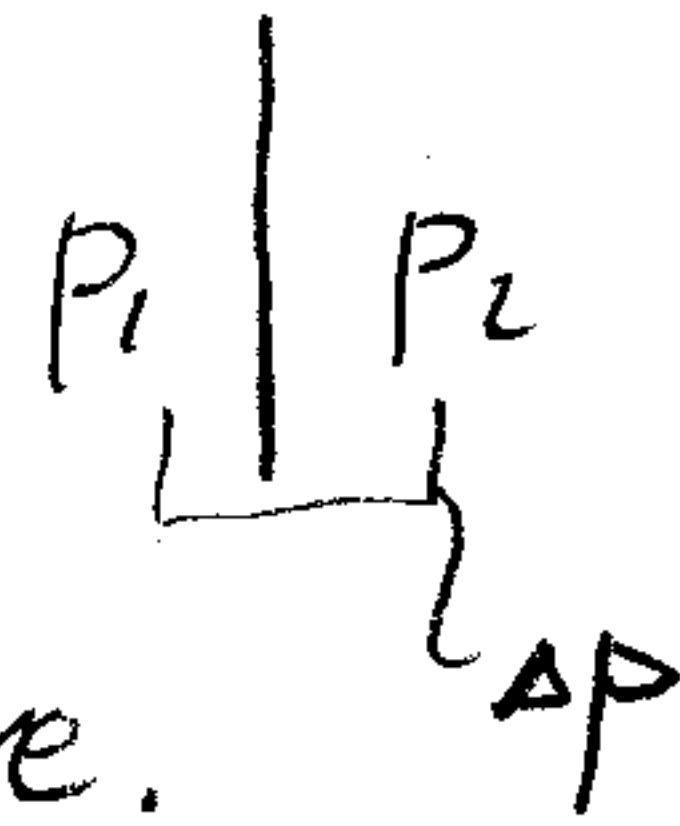


$$a) \text{dB} = 20 \log_{10} \left(\frac{\Delta p}{20 \times 10^{-6} \text{Pa}} \right) = 120 \rightarrow \Delta p = 20 \text{ Pa}$$



This Δp is the pressure change across a shock (sound) wave.

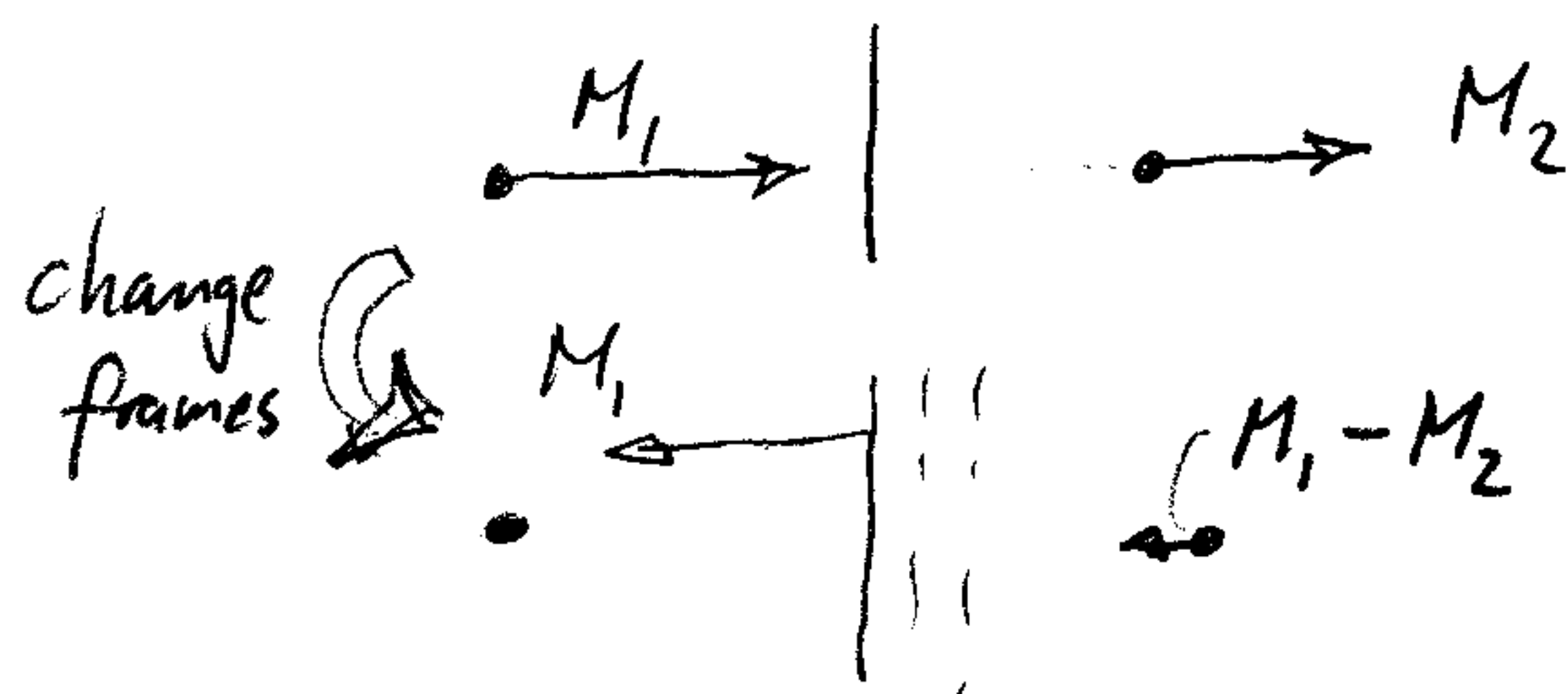
$$\Delta p = p_2 - p_1 \quad \text{where } p_1 \approx 10^5 \text{ Pa (atmosphere, sea level)}$$

$$\frac{p_2}{p_1} = \frac{p_1 + \Delta p}{p_1} = 1 + \frac{\Delta p}{p_1} = 1 + \frac{20}{100000} = 1 + \frac{2\gamma}{\gamma+1} (M_1^2 - 1)$$

$$\Rightarrow M_1^2 = 1.000171, \quad \boxed{M_1 = 1.000086} \quad \underline{\underline{\text{weak}}}$$

This M_1 is the Mach number of the sound wave propagating into still air.

$u_1 \approx a_1$ (weak shock)



$$b) \frac{T_2}{T_1} = \left[1 + \frac{2\gamma}{\gamma+1} (M_1^2 - 1) \right] \frac{2 + (\gamma-1)M_1^2}{(\gamma+1)M_1^2} = 1.000057$$

For $T_1 = 300 \text{ K}^\circ$, $T_2 = 300.017^\circ$

$$\boxed{\Delta T = 0.017 \text{ K}^\circ}$$

pretty wimpy.

a) Anderson p. 502, problem 15.

At 80000 ft = 15.15 mi = 24.38 km;

$$V_1 = V_\infty = 2112 \text{ mph} = 943.9 \text{ m/s}$$

$$M_1 = \frac{V_1}{a_1} = 3.17$$

$$\frac{T_2}{T_1} = \left[1 + \frac{2\gamma}{\gamma+1}(M_1^2 - 1) \right] \frac{2 + (\gamma-1)M_1^2}{(\gamma+1)M_1^2} = 2.885$$

$$T_2 = T_1 \cdot 2.885 = 638 \text{ K}^\circ = 1148 \text{ R}^\circ = 656 \text{ F}^\circ$$

