

## Problem Set No. 1

---

### Problem 1

$$\begin{cases} x_m = \left[ l - \left( \frac{R}{2} - \theta \right) R \right] \sin \theta - R(1 - \cos \theta) \\ y_m = - \left[ l - \left( \frac{R}{2} - \theta \right) R \right] \cos \theta + R \sin \theta \end{cases}$$

$$\begin{cases} \dot{x}_m = \left[ l - \left( \frac{R}{2} - \theta \right) R \right] \cos \theta \cdot \dot{\theta} \\ \dot{y}_m = \left[ l - \left( \frac{R}{2} - \theta \right) R \right] \sin \theta \cdot \dot{\theta} \end{cases}$$

$$\underline{v}_m = \dot{x}_m \hat{e}_x + \dot{y}_m \hat{e}_y$$

$$\underline{f} = |\underline{f}| \left( -\sin \theta \hat{e}_x + \cos \theta \hat{e}_y \right)$$

$$\therefore \underline{f} \cdot \underline{v}_m = 0$$

$\Rightarrow \underline{f}$  does not work  $\Rightarrow$  The only force that does work is

gravitational force which is conservative.  $\Rightarrow$  System is conservative

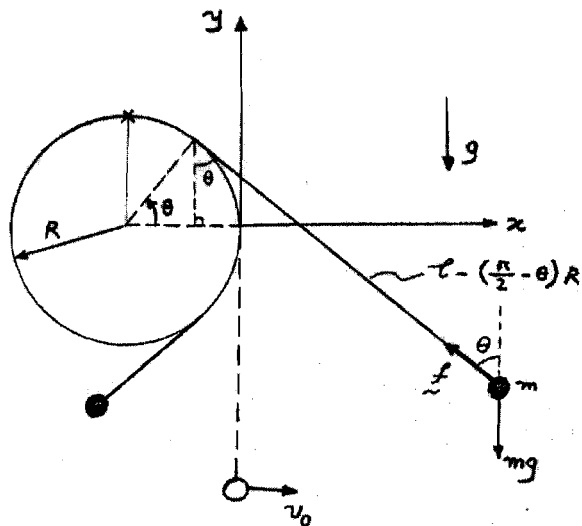
$$\Rightarrow \underline{KE + PE = \text{const.}}$$

$$\text{At } \underline{\theta = 0}, \begin{cases} KE = \frac{1}{2} m v_m^2 = \frac{1}{2} m v_0^2 \\ PE = mg y_m(\theta=0) = -mg \left( l - \frac{R}{2} \right) \end{cases}$$

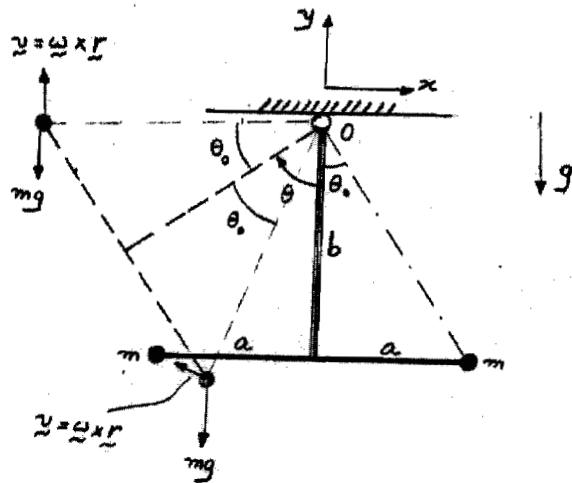
At two extreme deflections,  $v_m = 0 \Rightarrow KE = 0$

$$\therefore PE \Big|_{\theta = \theta_{\max/\min}} = (PE + KE) \Big|_{\theta = 0} = \frac{1}{2} m v_0^2 - mg \left( l - \frac{R}{2} \right) = mg y_m \Big|_{\theta = \theta_{\max/\min}}$$

$$\Rightarrow y_m \Big|_{\theta = \theta_{\max/\min}} = \frac{v_0^2}{2g} - l + \frac{R}{2} = \left[ R \sin \theta - \left[ l - \left( \frac{R}{2} - \theta \right) R \right] \cos \theta \right]_{\theta = \theta_{\max/\min}}$$



## Problem 2



$$\theta_0 = \tan^{-1}\left(\frac{a}{b}\right)$$

$$|\underline{r}| = \sqrt{a^2 + b^2}$$

$$|\underline{\omega}| = \dot{\theta}$$

$$\underline{v}_0 = 0 \quad \Rightarrow \quad \underline{\xi}_0^{\text{ext}} = \frac{d\underline{h}_0}{dt}$$

$$\begin{aligned} \underline{\xi}_0 &= \left\{ mg\sqrt{a^2+b^2} \sin(\theta+\theta_0) + mg\sqrt{a^2+b^2} \sin(\theta-\theta_0) \right\} \hat{\underline{e}}_z \\ &= mg\sqrt{a^2+b^2} (2 \sin\theta \cos\theta_0) \hat{\underline{e}}_z \end{aligned}$$

$$|\underline{v}| = \sqrt{a^2+b^2} \dot{\theta} \quad \rightarrow \quad |\underline{P}| = m\sqrt{a^2+b^2} \dot{\theta}$$

$$\underline{h}_0 = -2\sqrt{a^2+b^2} (m\sqrt{a^2+b^2} \dot{\theta}) \hat{\underline{e}}_z$$

