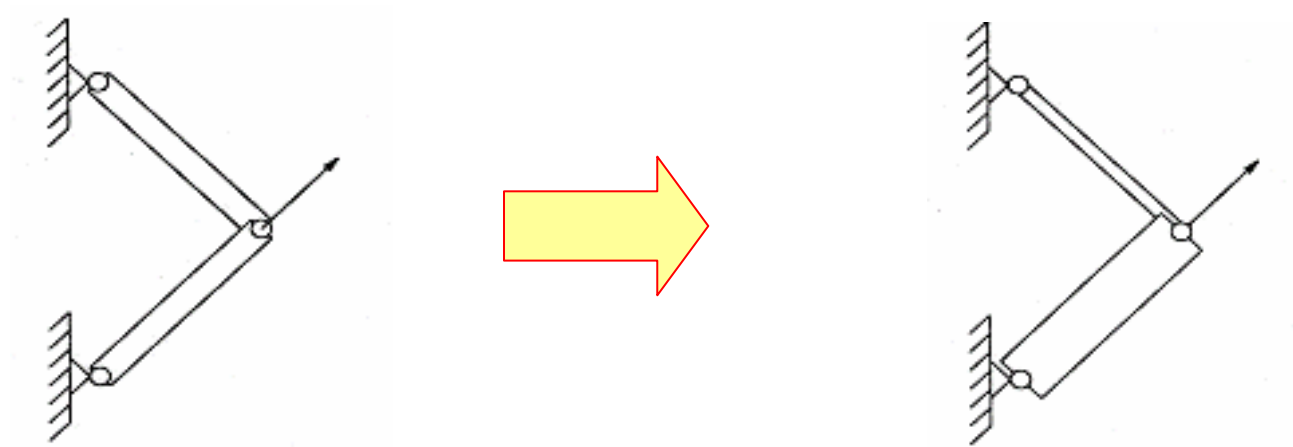
$f(\mathbf{x})$: compliance

$g(\mathbf{x})$: mass

Number of design variables (ndv)

ndv = 5

Size Optimization

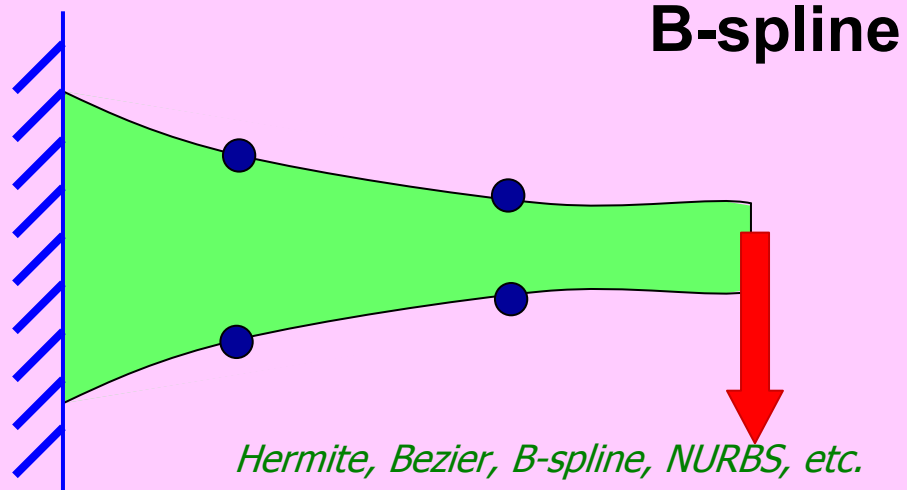


- Shape
Topology } are given

- **Optimize cross sections**

Shape Optimization

$$\begin{aligned} &\text{minimize } f(\mathbf{x}) \\ &\text{subject to } g(\mathbf{x}) \leq 0 \\ &\quad h(\mathbf{x}) = 0 \end{aligned}$$



Design variables (\mathbf{x})

\mathbf{x} : control points of the B-spline
(position of each control point)

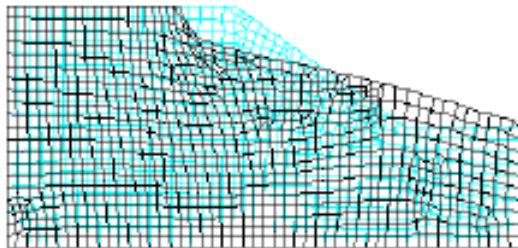
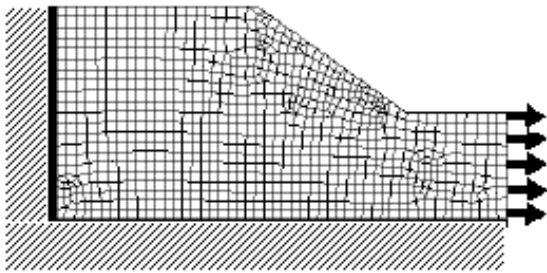
$f(\mathbf{x})$: compliance

$g(\mathbf{x})$: mass

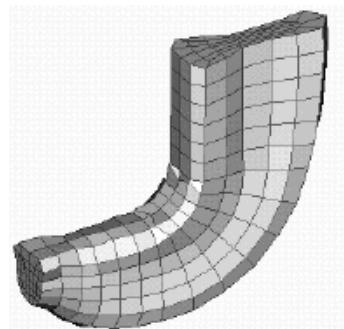
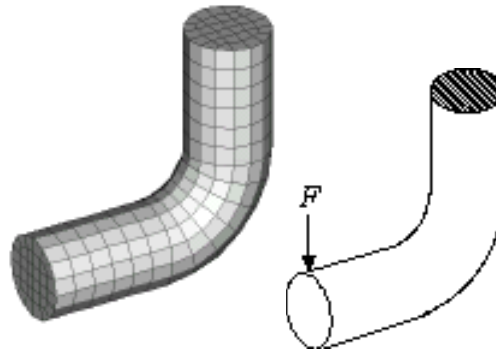
Number of design variables (ndv)

ndv = 8

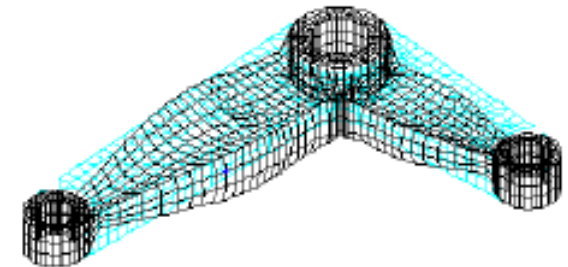
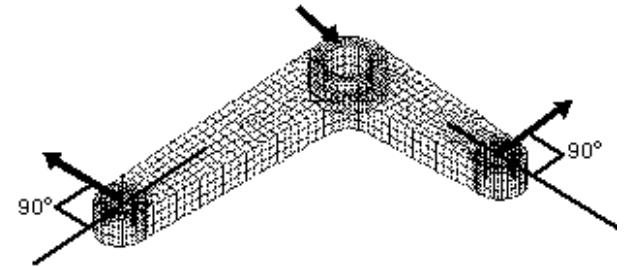
Fillet problem



Hook problem



Arm problem



Multiobjective & Multidisciplinary Shape Optimization

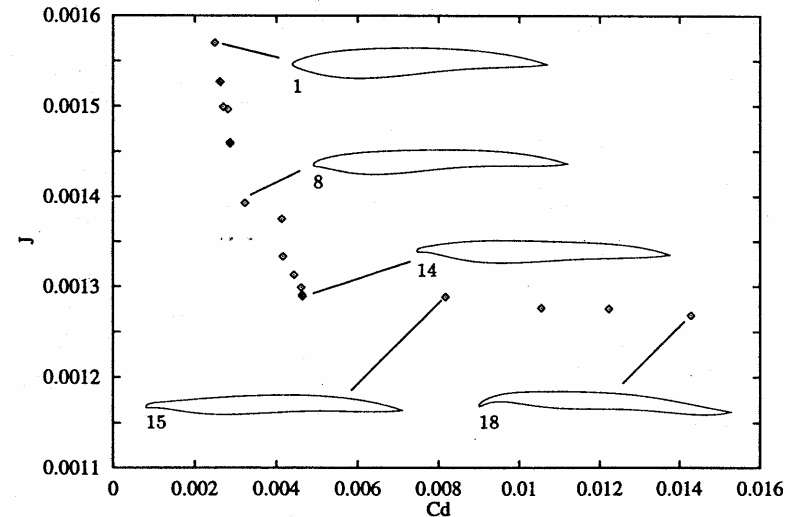
Objective function

1. Drag coefficient,
2. Amplitude of backscattered wave

Analysis

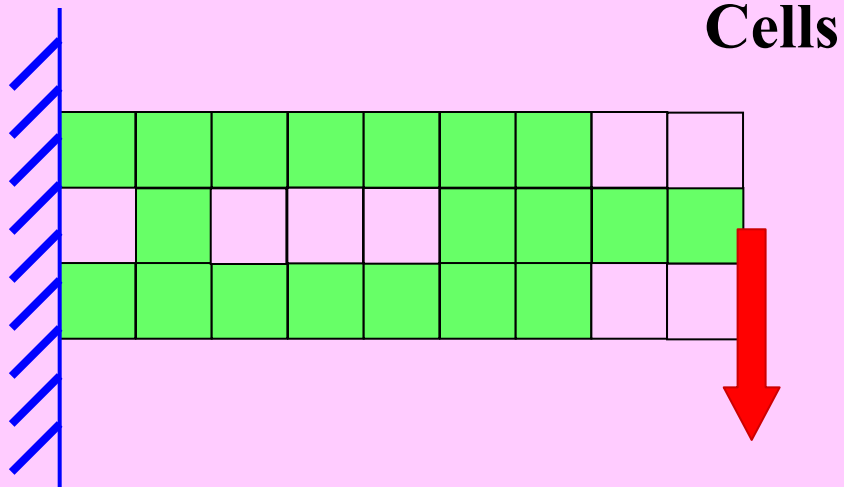
1. Computational Fluid Dynamics Analysis
2. Computational Electromagnetic Wave Field Analysis

Obtain Pareto Front



Raino A.E. Makinen et al., "Multidisciplinary shape optimization in aerodynamics and electromagnetics using genetic algorithms," International Journal for Numerical Methods in Fluids, Vol. 30, pp. 149-159, 1999

$$\begin{aligned} &\text{minimize } f(\mathbf{x}) \\ &\text{subject to } g(\mathbf{x}) \leq 0 \\ &\quad h(\mathbf{x}) = 0 \end{aligned}$$



Design variables (\mathbf{x})