Std Atmosphere

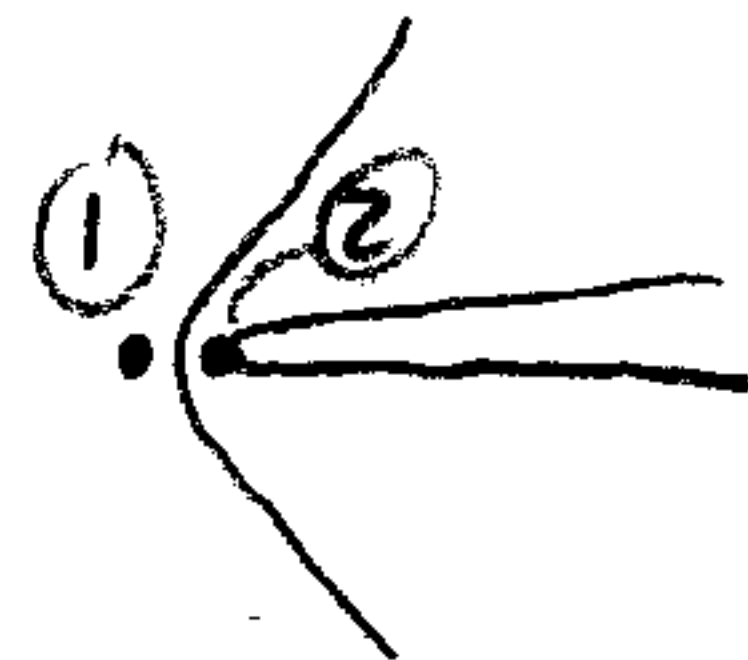
$$a_1 = 298 \text{ m/s}$$

$$\rho_1 = 0.0437 \text{ kg/m}^3$$

$$P_1 = 5430 \text{ Pa}$$

$$T_1 = 221 \text{ K}^\circ$$

b) P_{02} will be behind bow shock at tip



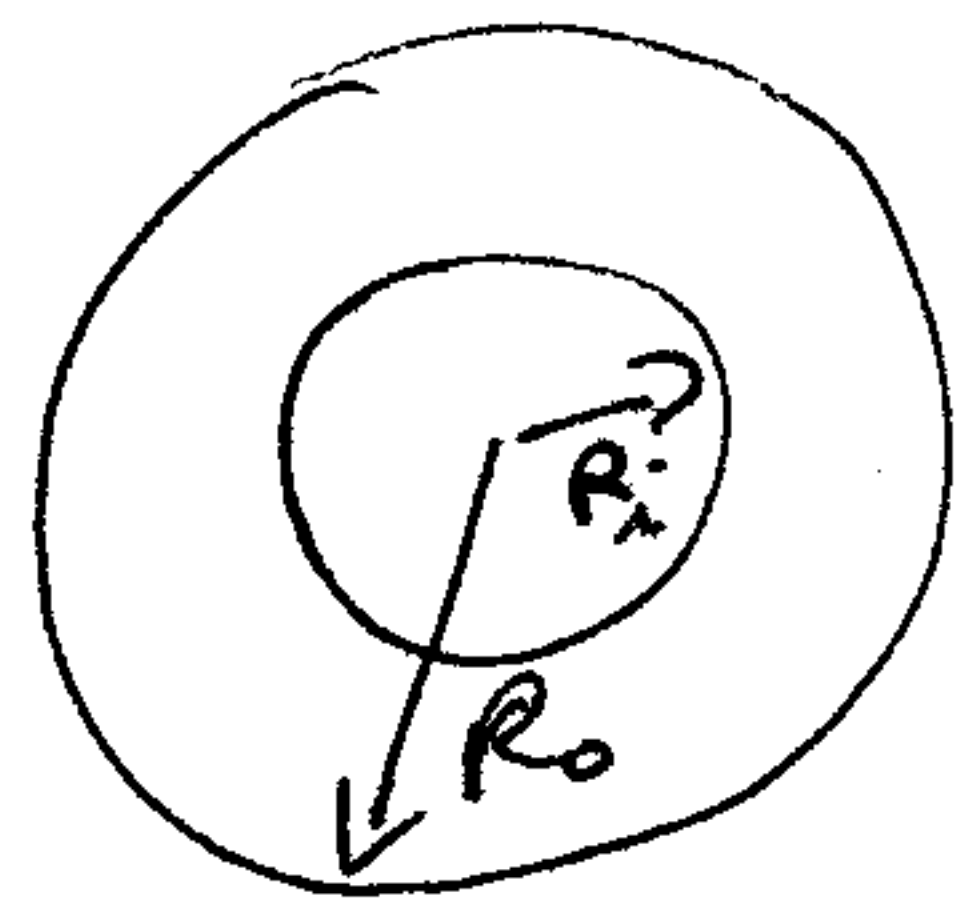
From Anderson Appendix B:

for $M_1 = 3.17 \rightarrow \frac{P_{02}}{P_1} = 13.4$

$$P_{02} = P_1 \cdot 13.4 = 7.28 \times 10^4 \text{ Pa}$$

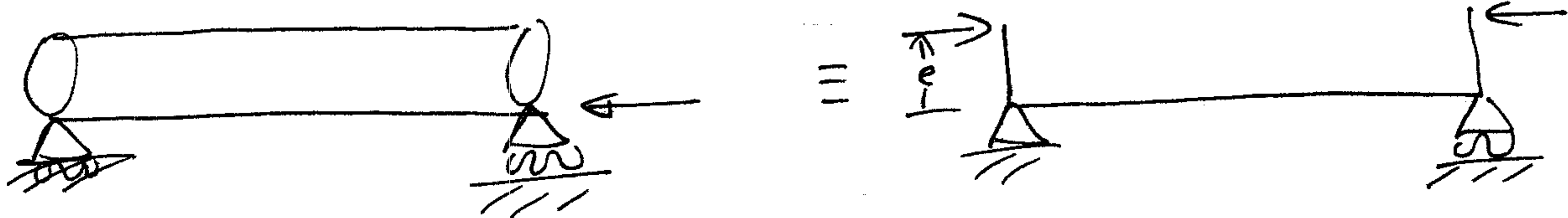
M14

$$E = 70 \text{ GPa}$$



$$R_i = \frac{4}{5} R_o$$

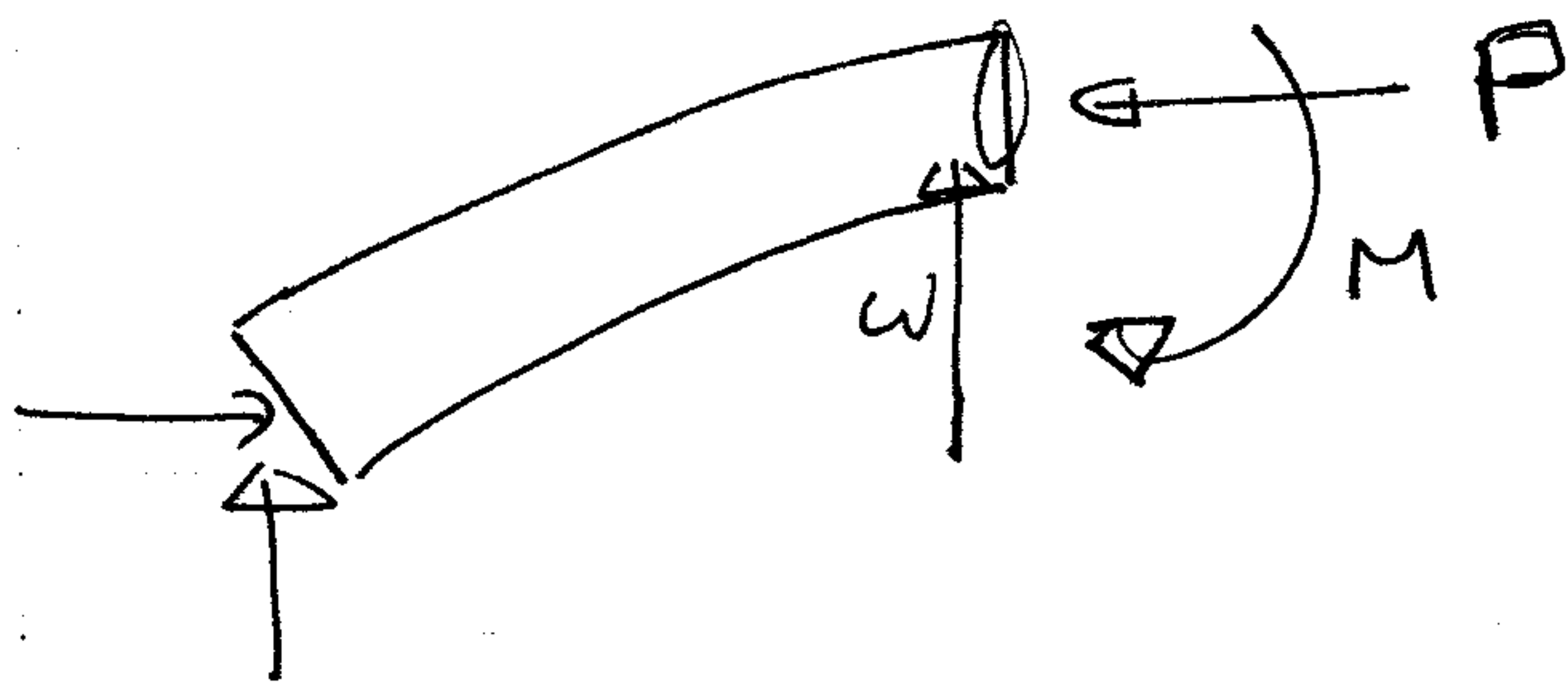
Basic Problem:



which we met in class

$$\therefore w = e \left(\frac{1 - \cos \sqrt{\frac{P}{EI}} L}{\sin \sqrt{\frac{P}{EI}} L} \sin \sqrt{\frac{P}{EI}} x + \cos \sqrt{\frac{P}{EI}} x - 1 \right)$$

