UE Fluids Problem 1 Solution Fall 06 a) weight = mg =  $0.175 \text{ kg} \cdot 9.8 \text{ m/s}^2 = 1.715 \text{ N}$ \_ ∆þ<sub>avg</sub> weight = lift =  $Ap_{arg} \cdot \frac{\pi}{4}d^2 = ing$  $Aren = \frac{1}{4}d^2$  $\Rightarrow \Delta P_{avg} = \frac{mg}{\frac{17}{7}d^2} = \frac{1.715N}{\frac{17}{7}(0.27m)^2} = 29.95 P_a$ d = 0,27 m b) Assume = pVas = spars  $\overline{V_{00}}_{min} = \sqrt{\frac{2}{P}} \frac{2}{P} = \sqrt{\frac{2}{1.22}} \frac{29.95 P_{a}}{1.22 \text{ kg/m}^{3}} = \overline{7.0 \text{ m/s}}$ =15,6 mph

Fall 06 UE Fluids Problem 2  $dp/dy = -pg = -(p_0 + (p_1 - p_0)\frac{y}{h}]g$ a) Hydrostatic Eq'n:  $dp = -(p_0 + (p_1 - p_0) \frac{y}{h})g dy$  $p = - \left[ p_{0}g + (p_{1} - p_{0}) \frac{1}{2} \frac{g^{2}}{h} \right] g + C$  $p_{1} = -[\rho_{0}h + (\rho_{1}-\rho_{0}) \frac{1}{2}h]g + C = -[\frac{1}{2}(\rho_{1}+\rho_{0})h]g + C$ Evaluate C: At y=h: p(y) = P, + ± (P, + Po) hg - [Poy + (P, -Po) = h]g | quadratic in y b) Method 1:  $F = (p_e - p_u)A$ ,  $p_e = p(y=4.5)$ ,  $p_u = p(y=5.5)$ ,  $A = lcu^2$ The expression gets a bit messy ...  $y^4$ UN Pr TT PL Method 2: Use Archimedes' Principle F = weight of displaced fluid = Parg. V.g Pu Pe For cube contered on y = 5 cm = h/2,  $\rho_{avg} = \frac{1}{2}(\rho_0 + \rho_1) = 1.1 \text{ g/cm}^3$  $F = \frac{1.1 \text{ g/cm}^3 \cdot 1 \text{ cm}^3}{980 \text{ cm/s}^2} = 1078 \text{ dyn} = 0.01078 \text{ N}$ Ci Po C) Look for y position where  $F = weight = wig = 1.15_{g} \cdot 980 \text{ cm/s}^2 = 1127 \text{ dyn}$ Or equivalently, where  $P(y) = Parg = Pcube = 1.15_{g}/cm^3$ By inspection,  $\rho = 1.15 \text{ g/cm}^3$  accurs at  $y = \frac{1}{4}h = 2.5 \text{ cm}$ h 4 1.0 1.1 1.2 1.5 Cube will come to rest centered on y= 2.5 cm

Fall 06 DE Fluids Problem 3 Civ a) Anderson 1.5.  $N = 12^{\circ}$   $C_{\mu} = 1.2$   $C_{\mu} = 0.03$ 120 1  $C_{1} = C_{N} \cos \alpha - C_{A} \sin \alpha = 1.1675$  $\overline{V}_{\infty}$ C14  $C_{D} = C_{W} \sin \kappa + C_{A} \cos \kappa = 0.2788$  $c_{\mathfrak{d}}$ b) Anderson 1.6: Using moment-translation formula: Mar = MLE + GLZ or  $C_{m_{cl_{\alpha}}} = C_{m_{LE}} + \frac{1}{4}C_{l}$  $\frac{C_m}{2} = \frac{C_m}{c_{l_4}} = \frac{1}{4}$ Equation for Xo: 3 Xo = -Mit/L 3 t  $cr \chi_{q}/c = -c_{m_{LF}}/c_{e} = -\left|c_{m_{C/q}} - \frac{1}{4}c_{e}\right|^{\frac{1}{2}}$  $\frac{x_{cp}}{c} = \frac{1}{4} - \frac{C_{mc/4}}{C_{l}}$ or, C runs to A %p/c La ` as C, -> 0 X С, Cmcla 00 0.25 -0.040 0.410 0.40 4° 0.64 -0.036 0.306 0.35 8° 1.08 -0.036 0.283 0.30 120 1,43 0.271 -0,030 0.25 X° 0 4 12