\mathbf{x} : density of each cell

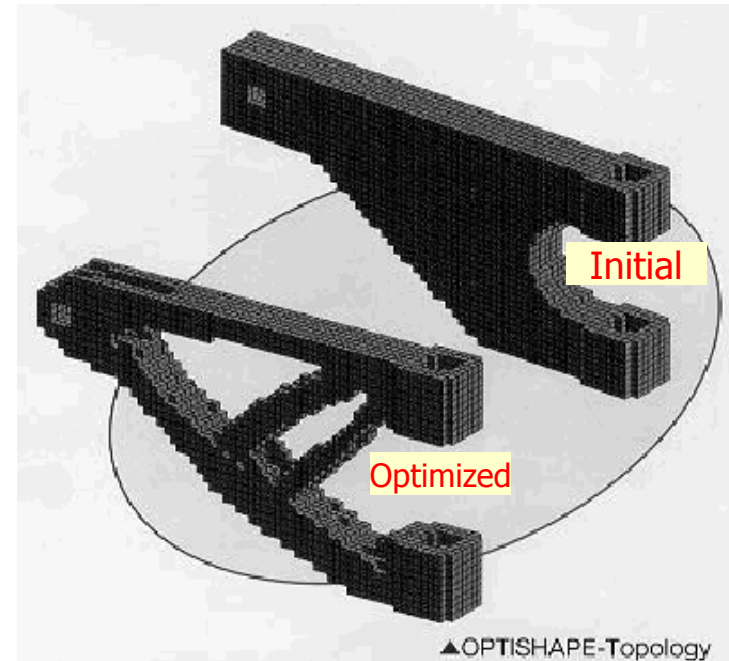
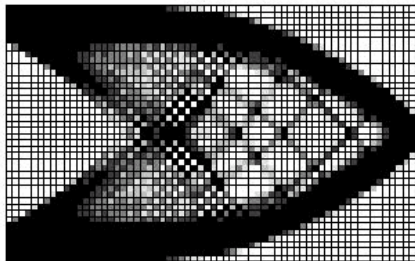
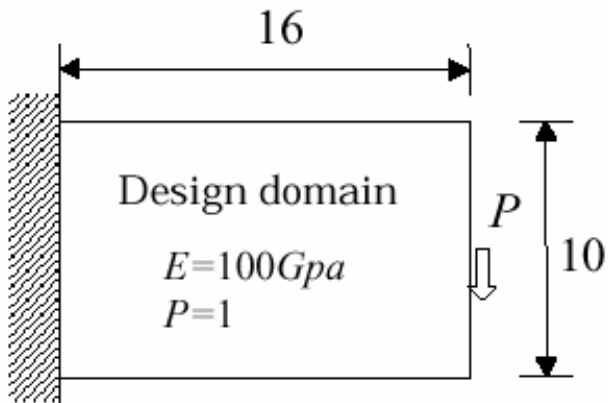
Number of design variables (ndv)

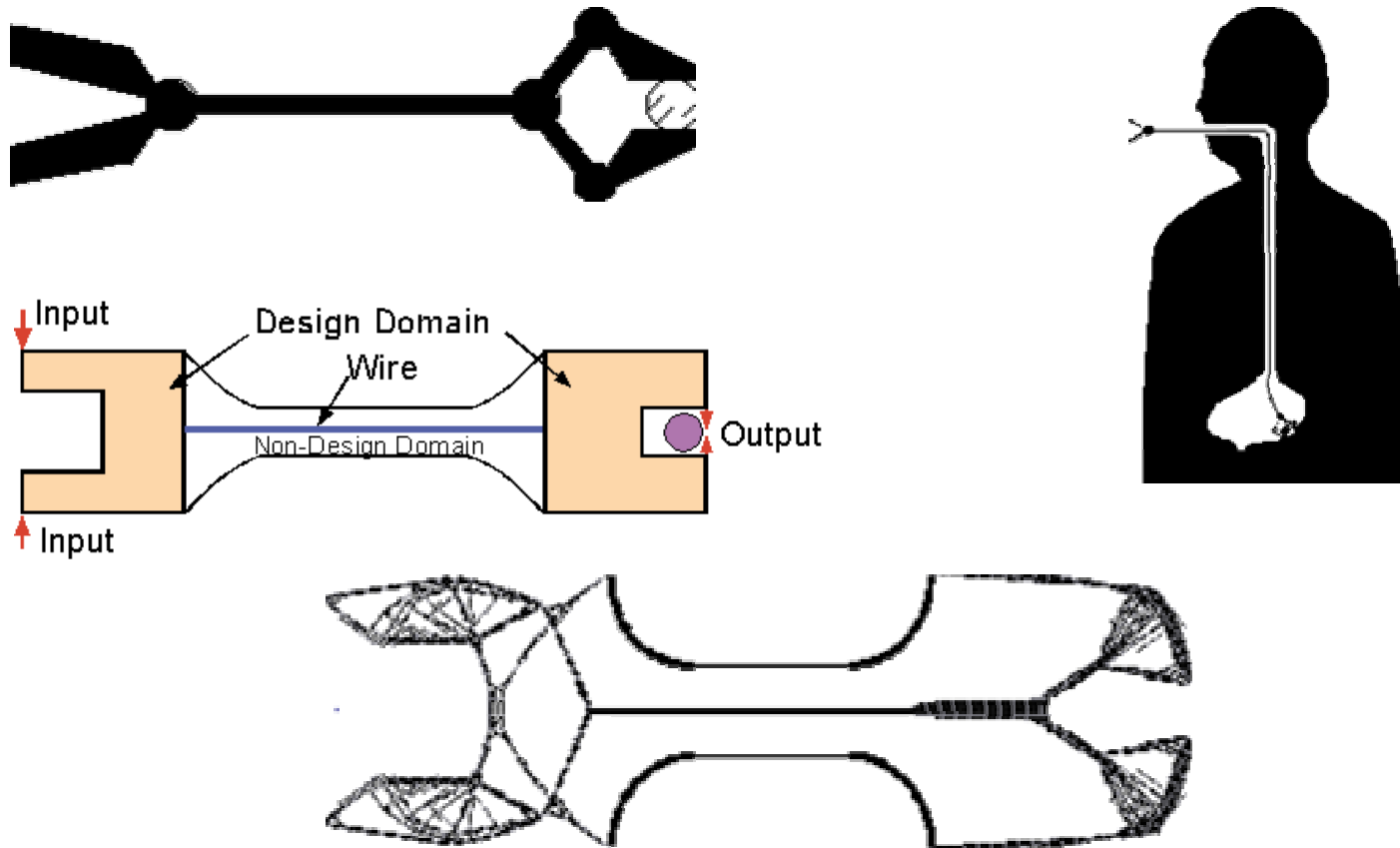
$$\text{ndv} = 27$$

$f(\mathbf{x})$: compliance

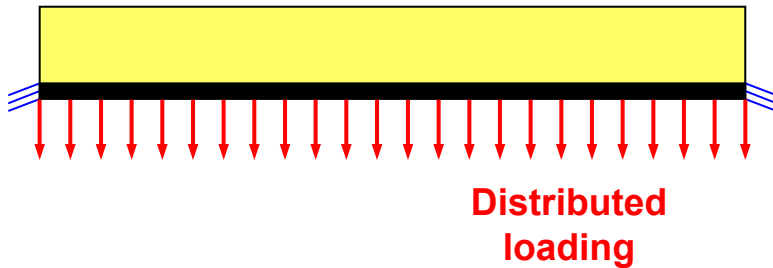
$g(\mathbf{x})$: mass

Short Cantilever problem





Bridge problem

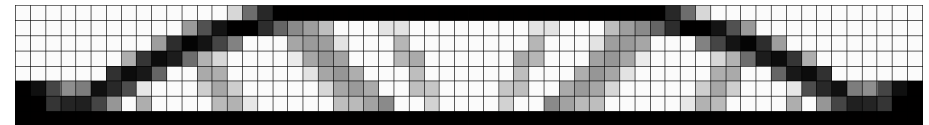


$$\text{Minimize } \int_{\Gamma} F^i z^i d\Gamma,$$

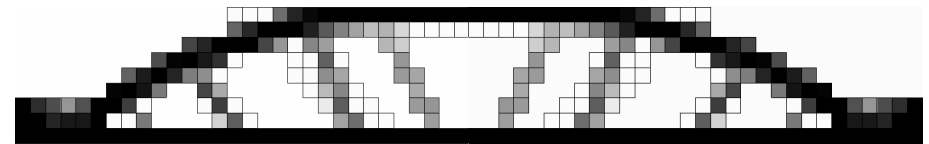
$$\text{Subject to } \int_{\Omega} \rho(x) d\Omega \leq M_o,$$