The stress at a given point



$$\sigma = \frac{P}{A} \pm \frac{Mz}{I}$$

axial bending

and $M = EI \frac{d^2 w}{dx^2}$

$$\frac{d^2 w}{dx^2} = -e \left(\frac{P}{EI} \right) \left[\frac{1 - \cos \sqrt{\frac{P}{EI}} L}{\sin \sqrt{\frac{P}{EI}} L} \sin \sqrt{\frac{P}{EI}} x + \cos \sqrt{\frac{P}{EI}} x - 1 \right]$$

Moment will be a maximum at $x = \frac{L}{2}$ since $w = \text{max}$.

$$M = EI \frac{d^2 w}{dx^2}$$

$$M_{\max} = -ep \left[\frac{1 - \cos \sqrt{\frac{P}{EI}} L}{\sin \sqrt{\frac{P}{EI}} L} \sin \left(\sqrt{\frac{P}{EI}} \frac{L}{2} \right) + \cos \left(\sqrt{\frac{P}{EI}} \frac{L}{2} \right) \right]$$

$$\text{let } \sqrt{\frac{P}{EI}} \frac{L}{2} = \theta$$

$$\sin 2\theta = 2 \sin \theta \cos \theta, \quad \cos 2\theta = \cos^2 \theta - \sin^2 \theta$$

$$M_{\max} = -ep \left[\frac{1 - (\cos^2 \theta - \sin^2 \theta) \sin \theta}{2 \sin \theta \cos \theta} + \cos \theta \right]$$

$$1 = \cos^2 \theta + \sin^2 \theta$$

$$M_{\max} = -Pe \left[\frac{\cos^2 \theta + \sin^2 \theta - \cos^2 \theta + \sin^2 \theta}{2 \cos \theta} + \cos \theta \right]$$

$$= -Pe \left[\frac{\sin^2 \theta}{\cos \theta} + \cos \theta \right]$$

$$= -Pe \left[\frac{\sin^2 \theta + \cos^2 \theta}{\cos \theta} \right] = -Pe \left[\frac{1}{\cos \theta} \right] !$$

$$\therefore M_{\max} = -Pe \left[\sec \sqrt{\frac{P}{EI}} \frac{L}{2} \right] \Leftarrow$$

$$\text{max stress} = \frac{P}{A} + \frac{Mz}{I} \quad \text{max for tensile stress}$$

$$A = \pi (R_o^2 - R_i^2) = \pi R_o^2 (1 - \alpha^2)$$

$$\text{let } \frac{r}{R_o} = \alpha \\ R_i = \alpha R_o$$

$$I = \frac{\pi}{4} (R_o^4 - R_i^4) = \frac{\pi R_o^4}{4} (1 - \alpha^4)$$

$$z = R_o, \quad e = R_o$$

$$\therefore \sigma = \frac{P}{\pi R_0^2 (1-\alpha^2)} + \frac{PR_0}{\pi R_0^4 (1-\alpha^4)} \left[\sec \sqrt{\frac{P}{EI}} \frac{L}{2} \right]$$

$$\sigma = \frac{P}{\pi R_0^2} \left[\frac{1}{(1-\alpha^2)} + \frac{1}{R_0 (1-\alpha^4)} \sec \sqrt{\frac{P}{EI}} \frac{L}{2} \right]$$

Need to iterate to solve. Calculate P_{crit} first = 20kN.

$$I = \frac{\pi \times (25 \times 10^{-3})^4}{4} \left(1 - \left(\frac{4}{5}\right)^4 \right) = 181.1 \times 10^{-9}$$

$$EI = 12.7 \times 10^3$$

$$\frac{L}{2} = 1.25 \text{ m}$$

$$\frac{1}{25 \times 10^{-2} \left(1 - \left(\frac{4}{5}\right)^4 \right)} = 67.8$$

$$P = 10 \text{ kN. } \sigma = 792 \times 10^6 \text{ Pa ! too high } \frac{1}{1-\alpha^2} = \frac{25}{9}$$

$$P = 1 \text{ kN } \sigma = 38.2 \text{ MPa}$$

$$P = 3 \text{ kN } \sigma = 131 \text{ MPa}$$

$$P = 2 \text{ kN } \sigma = 81.5 \text{ MPa}$$

$$\pi R_0^2 = 1.96 \times 10^{-3}$$

$$\frac{1}{\sqrt{EI}} \frac{L}{2} = 11.10 \times 10^{-3}$$

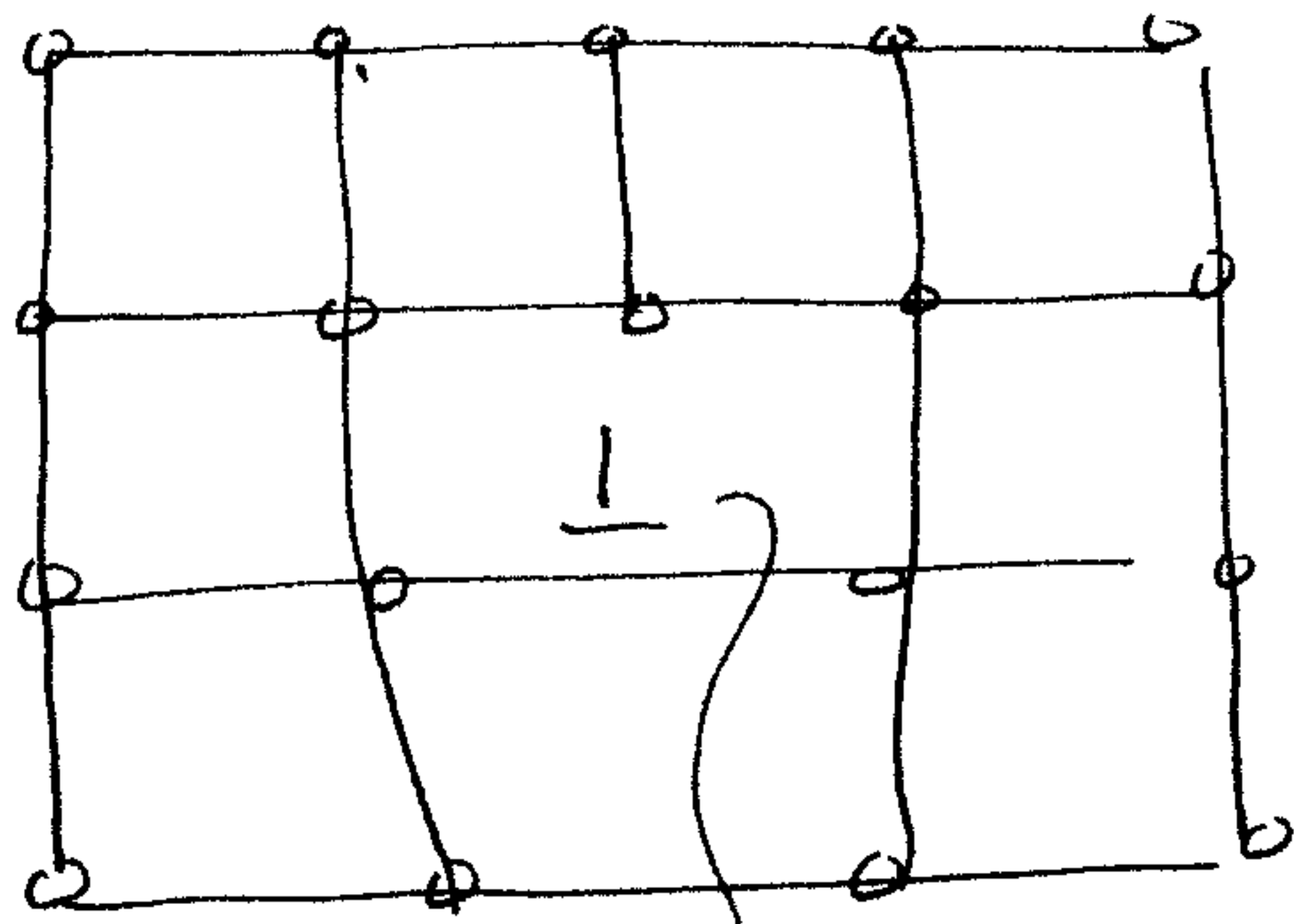
\therefore max load $\approx 2 \text{ kN}$

$$\text{For perfect column } P_{crit} = \frac{\pi^2 EI}{L^2}$$

$$= \frac{\pi^2 \times 12.7 \times 10^3}{2.5^2} = 20 \times 10^3 \text{ N}$$

M15

- a) A dislocation is a defect in a crystal lattice consisting of an extra half-plane of atoms (edge dislocation).



dislocation core

Application of a shear stress allows the dislocation to move by breaking one row of atoms at a time.