Fall 06 Problem 4 UE Fluids Units Parameter m 1/12  $\mathcal{D}$ N-K=8-3= 5 Pi groups expected. X m/13  $TT_{r} = \frac{D}{\frac{1}{2P} V^{2} b c} \equiv C_{b}$ 46 17 L/t Ø  $\widehat{T}_{\lambda} = \mathcal{K}$ m/l.t M 6 and the second se  $\widehat{\Pi}_3 = \frac{\rho V_c}{\mu} = Re Reynolds #$ l C 8 2,  $T_{A} = \frac{V}{\alpha} = M_{\infty} M_{\alpha}h #$  $TT_{=} = \frac{1}{C} = R$  aspect ratio Many other alternatives are possible, e.g.  $\overline{\Pi}_2 = \frac{\rho \dot{a} c}{\mu} = \frac{R_e}{M_e}$  weird, but OK or  $\overline{11}_3 = \frac{1}{N} = Re \cdot R$  even more weird TT\_= = = = A Unconventional etc

## Unified Engineering Problem Set Week 2 Fall, 2006

## **SOLUTIONS**

**M2.1** There are multiple design considerations for all structures and these inevitably involve tradeoffs. The important point here is to identify those considerations which are the most important (i.e. key) to the particular design and thus purpose of that structure. In this vein, there are no "right" answers here since the specific purposes of the structure were not defined. Thus, the first part to answering this question is to define, in your mind, what purpose(s) the structure serves.

It is also important to realize that true tradeoffs (i.e. *relative* importance) can only be done quantitatively when a clear objective is in mind and the factors associated with the different design factors can be quantified. Otherwise, only general statements about tradeoffs can be made. This should become clearer in the "answers" below.

- (a) <u>Space station</u>: The structures of a space station serve a number of important purposes. The habitat structure must protect the inhabitants and thus be resistant to meteorite impact. The station will be in operation for many years and thus must have longevity. In orbit, this means resistance to uv, atomic oxygen, fatigue, and other long-term considerations. In orbit, a structure goes in and out of earth's shadow and thus can undergo very large thermal gradients. Thus, dimensional stability is important. Weight and cost are always factors, but it is hard to evaluate these without knowing the scheme for getting the parts of the space station to orbit and also knowing how important the space station is to the country and thus its priority in the budget process.
- (b) <u>Commercial transport aircraft</u>: With safety as generally the number one consideration, a key item is strength -- ability to carry loads without failure. This also means that there are deformation resistance considerations at various parts of the aircraft. Clearly the wing must hold its shape to a certain margin, otherwise there will be a potential loss in the aerodynamic characteristics and thus the forces on the wing, known as lift and drag. The airplane is designed to last a number of years, so corrosion resistance is important as are considerations of fatigue and general durability. Weight is a clear consideration since this involves a tradeoff with additional payload/fuel/range. Finally the buyers of the aircraft must be able to afford the aircraft, so cost of the airplane is a key (including items such as operating cost), if not *the* key consideration.
- (c) <u>Glider</u>: A glider is also an airplane, but it looks quite different from a commercial transport. Glider wings have a very large "aspect ratio" (length of the wing/"width" of the wing). Due to this great length, glider wings must be very stiff, so deformation resistance is a key consideration with strength being an important design consideration but not as great a consideration as the stiffness/rigidity. Gliders are unpowered and rely on a tow to get to altitude and then on "thermals" to soar higher (Ever watch a hawk or eagle? They do the same thing!). Thus, weight is a

much more critical factor with regard to gliders. How about cost? Clearly, cost is a consideration in any consumer product. However, in the sports industry, many people with the "means" are willing to pay extra for extra performance. so cost becomes quite a different factor here since performance becomes a much greater consideration for which people are willing to pay. Thus, cost/performance becomes a key tradeoff.