$$0 \leq \rho(x) \leq 1$$

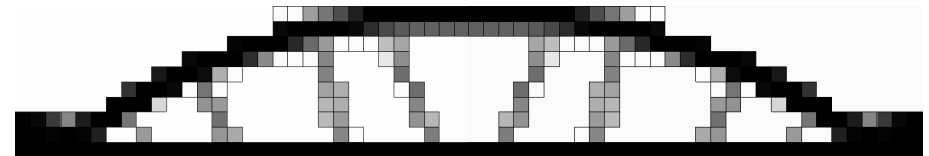
Mass constraints: 35%



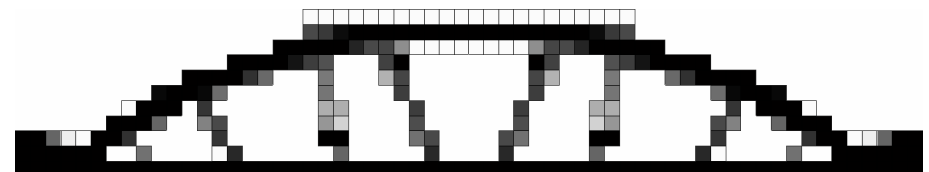
$$\text{Obj} = 4.16 \times 10^5$$



$$\text{Obj} = 3.29 \times 10^5$$

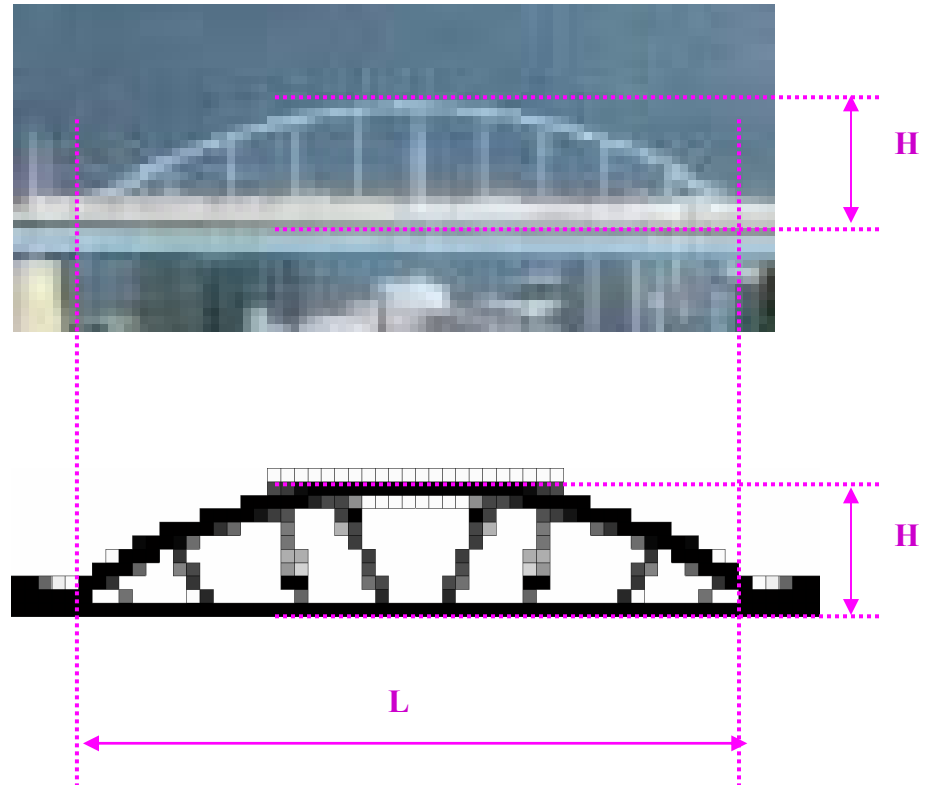


$$\text{Obj} = 2.88 \times 10^5$$



$$\text{Obj} = 2.73 \times 10^5$$

DongJak Bridge in Seoul, Korea



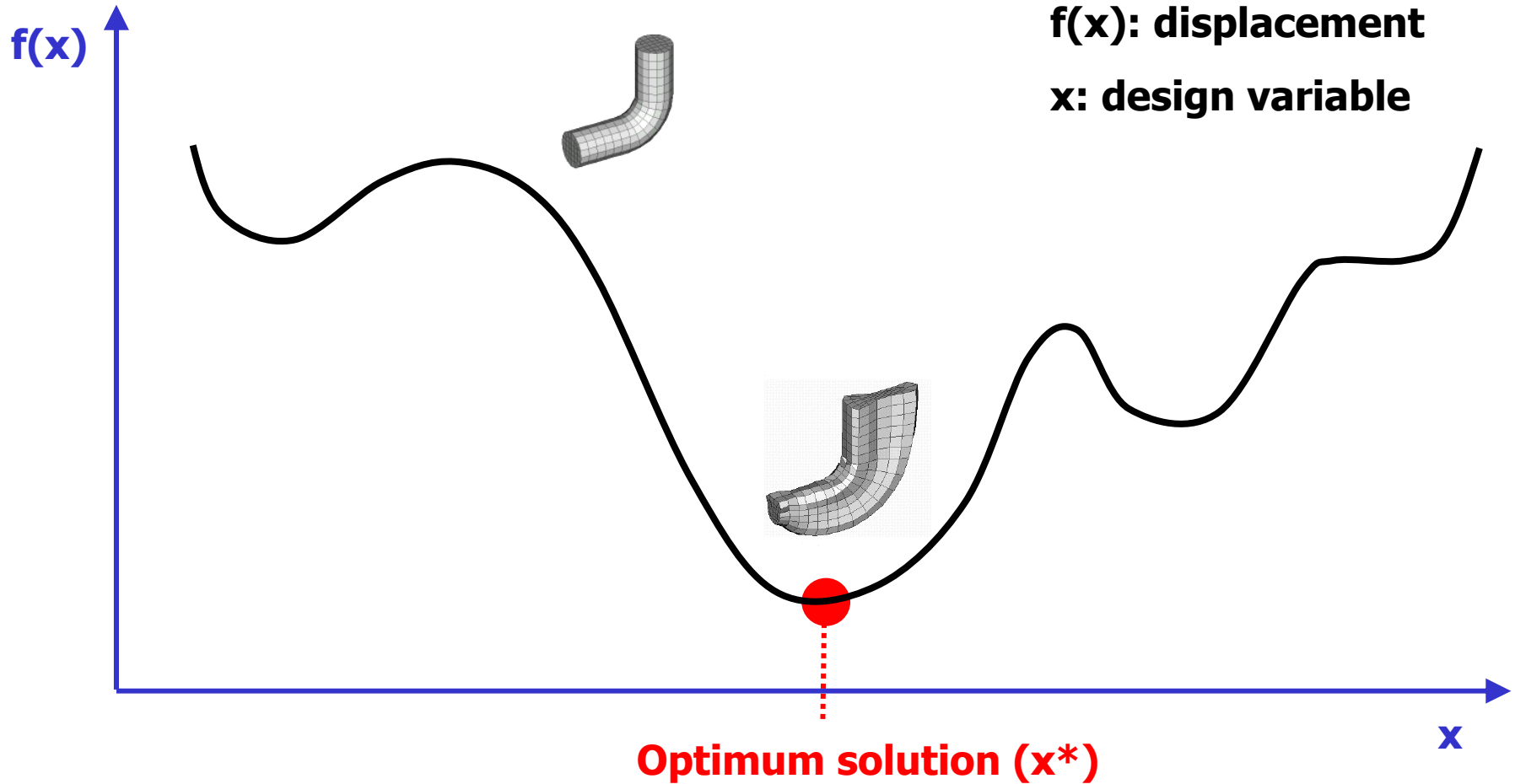
What determines the type of structural optimization?

Type of the design variable

(How to describe the design?)

Optimum Solution

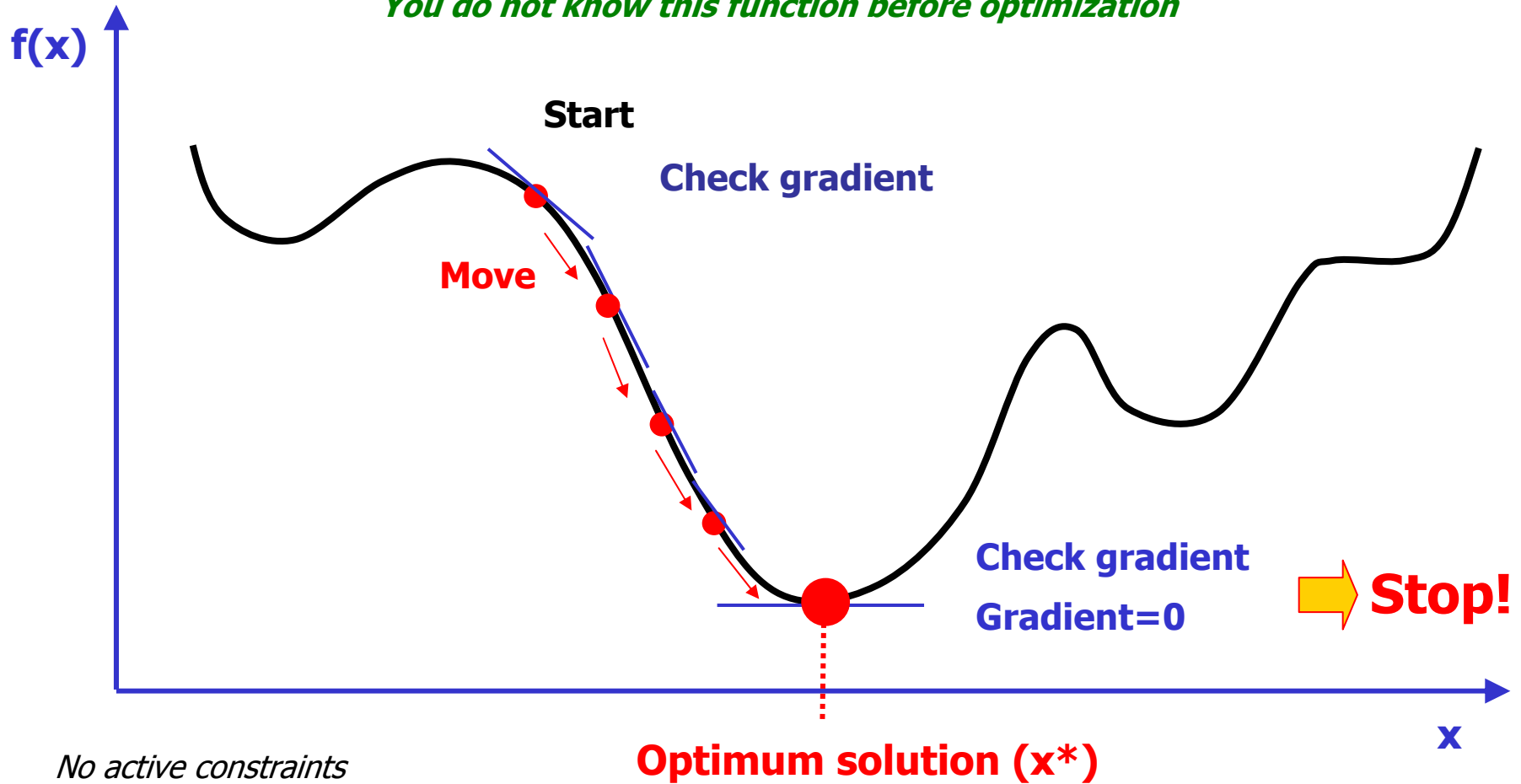
– Graphical Representation



Gradient-based methods

Heuristic methods

You do not know this function before optimization



(Termination criterion: Gradient=0)