- b) Rolling allows the billet of metal to be reduced in thickness. The hot rolling allows large reductions in thickness by allowing creep and diffusion processes to occur. The final step of cold rolling allows work hardening to occur which increases the strength of the resulting material.

M15

- c) Polycrystalline material contains grain boundaries which increase the resistance to dislocation motion. There are no such boundaries in a single crystal.
- d) The toughness of engineering alloys to a large extent reflects the contribution of plasticity to energy absorption at the crack tip. Lower yield stress materials tend to have higher toughnesses as they have more plastically deforming material at the crack tip.
- e) Carbon and glass are brittle materials. Their strength is determined by the size of flaws (cracks) present. By drawing the fibers down to a small diameter the maximum flaw size is limited and a high strength results.

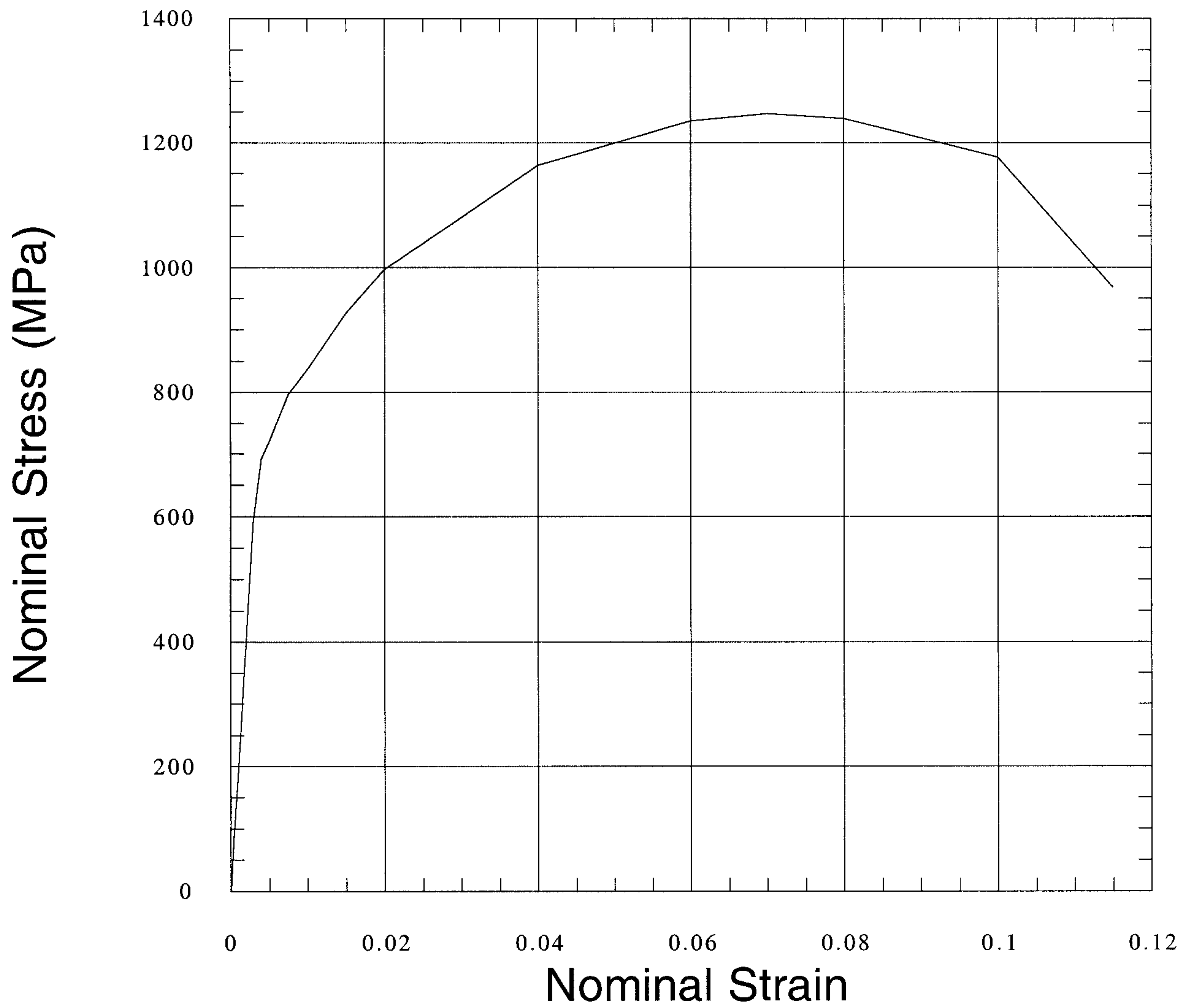
M15

f) Duralumin is an Al(Cu) alloy which is hardened by CuAl_2 precipitates. The extent to hardening is proportional to $\propto \frac{1}{L}$ the spacing of the precipitate particles.

The time dependence of the hardness reflects the growth of the particles from the solid solution. At short times the particles are too small to be effective at pinning dislocations. At very long times the particles have grown so large that "L" is also large. Thus there is a maximum at intermediate times when the particles are large enough to be effective and are still closely spaced.

M16 a)