- (d) <u>Automobile</u>: The main considerations in an automobile are safety and cost with cost probably being the most important. This is truly a consumer product and the general consumer is much less likely to care about technical innovations and performance capability. They want a product that will work, will get them there, won't break down, and will last. Thus, the next key design consideration is longevity. With care, especially car bodies, this is inherently linked to corrosion resistance. Another key design consideration is energy absorption in impacts (also known as "crashes"). Much of the body and structure of a car is designed to absorb the energy in such an incident in order to protect the occupant(s).
- (e) <u>Bridge</u>: A bridge is a pretty basic structure and the design considerations are also pretty straightforward. The structure must be strong enough to carry the loads without failure. However, the rigidity of the bridge can be an even more important consideration since deflection/deformation must be resisted (No one wants a "floppy" bridge). Bridges are exposed to the elements, so corrosion resistance is also an important consideration. And this is linked to the final item which is longevity. Bridges are made and used for decades and thus must not fatigue, corrode, etc. This ends up being a tradeoff with the other major consideration -- cost. It is a question of the up-front cost of the structure versus the cost "down the road" to maintain and repair. This should include consideration of the inconvenience caused to commuters, etc. when bridges are closed or traffic restricted when major repairs are made (You should have been around when they did this to the Harvard Bridge a few years ago!).
- (f) <u>Step ladder</u>: As a consumer product, the most important consideration is cost. However, safety must be right behind this, so this brings in considerations of strength, rigidity, and longevity. Longevity is less important here since it is relatively easy enough to replace this product and one can tell when the material starts to corrode, degrade, etc. Stiffness and strength, however, are clearly important. There are also other considerations. One of the main ones is electrical conductivity. Wood ladders were used for many years and then metal ones were introduced. The problem with metal (generally aluminum) ladders is that if they touch a wire, the person touching the ladder can be electrocuted (or at least shocked). Plastic/glassreinforced ladders have been introduced because of this. They keep down the weight, which is important because people have to be able to carry these things, but they are not conductive and provide that extra electrical safety.



Unified Engineering

Homework - SOLUTIONS week#2

M2.2 (a) Revolve each force into x and my components with the use of i and j't vectors in order to simplify plotting and tor describing eachas a vector



\*NoTT: At times unit vectors are noted via "hats" (^) rather than an under line (or overser) ion a general vetor. PAL

$$\begin{split} & F_{1}(-2,-1) = (4N) \{ \hat{i} \cos(73.5^{\circ}) + \hat{j} \sin(73.5^{\circ}) \} \\ &= (1.14N) \hat{i} + (3.84N) \hat{j} \\ & F_{2}(5,-5) = (5N) \{ \hat{i} \cos(135^{\circ}) + \hat{j} \sin(135^{\circ}) \} \\ &= (-354N) \hat{i} + (3.54N) \hat{j} \\ & F_{3}(-3,-4) = (2N) \{ \hat{i} \cos(-315^{\circ}) + \hat{j} \sin(-315^{\circ}) \} \\ &= (1.42N) \hat{i} + (1.42N) \hat{j} \\ & F_{4}(1,1) = (4N) \{ \hat{i} \cos(-106.5) + \hat{j} \sin(-106.5) \} \\ &= (-1.14N) \hat{i} + (-3.84N) \hat{j} \\ & F_{5}(3,2) = (3N) \{ \hat{i} \cos(-25)^{\circ} + \hat{j} \sin(-25^{\circ}) \} \\ &= (3.0N) \hat{i} \\ & F_{5}(-4,3) = (3N) \{ \hat{i} \cos(-25)^{\circ} + \hat{j} \sin(-25^{\circ}) \} \\ &= (2.72N) \hat{i} + (-1.27N) \hat{j} \\ & So the vector derchiption is fiven by the anguitation of the fores in each Greechim times the unit vectors, with indication of the (X, Y) \\ & Iocation than which the fores we to cate. Greechim times the unit vectors, with indication of the (X, Y) \\ & Iocation than which the fores (3.54N) \hat{j} \\ & F_{3}(-3,-4) = (1.42N) \hat{i} + (1.42N) \hat{j} \\ & F_{3}(-3,-4) = (3.0N) \hat{i} \\ & F_{5}(3,2) = (3.0N) \hat{i} \\ & F_{5}(-4,3) = (2.72N) \hat{i} - (1.27N) \hat{j} \\ \end{split}$$