Steepest Descent

UNCONSTRAINED

Conjugate Gradient

Quasi-Newton

Newton

Simplex – linear

CONSTRAINED

SLP – linear

SQP – nonlinear, expensive, common in engineering applications

Exterior Penalty – nonlinear, discontinuous design spaces

Interior Penalty – nonlinear

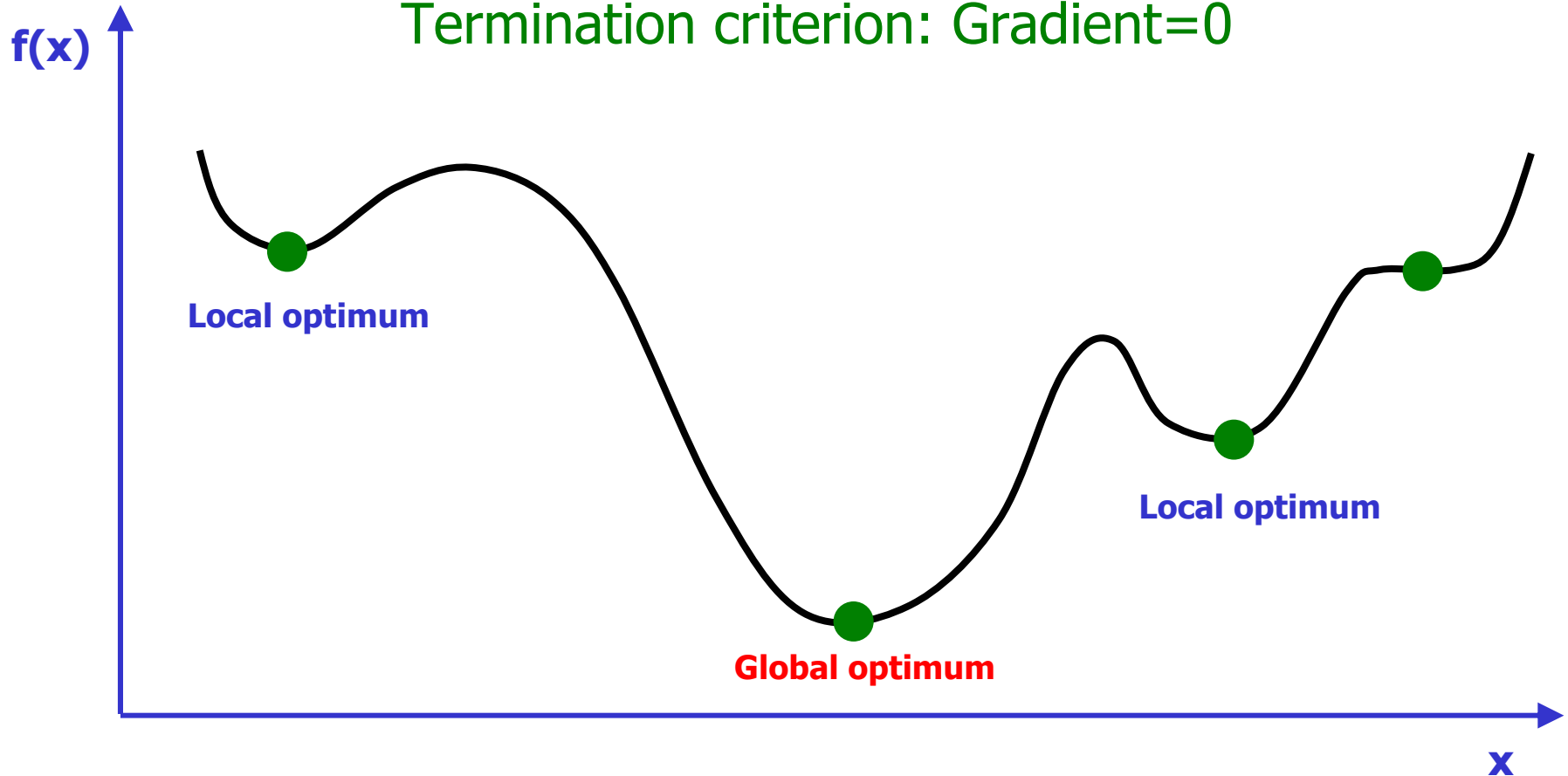
Generalized Reduced Gradient – nonlinear

Method of Feasible Directions – nonlinear

Mixed Integer Programming

Global optimum vs. Local optimum

Termination criterion: Gradient=0

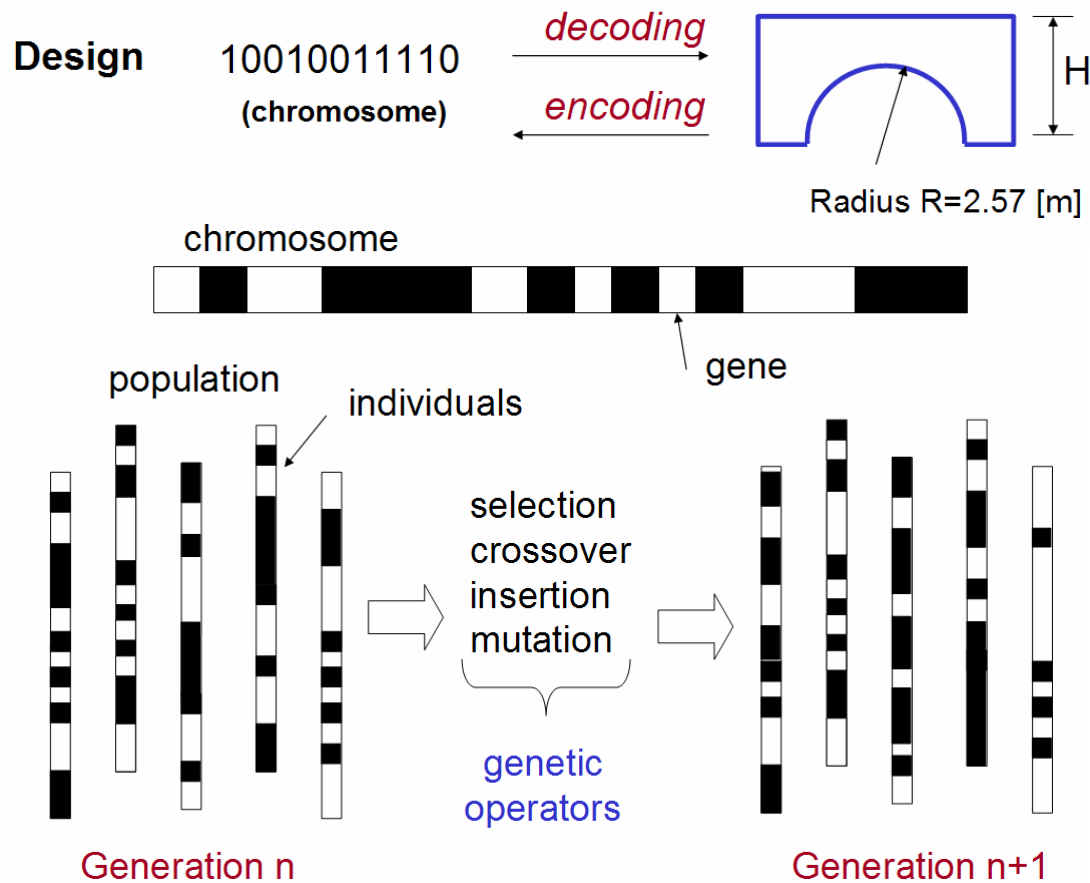


No active constraints

- A **Heuristic** is simply a **rule of thumb** that hopefully will find a good answer.
- **Why** use a Heuristic?
 - Heuristics are typically used to solve **complex optimization problems** that are difficult to solve to optimality.
- Heuristics are **good at dealing with local optima** without getting stuck in them while searching for the global optimum.

Schulz, A.S., "Metaheuristics," 15.057 Systems Optimization Course Notes, MIT, 1999.

Principle by Charles Darwin - Natural Selection



- Heuristics Often Incorporate Randomization
- **3 Most Common Heuristic Techniques**
 - Genetic Algorithms
 - Simulated Annealing
 - Tabu Search

- **iSIGHT**
- **DOT**
- **Matlab (fmincon)**

16.810 Topology Optimization Software

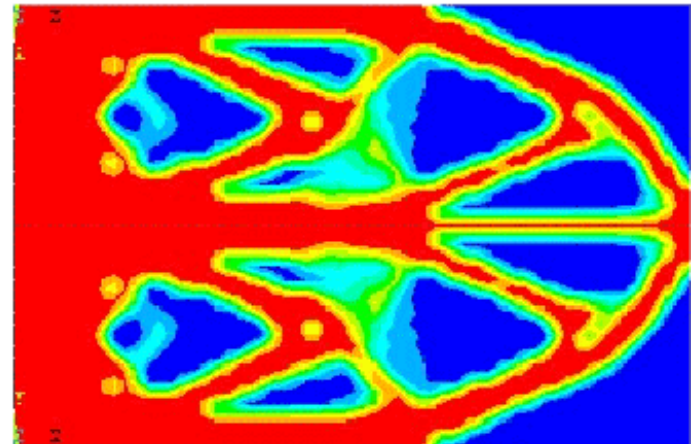
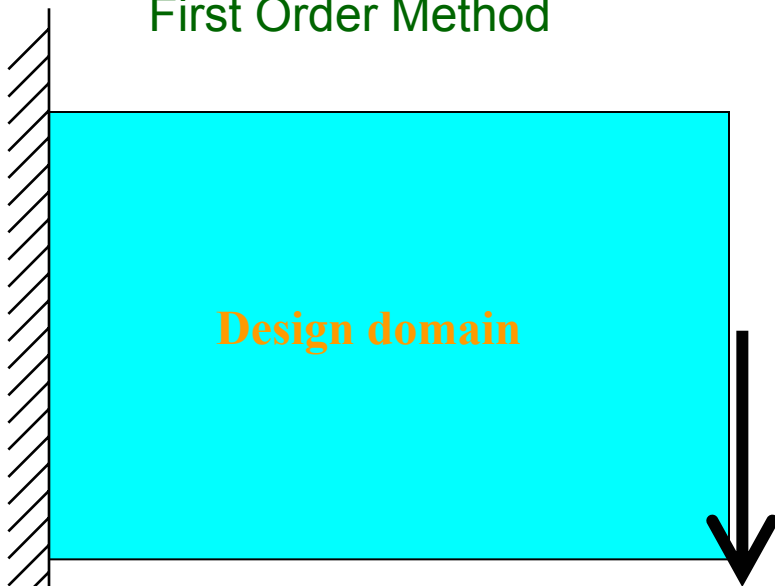
❖ ANSYS



- Static Topology Optimization
- Dynamic Topology Optimization
- Electromagnetic Topology Optimization

Subproblem Approximation Method

First Order Method



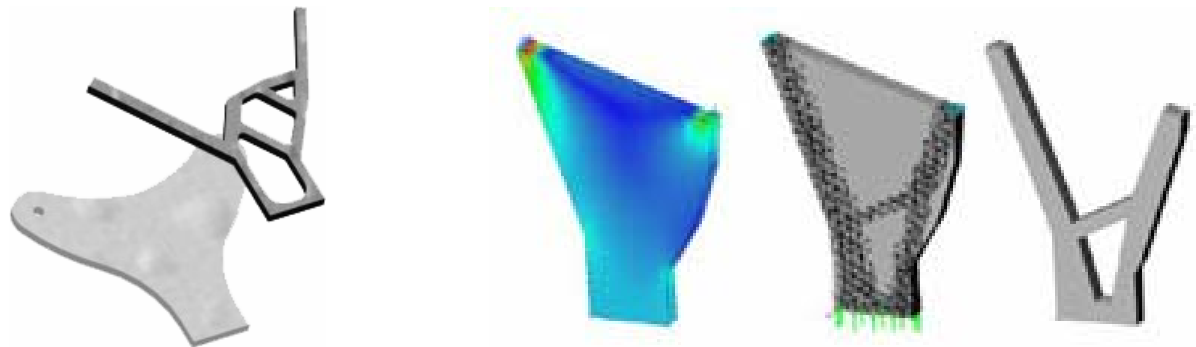
16.810 Topology Optimization Software

❖ MSC. Visual Nastran FEA



Elements of lowest stress are removed gradually.