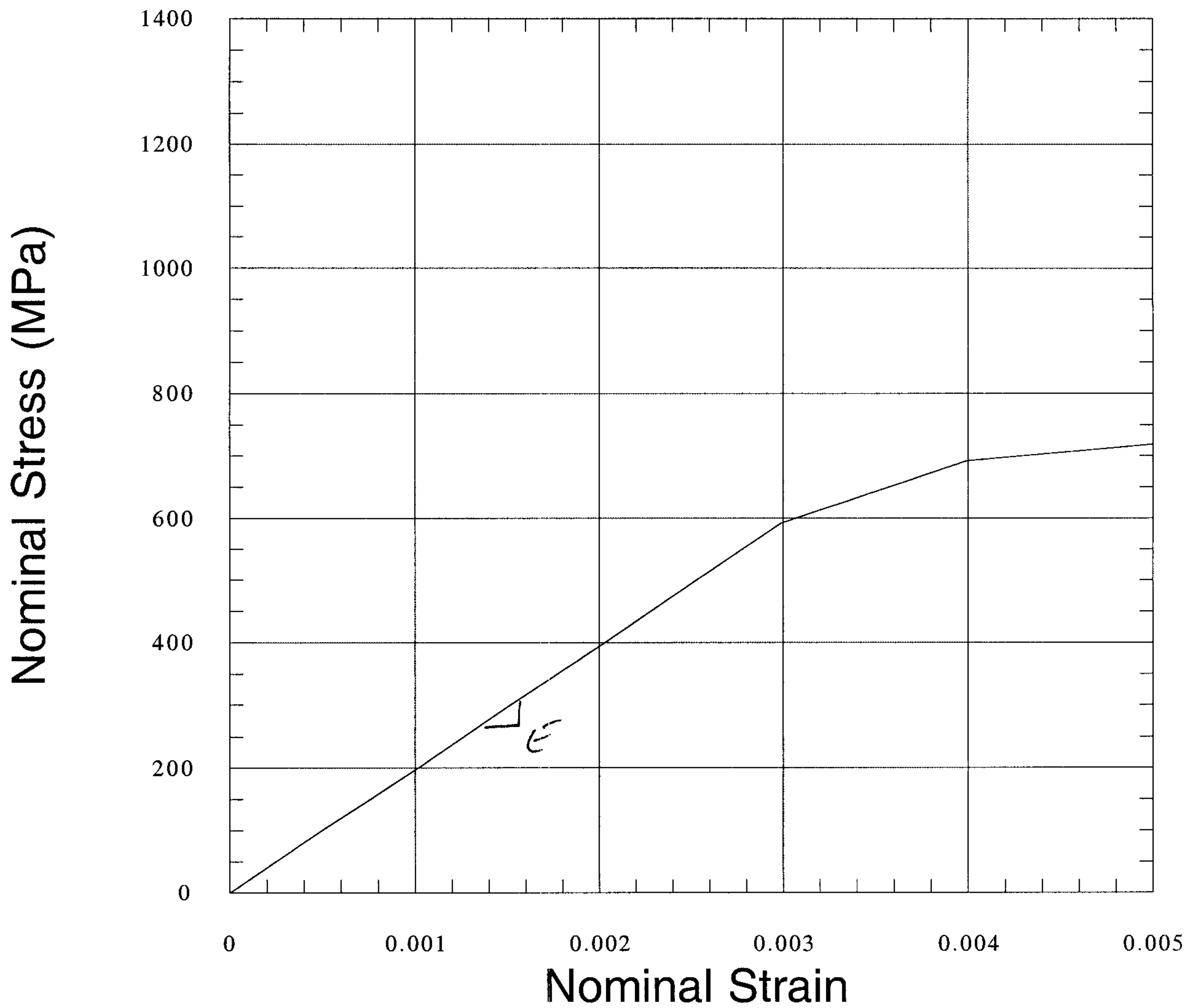
M16 Data



M16 b)

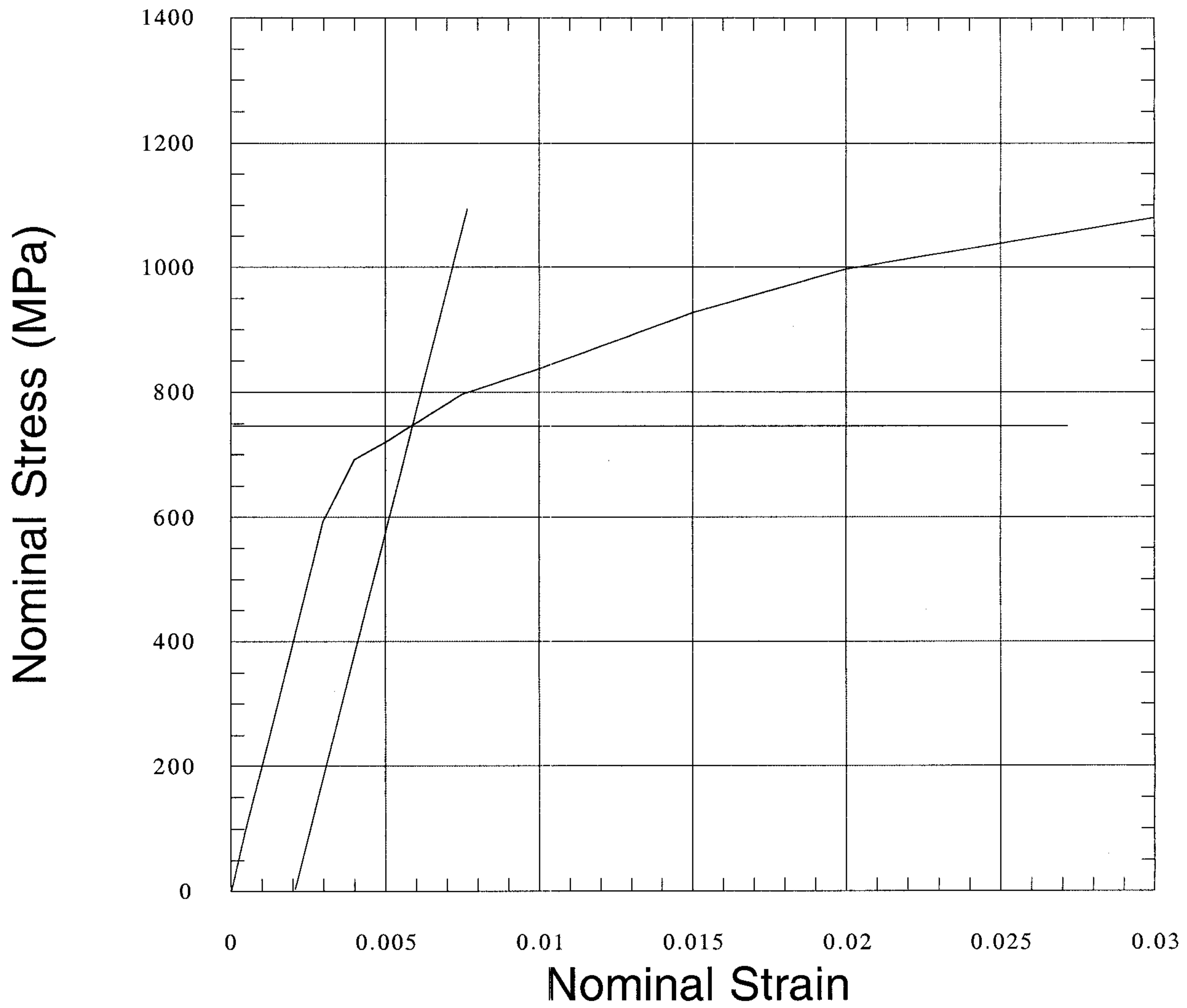
$$E = \frac{\sigma}{\epsilon} = \frac{400 \times 10^6}{0.002} = 200 \text{ GPa. } \llcorner$$