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(b)  $E_{total} = \sum F_i$   $e_{chist} = t_{total} = \sum F_i$   $e_{cm} ponents to sum <math>\hat{i}$  and  $\hat{j}$  component (i.e. form com ponents tin x and y directions)  $\Rightarrow F_{total} = \hat{i} (1.14N+3.54N+1.42N-1.14N+3.0N+2.72N)$   $+\hat{j} (3.84N+3.54N+1.42N-3.84N-1.27N)$  $= \hat{F}_{total} = (3.6N)\hat{i} + (3.69N)\hat{j}$ 

the overall magnitude is  

$$|F| = \sqrt{(F_x)^2 + (F_y)^2}$$
  
 $\Rightarrow |F_{\text{total}} = \sqrt{(3.6N)^2 + (3.64N)^2}$   
 $|F_{\text{total}} = 5.16N$ 

(c) The definition of a couple is it refults from two parallel (coplanad forces of equal magnitude and opposite direction such that there is a moment but no net force

Notice on the grid that F, and Eq are equal in magnitude and opposite in direction and thur ratis tythis detuition. For the answeris: [VES -- F, and fq]

Now use the expression for the couple:  

$$\underline{C} = \underline{r} \times \underline{F}$$

where: E is one of the vectors r is the position vector from me to the other

to get 
$$\underline{r}$$
, subtract one  $(x, y)$  position from  
the other and use the unit vectors for each  
component: [from  $F_1$  to  $F_4$ ]  
 $\Rightarrow \underline{r} = (1 - (-2))\hat{i} + (1 - (-1))\hat{j}$   
 $= 3\hat{i} + 2\hat{j}$  units in [m]

Thus:  

$$\underline{C} = (3\hat{i} + 2\hat{j}) \times (-1.14\hat{i} = 3.64\hat{j}) [N.m]$$
Recall:  $\hat{i} \times \hat{j} = \hat{k} (\frac{1}{2} - direction)$   
 $\hat{j} \times \hat{i} = -\hat{k}$   
 $\hat{i} \times \hat{i} = 0$   
 $\hat{j} \times \hat{j} \times \hat{j} = 0$   
 $\hat{j} \times \hat{j} \times \hat{j} = 0$   
 $\hat{j} \times \hat{j} \times$ 

(d) Determine the moment about the origin of each force (E;) >> Mo; and sum there up to determine the vet moment.