Optimization results



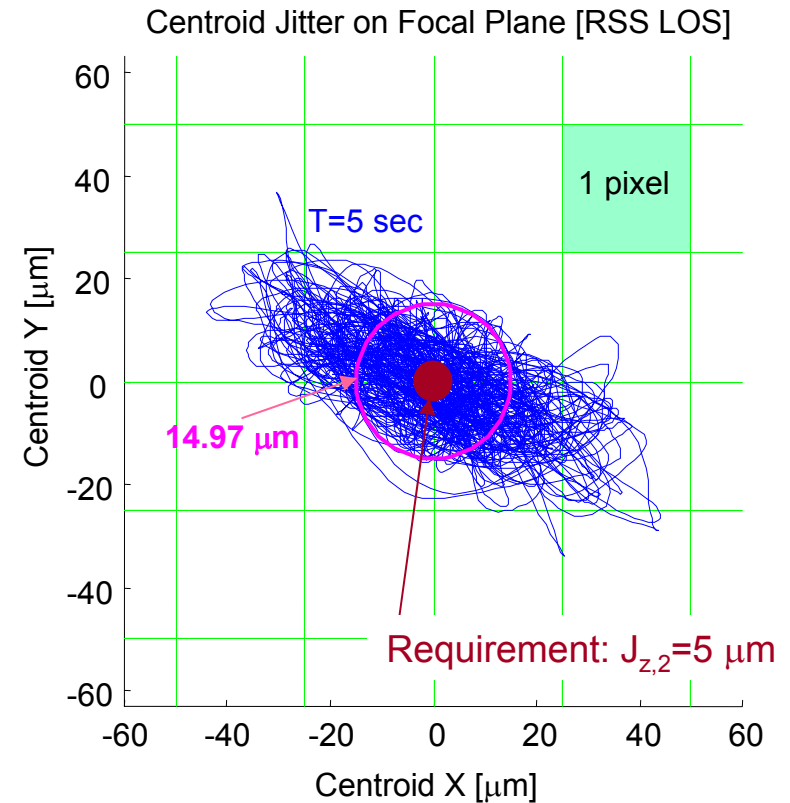
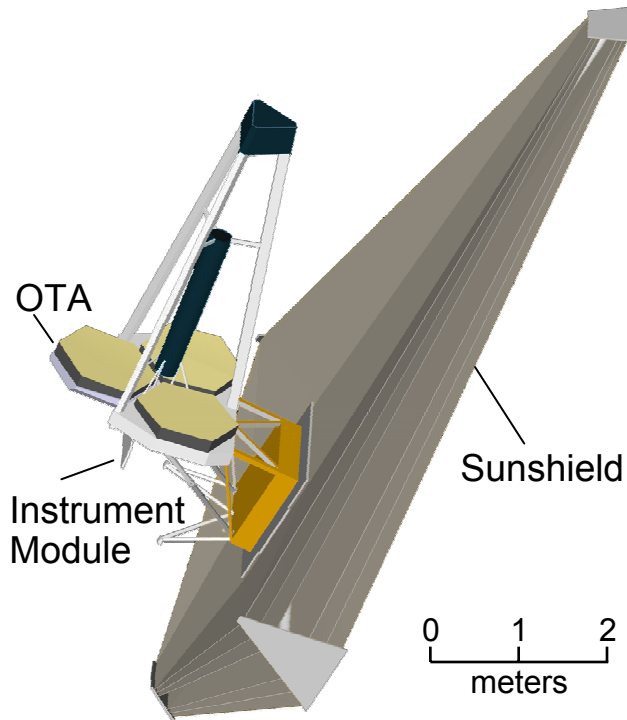
Optimization results illustration



Multidisciplinary Design Optimization

16.810 Multidisciplinary Design Optimization

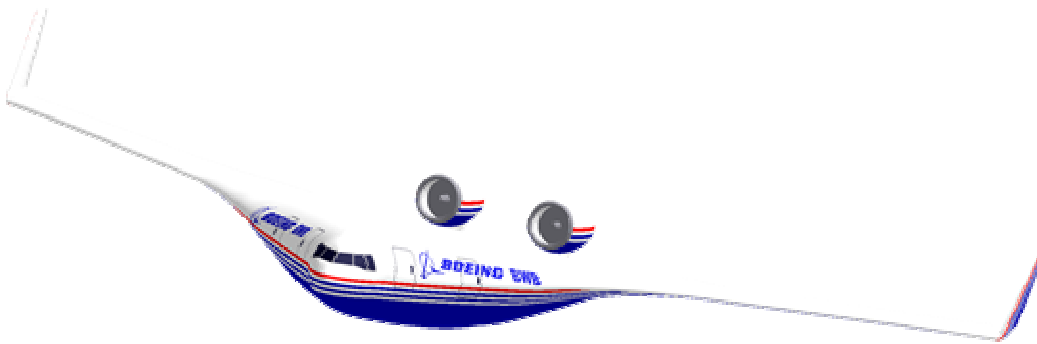
NASA Nexus Spacecraft Concept



Goal: Find a “balanced” system design, where the flexible structure, the optics and the control systems work together to achieve a desired pointing performance, given various constraints

16.810 Multidisciplinary Design Optimization

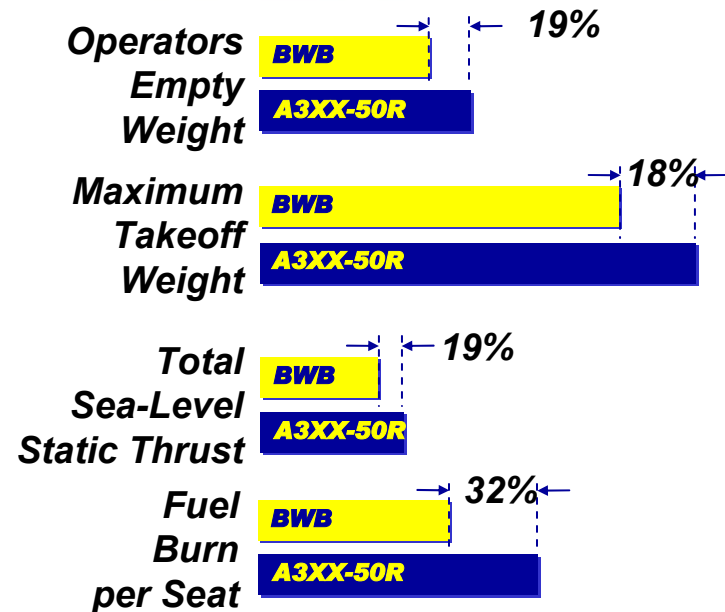
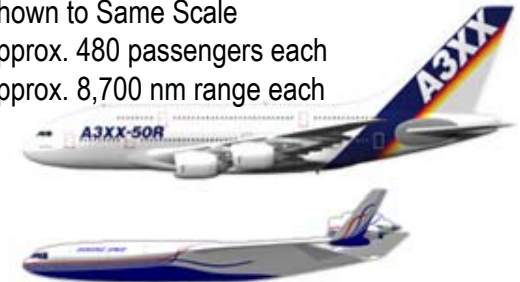
Boeing Blended Wing Body Concept



Goal: Find a design for a family of blended wing aircraft that will combine aerodynamics, structures, propulsion and controls such that a competitive system emerges - as measured by a set of operator metrics.

Aircraft Comparison

Shown to Same Scale
Approx. 480 passengers each
Approx. 8,700 nm range each



© Boeing

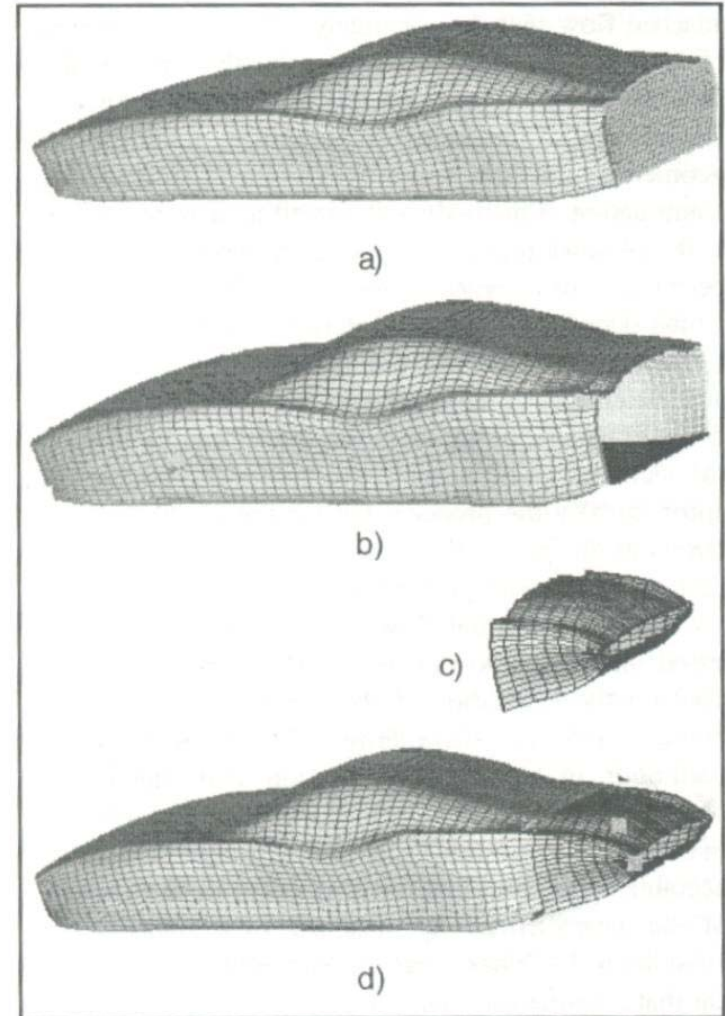
16.810 Multidisciplinary Design Optimization

Ferrari 360 Spider



Goal: High end vehicle shape optimization while improving car safety for fixed performance level and given geometric constraints

Reference: G. Lombardi, A. Vicere, H. Paap, G. Manacorda, "Optimized Aerodynamic Design for High Performance Cars", AIAA-98-4789, MAO Conference, St. Louis, 1998



16.810 Multidisciplinary Design Optimization

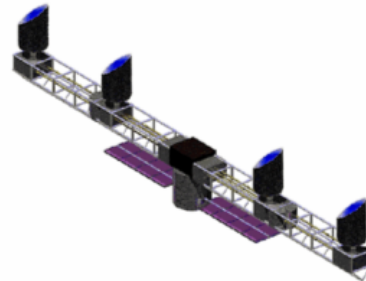
Aircraft:

Aerodynamics
Propulsion
Structures
Controls
Avionics/Software
Manufacturing
others



Spacecraft:

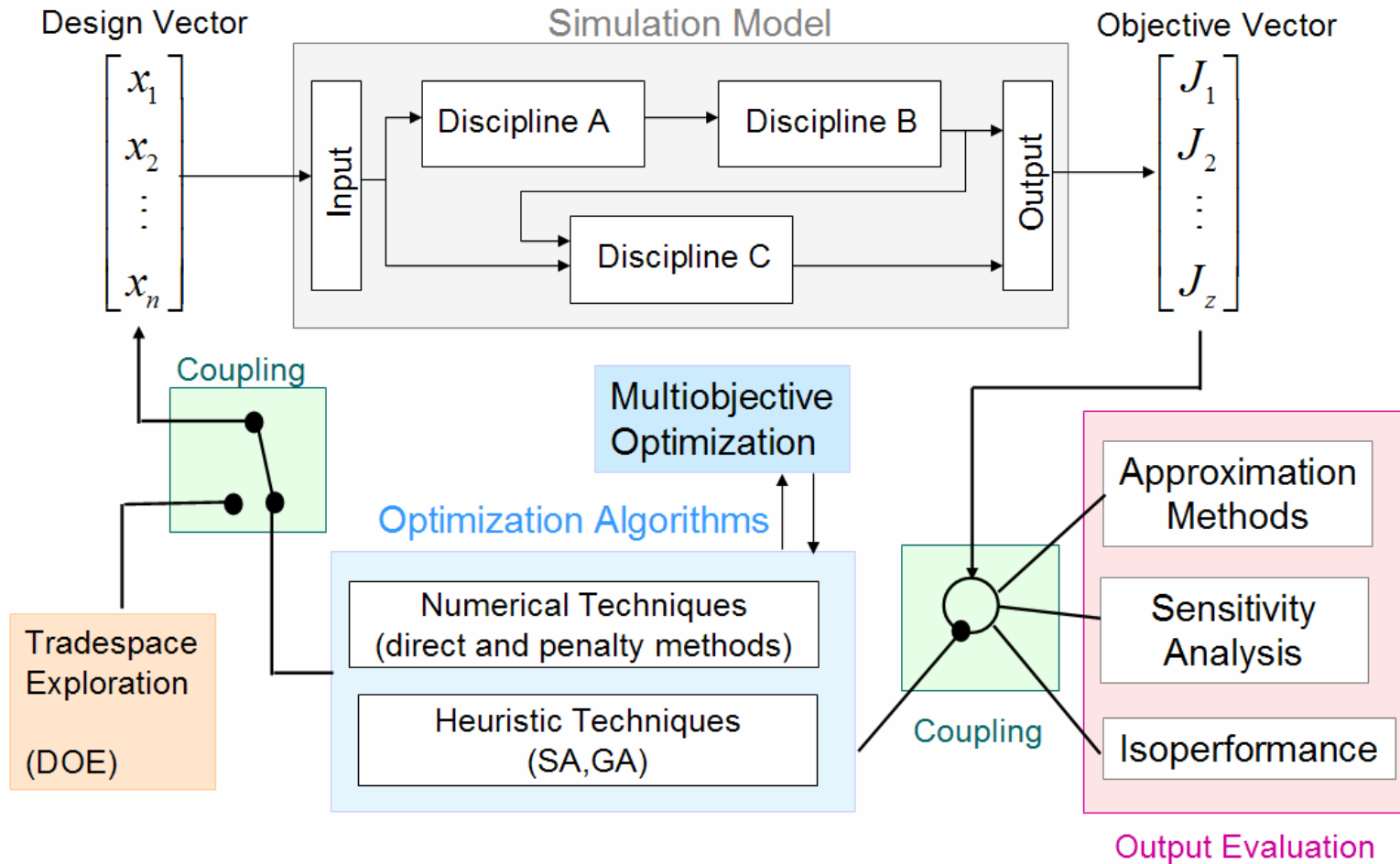
Astrodynamics
Thermodynamics
Communications
Payload & Sensor
Structures
Optics
Guidance & Control



Automobiles:

Engines
Body/chassis
Aerodynamics
Electronics
Hydraulics
Industrial design
others





16.810 Multidisciplinary Design Optimization

Do you want to learn more about MDO?

Take this course!

16.888/ESD.77

**Multidisciplinary System
Design Optimization (MSDO)**

Prof. Olivier de Weck

Prof. Karen Willcox

Do you want to learn
more about GA?

Take part in this GA
game experiment!

Learn about Genetic Algorithms and earn money during IAP...

Take part in a pedagogical experiment (requiring only 2 hours of your time) where you will learn about Genetic Algorithms and earn **\$20** for your participation!

When: Thursday, January 29th

1:00 p.m.- 3:00 p.m. (Group 1)

3:00 p.m.- 5:00 p.m. (Group 2)

Where: 33-419

Who may participate: Any undergraduate engineering student at MIT

Please contact Jackie Dilley at 324-0092 if you are interested.

Spaces are filling up fast, so please contact us soon!

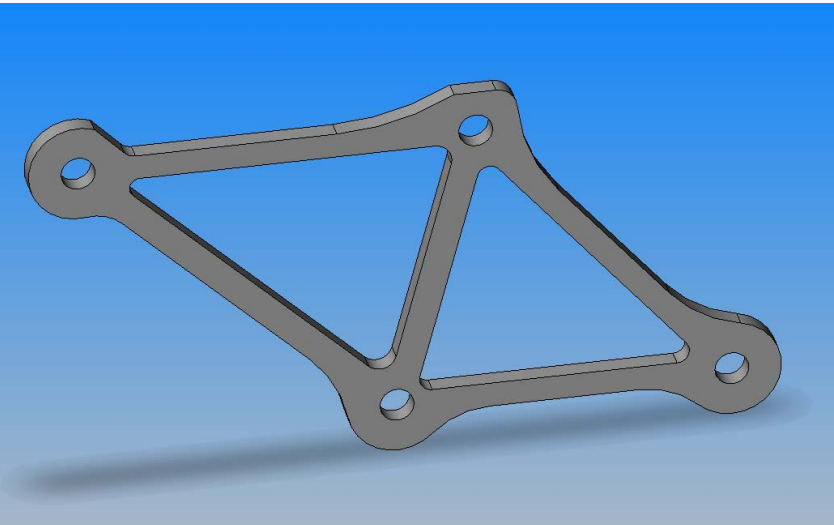
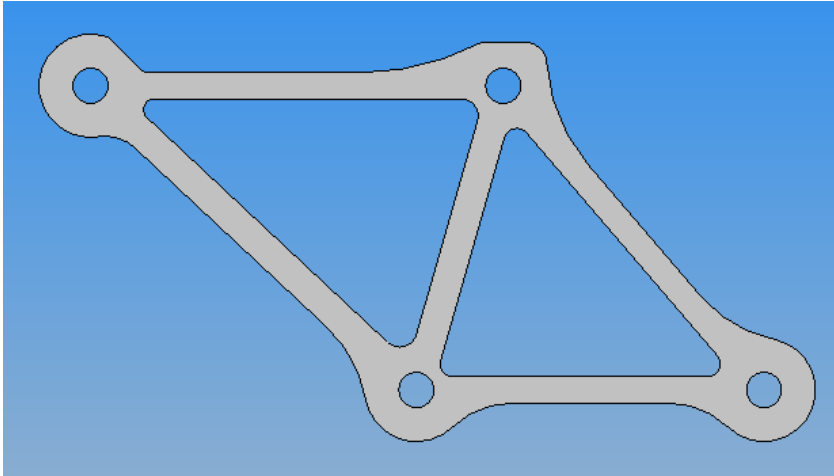
Refreshments will be served!

Performance

Natural frequency analysis

Design requirements

Baseline Design



Performance and cost

$$\delta_1 = 0.070 \text{ mm}$$

$$\delta_2 = 0.011 \text{ mm}$$

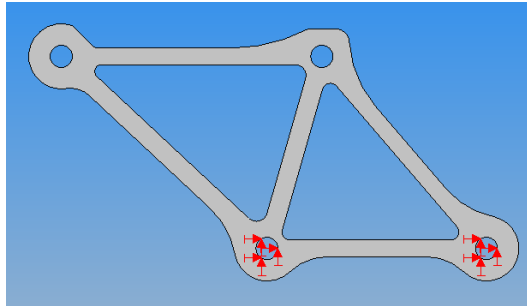
$$f = 245 \text{ Hz}$$

$$m = 0.224 \text{ lbs}$$

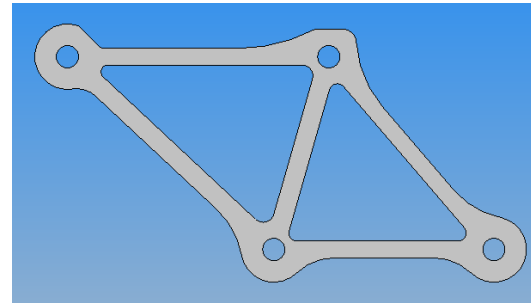
$$C = 5.16 \text{ \$}$$

Baseline Design

245 Hz



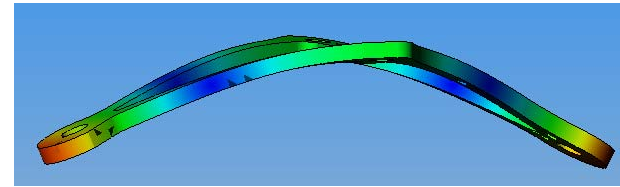
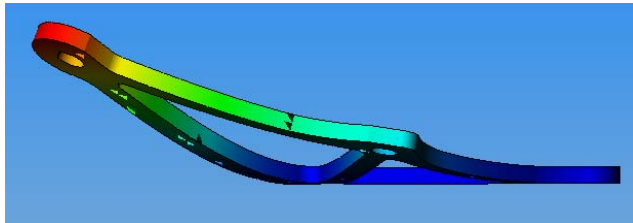
421 Hz



$f_1=245$ Hz

$f_2=490$ Hz

$f_3=1656$ Hz



$f_1=0$

$f_2=0$

$f_3=0$

$f_4=0$

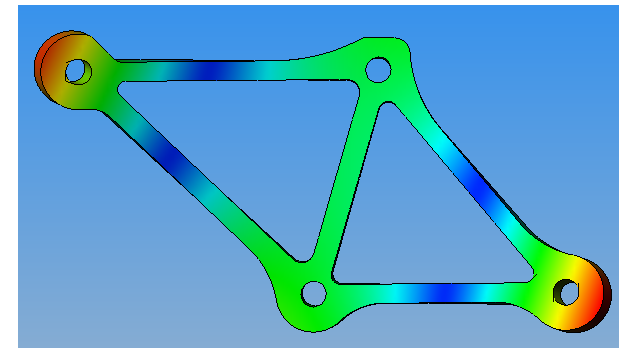
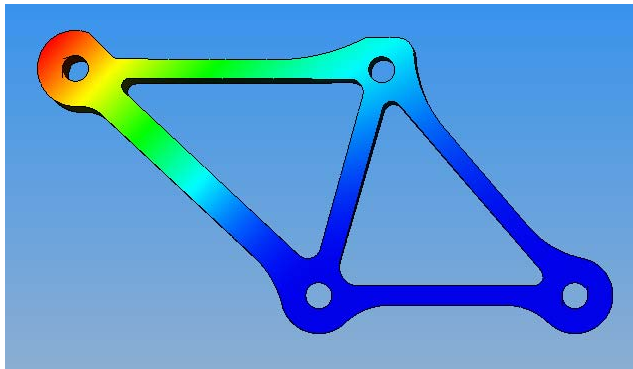
$f_5=0$