M16 Data

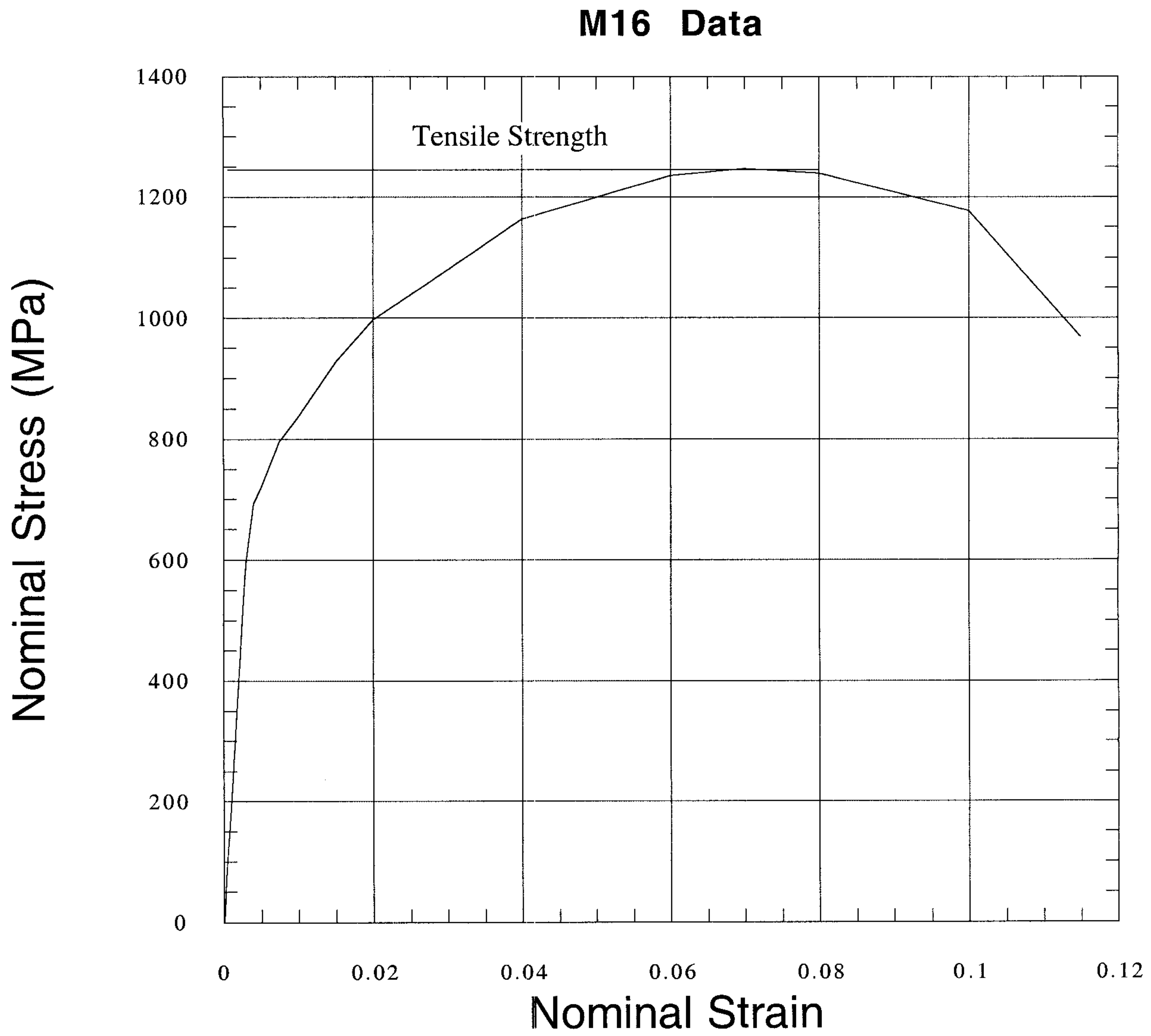


M16 c) 0.2% offset $\sigma_y = 750 \text{ MPa}$.