$$\begin{split} & M_{01} = (-2\hat{i} - \hat{j})[m] \times (1.14\hat{i} + 3.84\hat{j})[N] \\ & = (-7.68\hat{k} + (-1.14)(\cdot\hat{k}))[N\cdotm]^{\pm} - 6.54Nm\hat{k} \\ & M_{02} = (5\hat{i} - 5\hat{j})[m] \times (-3.54\hat{i} + 3.54\hat{j})[N] \\ & = (-17.7\hat{k} + 17.7(-\hat{k}))[M] = Q \\ & M_{03} = (-3\hat{i} - 4\hat{j})[m] \times (1.4\hat{i} + 1.4\hat{j})[N] \\ & = (-4.26\hat{k} + (-5.68)(-\hat{k}))[V\cdotm]^{\pm} 1.4\hat{2}N\cdotm\hat{k} \\ & M_{04} = (\hat{i} + \hat{j})[m] \times (-1.14\hat{i} - 3.84\hat{j})[N] \\ & = (-3.84\hat{k} + (-1.14)(-\hat{k}))[N\cdotm]^{\pm} - 2.70N\cdotm\hat{k} \\ & M_{05} = (3\hat{i} + 2\hat{j})[m] \times (3.0\hat{i})[N] \\ & = -6N\cdotm\hat{k} \\ & M_{06} = (-4\hat{i} + 3\hat{j})[m] \times (2.72\hat{i} - 1.27\hat{j})[N] \\ & = (5.0\hat{k}\hat{k} + 8.16(-\hat{k}))[N\cdotm]^{\pm} - 3.0\hat{k}N\hat{k} \end{split}$$

(e) This is the same set of calculations as in part (d), exapt the position vector in each case, I, is from (-Sm, -Sm) to the vector (rather than the origin). Thus:

$$\begin{split} \underline{r}(i = (x_i - (-5m))\hat{i} + (y_i - (-5m))\hat{j} \\ \text{and Grain:} \\ \underline{M}_i = \underline{r}(x \in \underline{r}) \\ \underline{M}_i = (3\hat{i} + 4\hat{j})\underline{lm} \times (1.14\hat{i} + 3.84\hat{j})\underline{lN}] \\ = (1.52\hat{k} + 4.56(-\hat{k}))\underline{lN}\underline{m}] = 6.96N\underline{m}\hat{k} \\ \underline{M}_2 = (10\hat{i})\underline{lm} \times (-3.54\hat{i} + 3.54\hat{j})\underline{lN}] = 35.4N\underline{m}\hat{k} \\ \underline{M}_3 = (2\hat{i} + \hat{j})\underline{lm} \times (-3.54\hat{i} + 3.54\hat{j})\underline{lN}] \\ = (2.84\hat{k} + 1.42(-\hat{k}))\underline{lN}\underline{m}] = 1.42N\underline{m}\hat{k} \\ \underline{M}_4 = (6\hat{i} + 6\hat{j})\underline{lm}] \times (-1.14\hat{i} - 3.54\hat{j})\underline{lN}] \\ = (-33.04\hat{k} + (-6.84)(-\hat{k}))\underline{lN}\underline{m}] = -16.2N\underline{m}\hat{k} \\ \underline{M}_5 = (8\hat{i} + 7\hat{j})\underline{lm}] \times (3.0\hat{i})\underline{lN}\underline{m} = -21.0N\underline{m}\hat{k} \\ \underline{M}_6 = (\hat{i} + 8\hat{j})\underline{lm}] \times (2.72\hat{i} - 1.27\hat{j})\underline{lN}] \\ = (-1.27\hat{k} + 21.76(-\hat{k}))\underline{lN}\underline{m}] = 23433N\underline{m}\hat{k} \end{split}$$

So with: 
$$M_{net} = \Sigma Mi$$
  

$$\implies M_{net} = 29.61 N \cdot m k$$

$$Out(-5m,-5m)$$

PAL

(f) By examination, one can see there are no components acting about the y-axis or the x-axis since the only unit vector in the expression for the momente is  $\hat{k}$  (+-direction)

Finilorly:  

$$My = Moment component about y-axis$$
  
 $= \hat{j} \cdot \underline{M}_{net}$   
 $\min \hat{j} \cdot \hat{k} = 0$   
 $= \sum My = 0$ 



(a) This system is not in equilibrium. One canse that both its net force and net manent are non-zero Call the ION force F, and the 6N force Fz Express each as vectors: F = 10N is at (1m, -1m)  $F_2 = -6N\hat{c}_2$  at (-1m, 1m)