$f_6=0$

$f_7=421$ Hz

$f_8=1284$ Hz

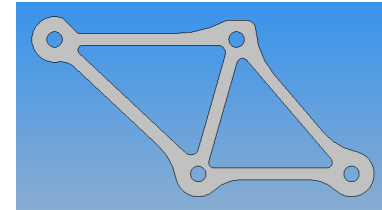
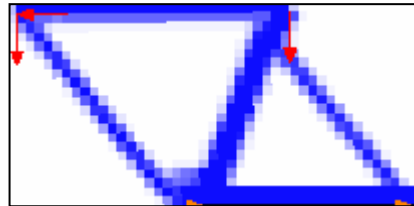
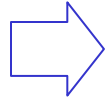
$f_9=1310$ Hz



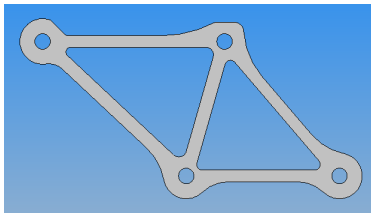
Design Requirement for Each Team

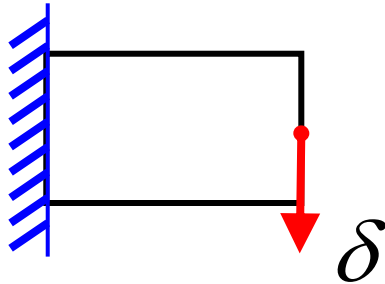
#	Product name	mass (m)	Cost (c)	Disp (δ_1)	Disp (δ_2)	Nat Freq (f)	Quality	F1 (lbs)	F2 (lbs)	F3 (lbs)	Const	Optim	Acc
0	Base line	0.224 lbs	5.16 \$	0.070 mm	0.011 mm	245 Hz	3	50	50	100	c	m	δ_1, δ_2, f
1	Family economy	20%	-30%	10%	10%	-20%	2	50	50	100	c	m	δ_1, δ_2, f
2	Family deluxe	10%	-10%	-10%	-10%	10%	4	50	50	100	m	c	δ_1, δ_2, f
3	Cross over	20%	0%	-15%	-15%	20%	4	50	75	75	m	c	δ_1, δ_2, f
4	City bike	-20%	-20%	0%	0%	0%	3	50	75	75	c	m	δ_1, δ_2, f
5	Racing	-30%	50%	0%	0%	20%	5	100	100	50	m	δ_1, δ_2, f	c
6	Mountain	30%	30%	-20%	-20%	30%	4	50	100	50	δ_1, δ_2, f	m	c
7	BMX	0%	65%	-15%	-15%	40%	4	75	100	75	δ_1, δ_2, f	m	c
8	Acrobatic	-30%	100%	-10%	-10%	50%	5	100	100	100	δ_1, δ_2, f	m	c
9	Motor bike	50%	10%	-20%	-20%	0%	3	50	75	100	δ_1, δ_2, f	c	m

Topology optimization

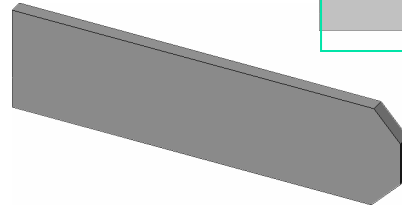


Shape optimization



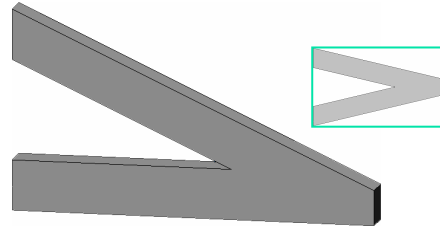


1 bar



$$\delta = 2.50 \text{ mm}$$

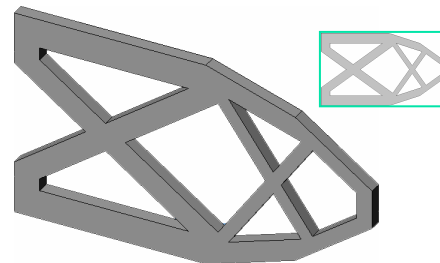
2 bars



$$\delta = 0.80 \text{ mm}$$

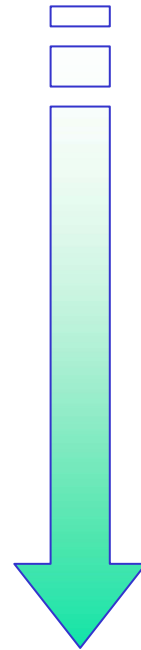
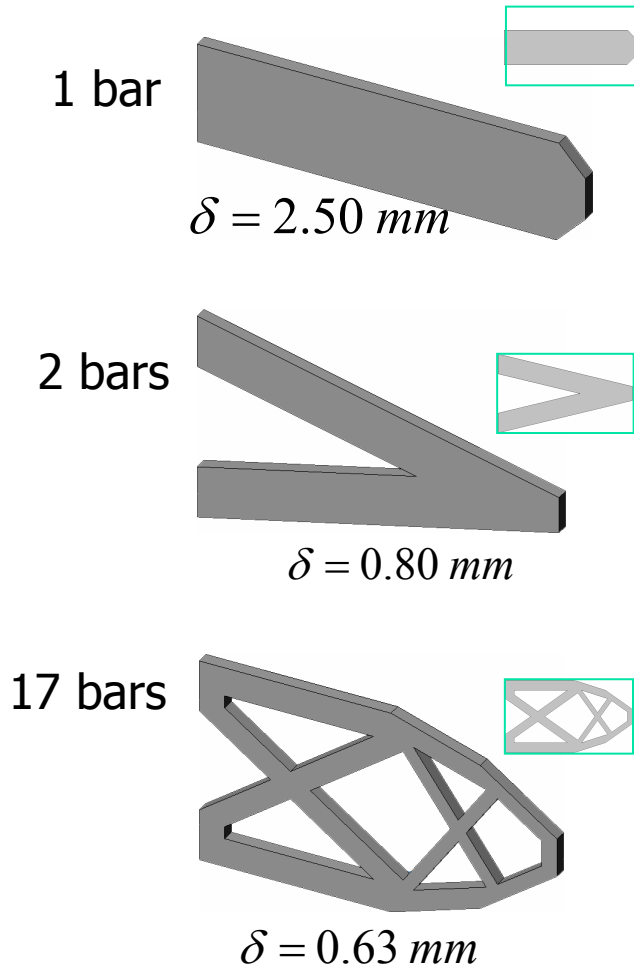
Volume is the same.

17 bars

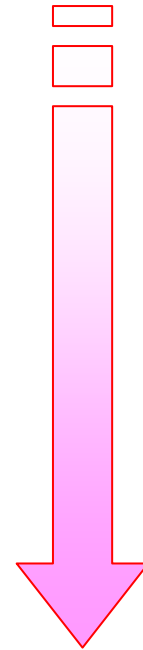


$$\delta = 0.63 \text{ mm}$$

Design Freedom

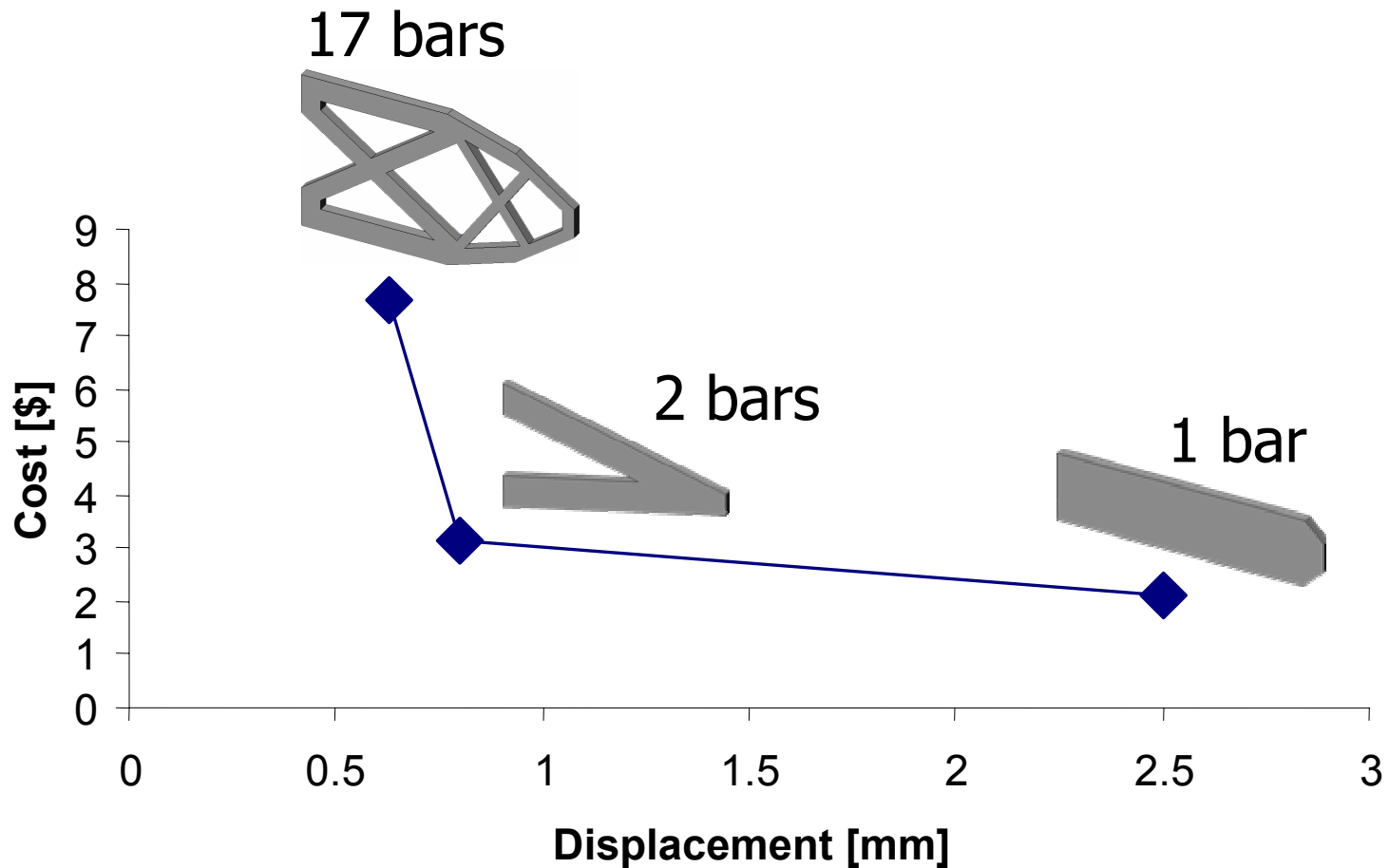


**More design freedom
(Better performance)**



**More complex
(More difficult to optimize)**

16.810 Cost versus Performance



Class Survey

Jan 24 (Saturday) 7 am – Jan 26 (Monday) 11am

Company tour

Jan 26 (Monday) : 1 pm – 4 pm

Guest Lecture (Prof. Wilson, Bicycle Science)