M16 Data

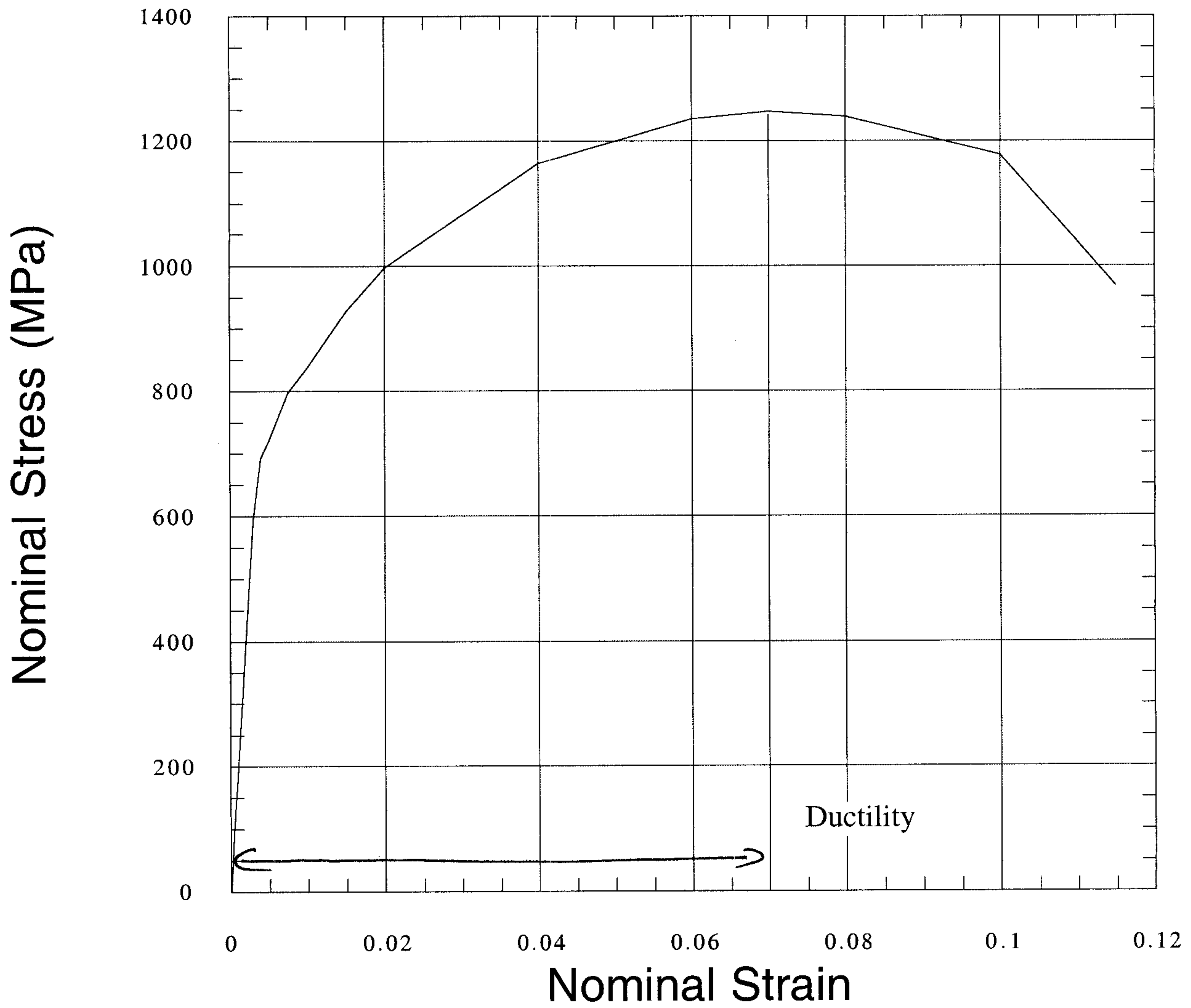


M16 d) Tensile strength = 1250 MPa. \Leftarrow



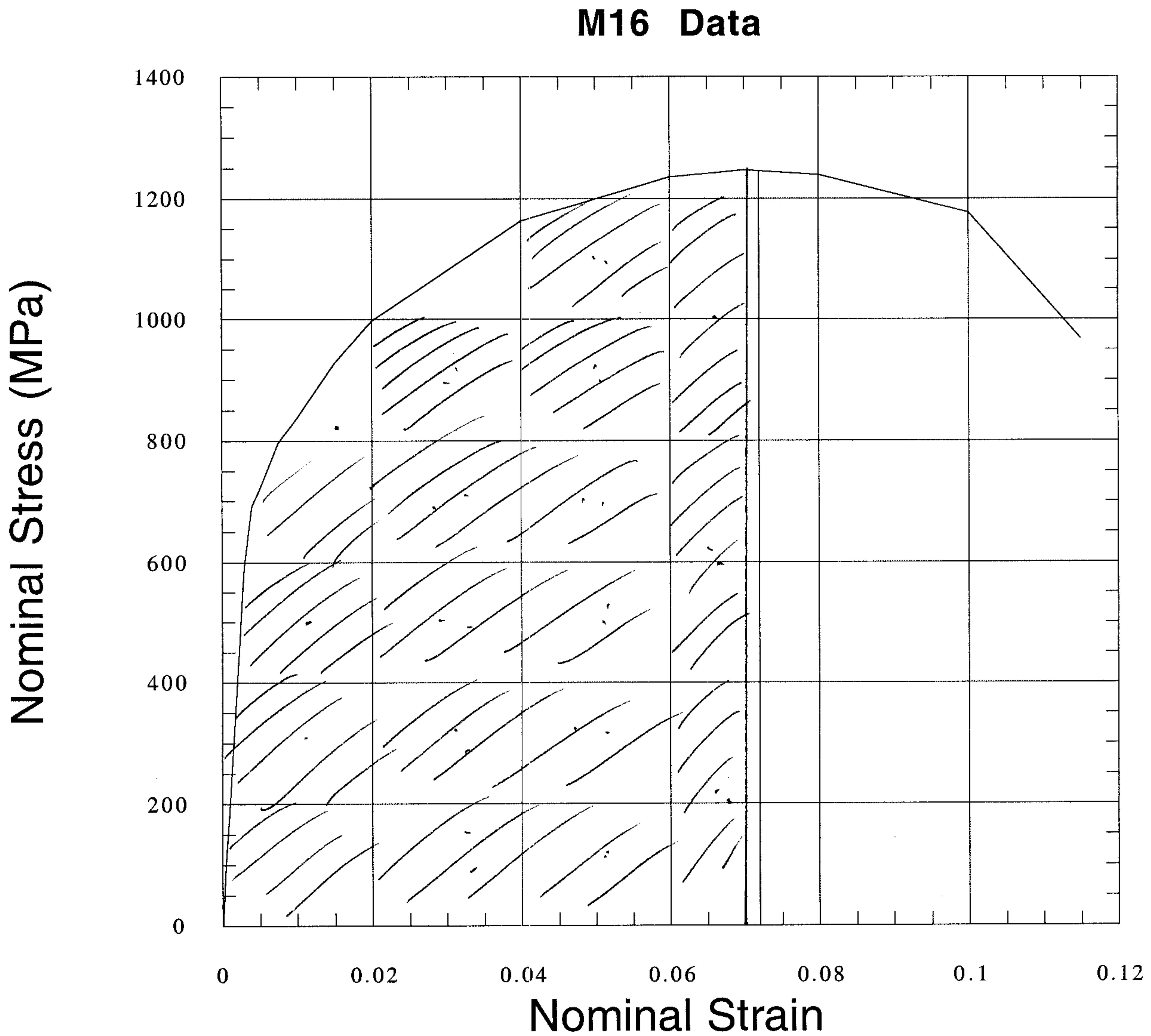
M16 e) Ductility = 0.07 = 7%

M16 Data



M16 f) $18\frac{1}{2}$ Squares $0.02 \times 200 \text{ MPa} = 4 \text{ MJ/m}^2$

$$\therefore \text{Tensile energy} = 4 \times 10^6 \times 50.8 \times 10^{-3} \times \pi \times (6.4 \times 10^{-2})^2$$
$$= 264 \text{ J. } \Leftarrow$$



Approx $18\frac{1}{2}$ $0.02 \times 200 \text{ MPa}$ squares

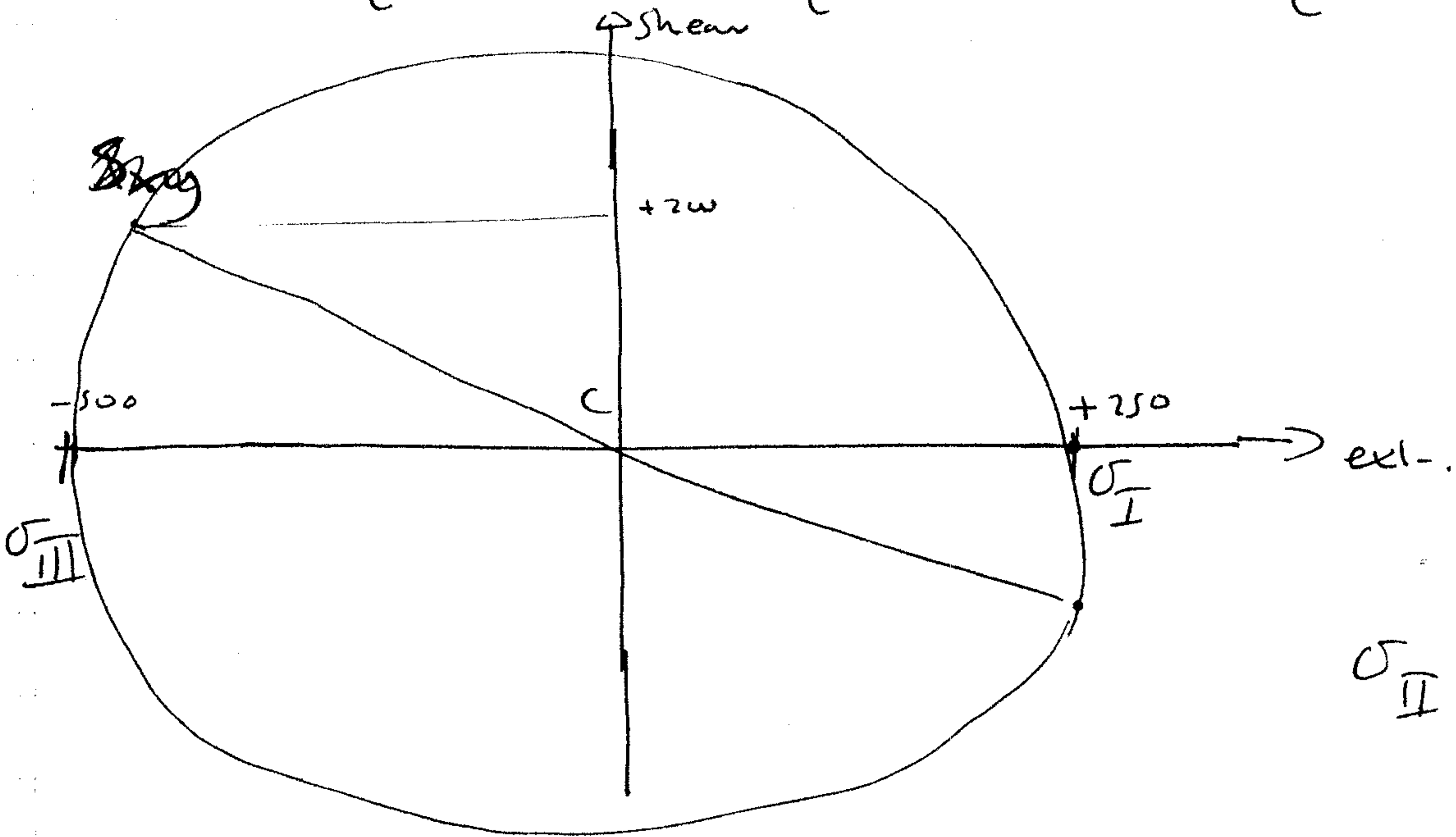
M17

$$\sigma_y = \frac{N_y}{F}$$

$$\sigma_x = \frac{N_x}{F}$$

$$\sigma_{xy} = \frac{S_{xy}}{F}$$

①



$\sigma_{II} = 0$
through the axis

$$\sigma_{I,II} \text{ c} = \frac{+250 - (-500)}{2} + (-500) = -125$$

$$R = \sqrt{(250 - (-125))^2 + (2w)^2} = 425.$$

$$\therefore \sigma_I = -125 + 425 = +\frac{300 \times 10^3 \text{ Pa}}{F} \quad \Leftarrow$$

$$\sigma_{III} = -125 - 425 = -\frac{550 \times 10^3 \text{ Pa}}{F} \quad \Leftarrow$$

Von Mises

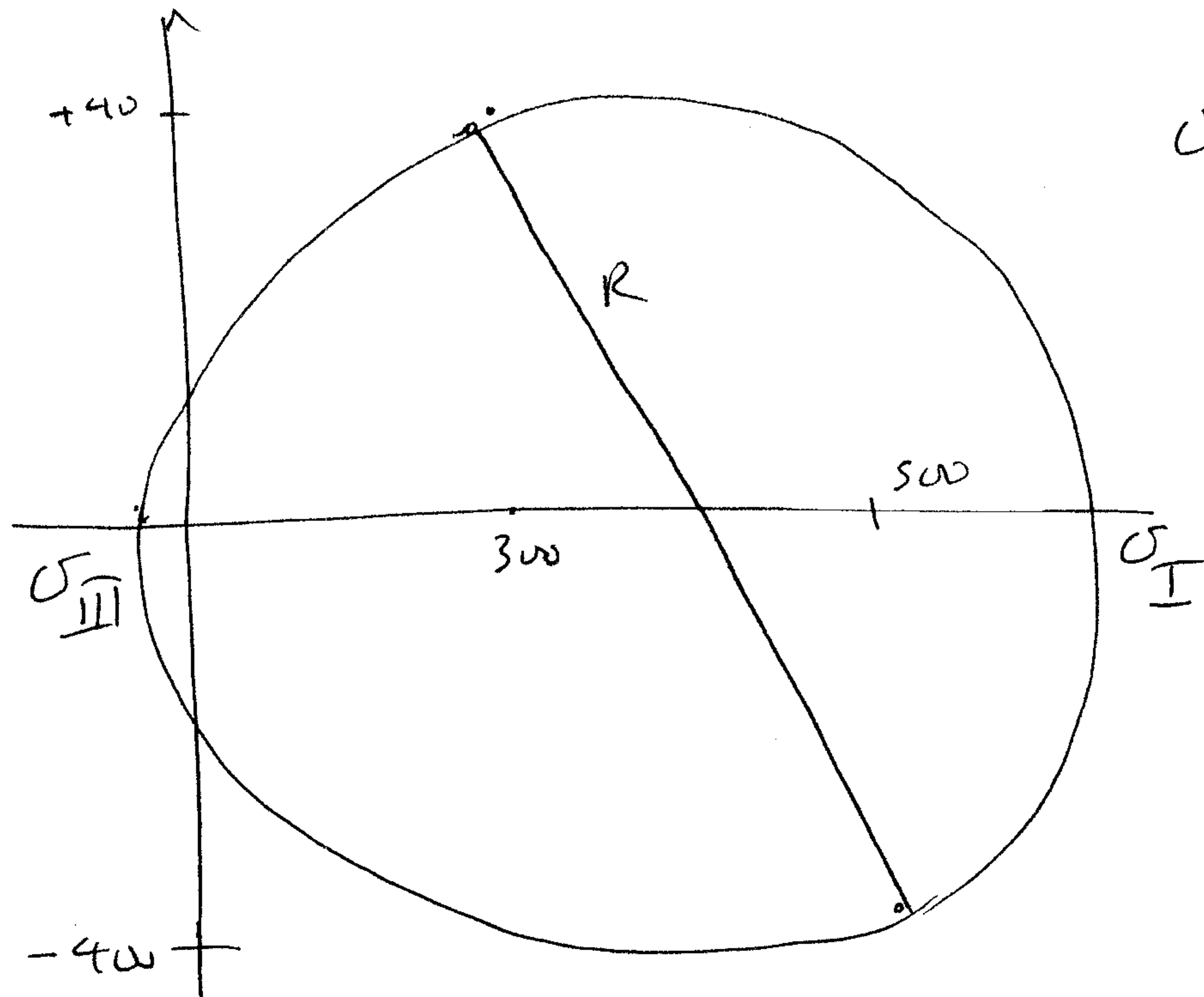
$$\left(\frac{300 \times 10^3}{F} - 0 \right)^2 + (0 + 550)^2 + (550 - \frac{300}{F})^2 \geq 2\sigma_y^2$$