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Thus, the net force, FN, is:  $\overline{F}_{N} = \overline{F}_{1} + \overline{F}_{2} = (\mathbf{O} \mathcal{N})^{2}_{3} = 6 \mathcal{N}^{2}_{2}$ The acceleration of the sifilly connected system is: a = <u>Enet</u> where m is the total mais = (4 × 3/cg)+(4×1/cg)+2/cy = 18kf  $\Delta = \frac{10N}{18k_{\rm F}} \hat{c}_3 - \frac{6N}{18k_{\rm F}} \hat{k}_{\rm F}$ 50: (NOTF: worth unite arky: 103) and N: Ict.m 50: a = 5 m/z i3 - 3 m/sz iz So the system has a linear acceleration of 5/g m/s 2 in the + x3 direction and '3 m/s2 in the - X2 direction There will also be a counterclo chrise rotation about the onfin This is due to the net moment: Mnet = E(Fix Ei) where: V:= Position vector to force vector Kon on fin

one can tind:  $\frac{M}{-net} = (1m)\hat{i}_{2} \times (10N)\hat{i}_{3} + (1m)\hat{i}_{3} \times (-6N)\hat{i}_{2}$ Also noting: iz x îz = i,  $\hat{i}_3 \times \hat{c}_2 = \hat{c}_1$ ⇒ Mnet = 10 N·m î, + (-6N·m) (-î) = 16 N·m ĉ, (6) By apply my other torcer and manent me can achieve equilibrium if the net fine and moment are zero. Thus, by applying aforce, one can get the net force to be zero. The applied force must be equal in magnifide and opposite indirection to the current net force (as existed  $\Rightarrow apply E = -10N\hat{c}_3 + 6N\hat{c}_2$ and the linear acceleration will be zero. BUT, by opplying a torce at the origin one cannot couse a moment about the origin (since r = 0), so the net mannent cannot Dewnezero. equilisium  $\Rightarrow (N \circ |$ Cannof the achi ered

(c) we have the opposite in this care. By applying a moment as out the origin, me can get the net miment to be zero (apply one équal in magnitude and opposité in direction => Mapplied = -16 N·mi, However, one cannot get the net force to be zero and thus there will be linear acceleration and motion equilibrium cannot be  $\Rightarrow NO$ Achieved

(d) Go back to the rear uningr of parts (b) and (c). Add these two (superposition applied) and equilibrium can be achieved  $= \sum [YFS]$ Apply:  $F = -10N\hat{i}_3 + 6N\hat{i}_2$  $M = -16N \cdot m\hat{i}_1$ 

(e) The same reasoning applies as in the answer to part (c). A couple results in a net moment but no net force. No a couple can cause the rotation to not occur, but the total net force will still be nonzero and thus there will be linear acceleration and motion. = NO equilibrium cannot be curved part Page 15 of 15

(f) A time pan achieve a desired in ment. by paper placement. Thus, wing as mila rearing as (d) a time properly placed fires the right opposite manent a southe origin and the time must be ar in (b) to achieve a net tora of zero.  $fo^{\prime}$  F = -10N  $\hat{c}_3 + 6N \hat{c}_2$ one must have  $\underline{M} = \underline{r} \times \underline{F} = -16 N \cdot m \hat{i},$  $\Rightarrow (x_2 \hat{i}_3 + x_2 \hat{i}_2) \times (-10N\hat{i}_3 + 6N\hat{i}_2) = -16N \cdot m\hat{i}_1$ where x3 and x2 identity the location of the force vector using:  $\hat{i}_z \times \hat{i}_3 = \hat{i}_1$ and  $\hat{i}_3 \times \hat{i}_2 = -\hat{i}_1$  $\Rightarrow -6x_3 N \hat{i}_1 - 10 x_2 N \hat{i}_1 = -16 N \cdot m \hat{i}_1$  $\Rightarrow -6x_3 - 10x_2 = 16m$ so this dishes a line along which /  $F = -10 \mathcal{N} \hat{\epsilon}_3 + 6 \mathcal{N} \hat{\epsilon}_2$ can suplaced to get egni lis hunned