Jan 28 (Wednesday) : 2 pm – 3:30 pm

Manufacturing Bicycle Frames (Version 2)

Jan 28 (Wednesday) : 9 am – 4:30 pm

Jan 29 (Thursday) : 9 am – 12 pm

Testing

Jan 29 (Thursday) : 10 am – 2 pm

GA Games

Jan 29 (Thursday) : 1 pm – 5 pm

Guest Lecture, Student Presentation (5~10 min/team)

Jan 30 (Friday) : 1 pm – 4 pm

P. Y. Papalambros, Principles of optimal design, Cambridge University Press, 2000

O. de Weck and K. Willcox, Multidisciplinary System Design Optimization, MIT lecture note, 2003

M. O. Bendsoe and N. Kikuchi, "Generating optimal topologies in structural design using a homogenization method," comp. Meth. Appl. Mech. Engng, Vol. 71, pp. 197-224, 1988

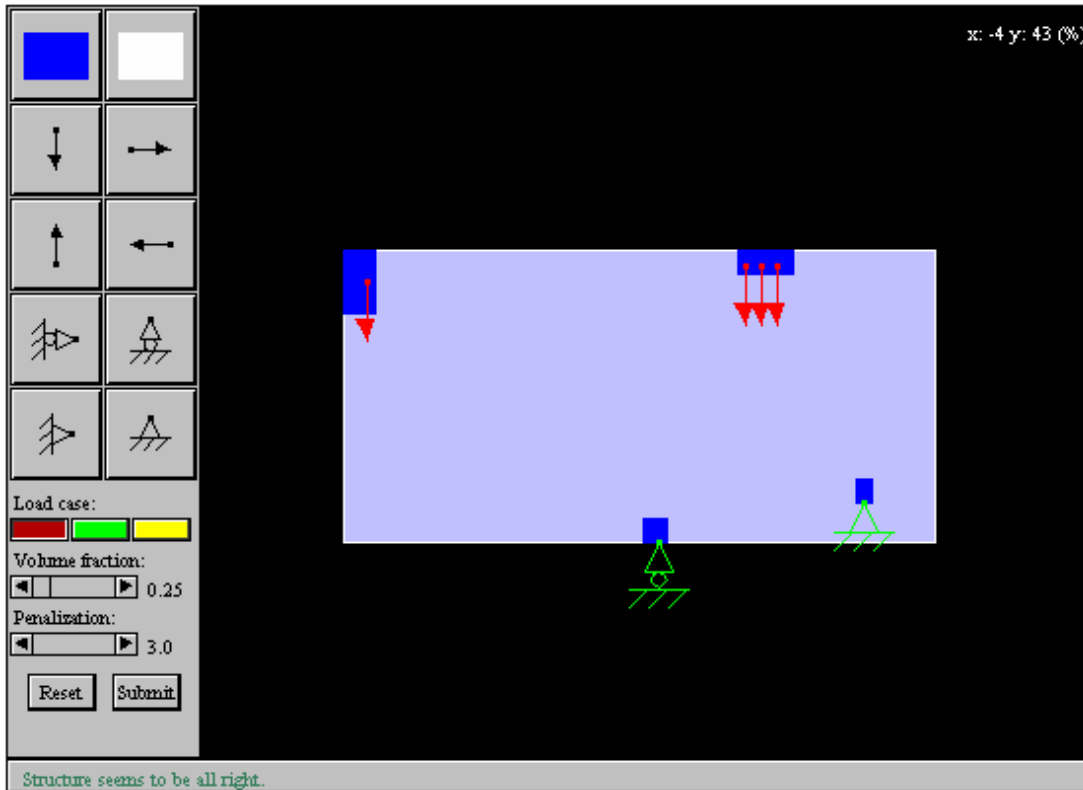
Raino A.E. Makinen et al., "Multidisciplinary shape optimization in aerodynamics and electromagnetics using genetic algorithms," International Journal for Numerical Methods in Fluids, Vol. 30, pp. 149-159, 1999

Il Yong Kim and Byung Man Kwak, "Design space optimization using a numerical design continuation method," International Journal for Numerical Methods in Engineering, Vol. 53, Issue 8, pp. 1979-2002, March 20, 2002.

Developed and maintained by [Dmitri Tcherniak](#), [Ole Sigmund](#), [Thomas A. Poulsen](#) and [Thomas Buhl](#).

Features:

- 1.2-D
2. Rectangular design domain
3. 1000 design variables (1000 square elements)
4. Objective function: compliance ($F \times \delta$)
5. Constraint: volume



Objective function

-Compliance ($F \times \delta$)

Constraint

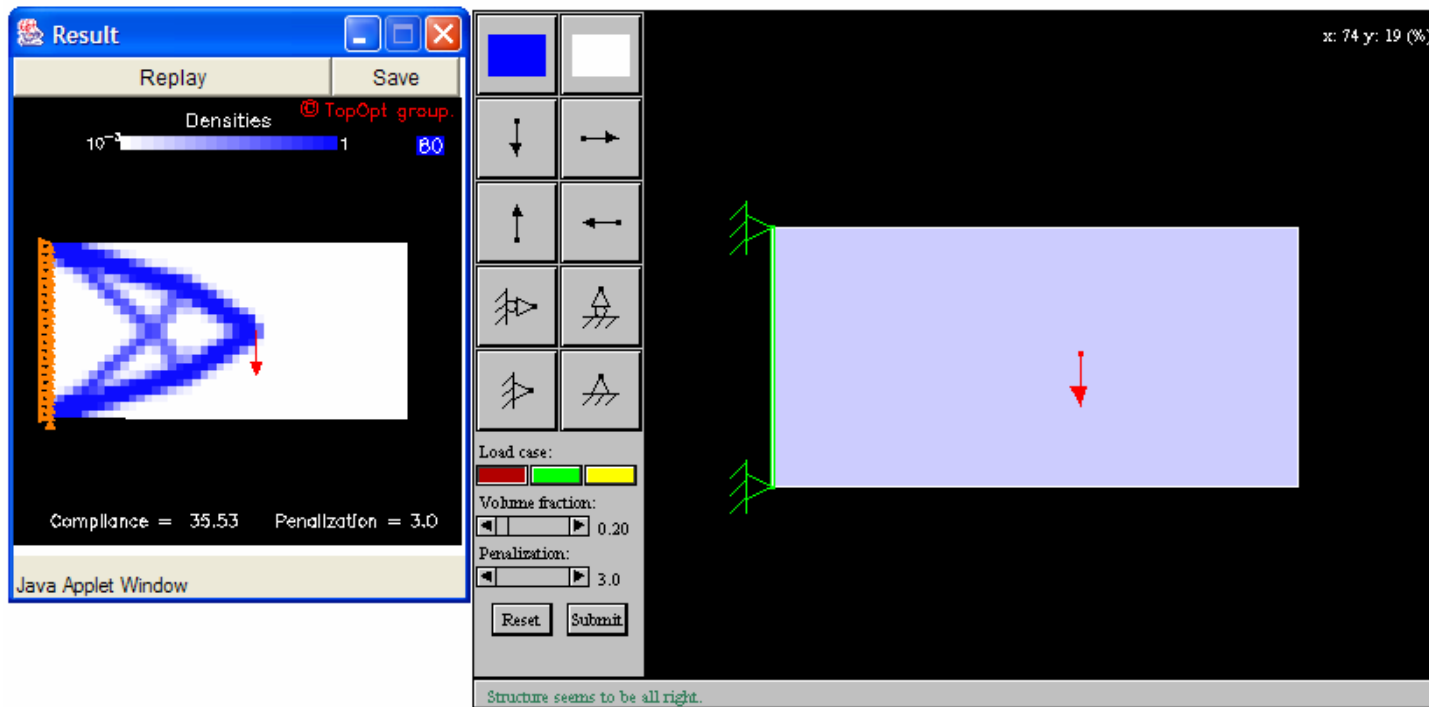
-Volume

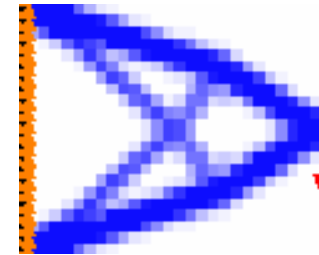
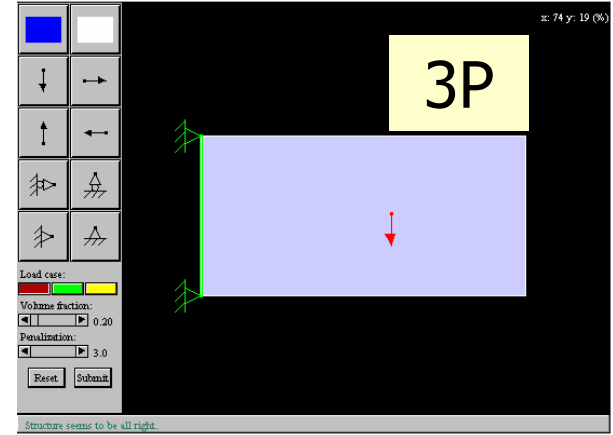
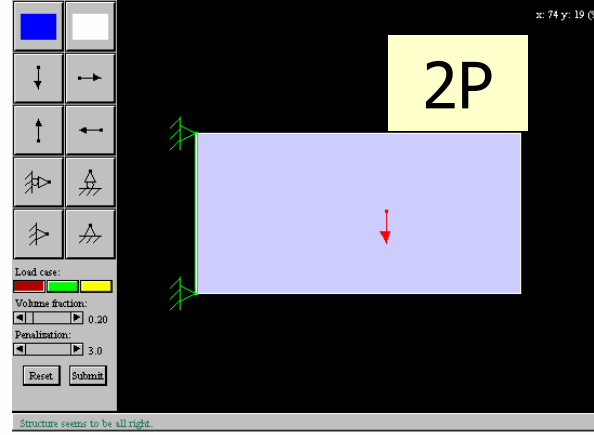
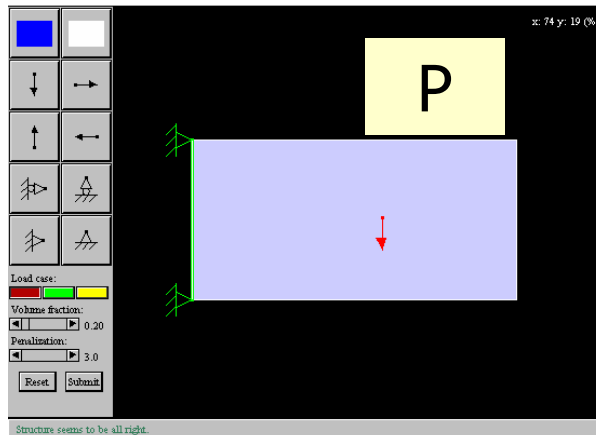
Design variables

- Density of each design cell

No numerical results are obtained.

Optimum layout is obtained.





Absolute magnitude of load does not affect optimum solution



Web-based topology optimization program

<http://www.topopt.dtu.dk>