$$(300 - 0)^2 + (0 + 550)^2 + (-550 - 300)^2 = \frac{2 \times (500 \times 10^6)^2}{(10^3)^2 \times 1.5^2}$$

safety factor

$$t = \sqrt{\frac{1115 \times 10^3 \times (10^3)^2 \times (1.5)^2}{2 \times (500 \times 10^6)^2}} = 2.24 \text{ mm} \Leftarrow$$

(2)



$$C = 400$$

$$R = \sqrt{(500 - 400)^2 + 400^2} = 412.3$$

$$\therefore \sigma_I = 400 + 412.3 = \frac{812.3}{t}$$

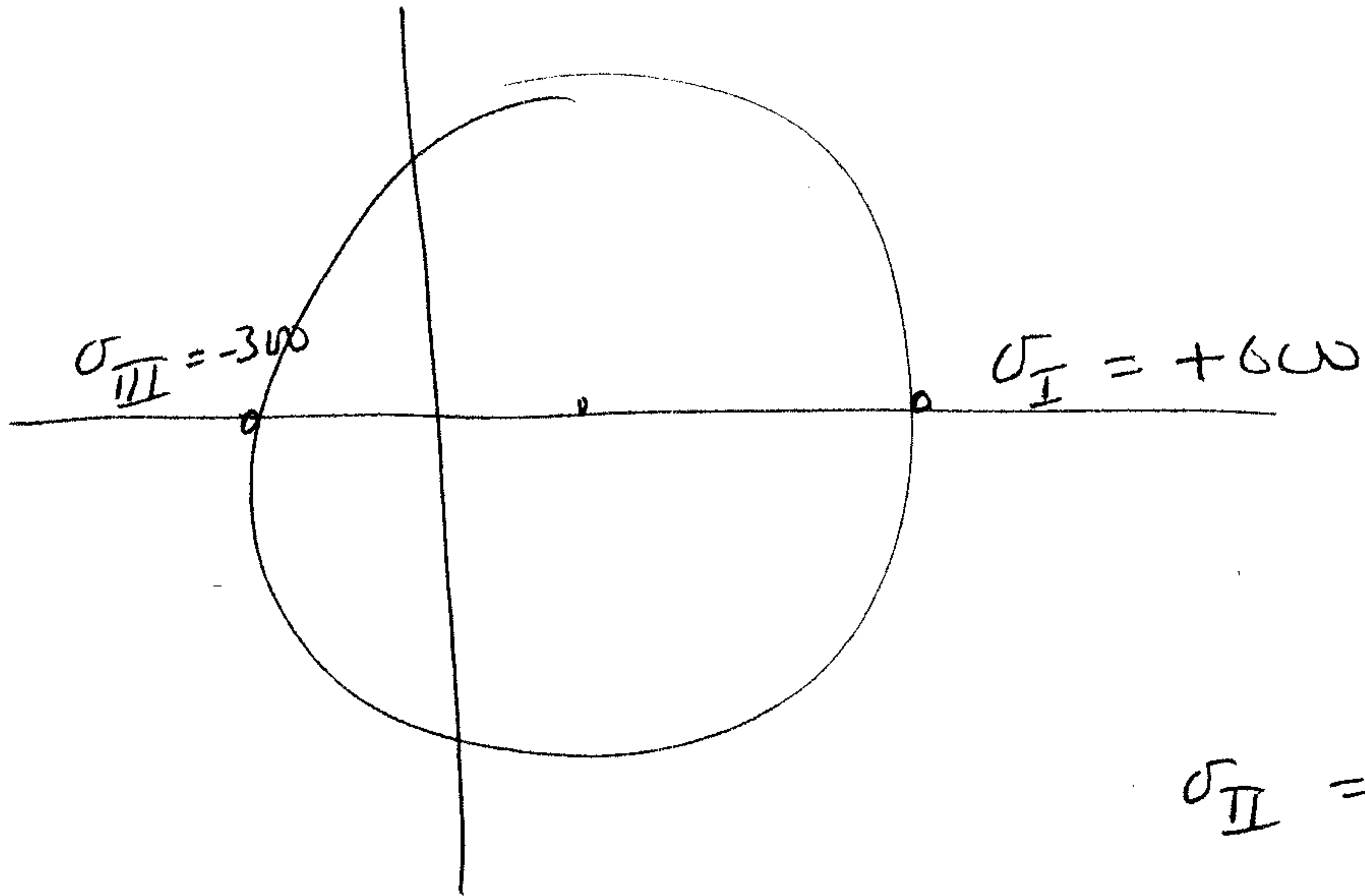
$$\sigma_{II} = 400 - 412.3 = -\frac{12.3}{t}$$

$$\text{Von Mises } (812 - 0)^2 + (0 + 12.3)^2 + (812 + 12.3)^2 = 2 \times 500 \times 10^6$$

$$t = \sqrt{\frac{1.34 \times 10^6 \times (10^3)^2 \times (1.5)^2}{2 \times (500 \times 10^6)^2}} = 2.45 \times 10^3 \Leftarrow$$

$$= 2.45 \text{ mm} \Leftarrow$$

3



$$\sigma_{II} = 0$$

von Mises $(600)^2 + (-300)^2 + (600+300)^2 = \frac{2 \times (500 \times 10^6)^2 t^2}{(10^3)^2 \times (1.5)^2}$

$$t = \sqrt{\frac{1.26 \times 10^6 \times (10^3)^2 \times (1.5)^2}{2 \times (500 \times 10^6)^2}} = 2.38 \text{ mm.} \Leftarrow$$

choose thickest required size 2.45 mm \Leftarrow

M18. Cracks will propagate at highest ~~stress~~ tensile stress

This is in case (2) $\sigma_I = \frac{812.3 \times 10^3}{2.45 \times 10^3} = 331.6 \text{ MPa}$.

$$\sigma_{fc} = \text{Critical crack size} = \frac{1}{\pi} \left(\frac{K_{Ic}}{\sigma} \right)^2 = 1.7 \times 10^{-3}$$

\therefore total crack size = ~~1.7~~ 3.4 mm \Leftarrow

Need to reduce stress to point where critical crack half length = 2.5 mm \Leftarrow .

$$\therefore \sigma = 24 \times 10^6 \sqrt{\pi \times 2.5 \times 10^{-3}} = 270.8 \text{ MPa}$$

$$\therefore \frac{812.3 \times 10^3}{t} = 270.8 \text{ MPa} \Rightarrow t = \frac{812.3 \times 10^3}{270.8 \times 10^6} = 3 \text{ mm} \Leftarrow$$