Angular momentum about point 0:

$$2mg\sqrt{a^2+b^2} \sin\theta \cos\theta_0 = -2m(a^2+b^2) \ddot{\theta}$$

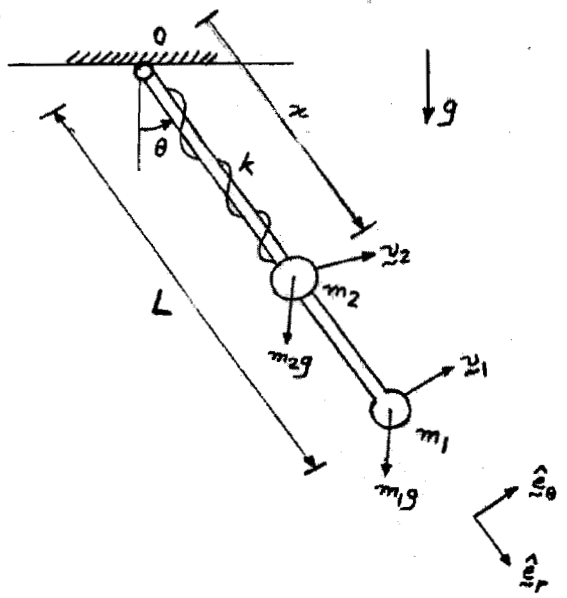
$$\cos\theta_0 = \frac{b}{\sqrt{a^2+b^2}}$$

$$\therefore \ddot{\theta} + \frac{gb}{a^2+b^2} \sin\theta = 0$$

equation of motion for  $\theta(t)$

Problem 3

Assume free length of spring to be  $L_0$ .



a) One needs two coordinates  $\theta$  and  $x$  to describe the motion of the system.  
 $\theta$  and  $x$  are a complete and independent set of generalized coordinates.

b) We need to find two equations:

First, angular momentum about point O for the system:

$$\tau_0^{ext} = \frac{dh_0}{dt}$$

$$\tau_0 = -(m_1 g L \sin\theta + m_2 g x \sin\theta) \hat{e}_z$$

$$v_1 = L \dot{\theta} \hat{e}_\theta \quad \& \quad v_2 = \dot{x} \hat{e}_r + x \dot{\theta} \hat{e}_\theta$$

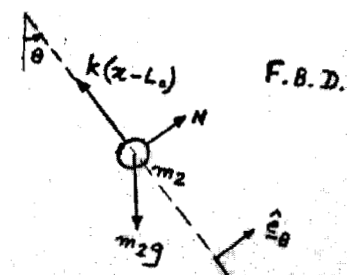
$$h_0 = \sum_{i=1}^2 r_i \times p_i = (L \hat{e}_r) \times (m_1 L \dot{\theta} \hat{e}_\theta) + (x \hat{e}_r) \times [m_2 (\dot{x} \hat{e}_r + x \dot{\theta} \hat{e}_\theta)]$$

$$\Rightarrow h_0 = (m_1 L^2 \dot{\theta} + m_2 x^2 \dot{\theta}) \hat{e}_z$$

$$\therefore -(m_1 g L \sin\theta + m_2 g x \sin\theta) = (m_1 L^2 + m_2 x^2) \ddot{\theta} + 2m_2 x \dot{x} \dot{\theta}$$

$$\Rightarrow (m_1 L^2 + m_2 x^2) \ddot{\theta} + 2m_2 x \dot{x} \dot{\theta} + (m_1 L + m_2 x) g \sin\theta = 0$$

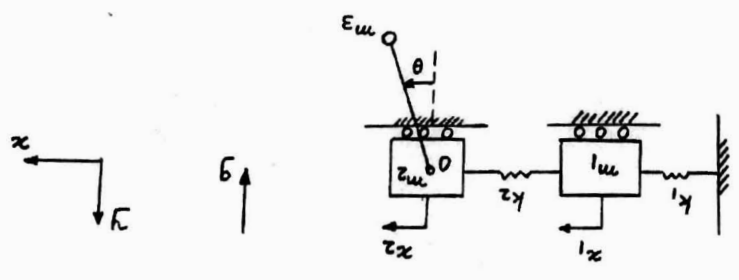
To find the second equation, apply linear momentum in the radial direction for  $m_2$ :  
 $m_2$  is free to slide along the rod so the force  $N$  has no component along the radial direction.





Problem 4

pendulum length = L



Linear momentum in x direction for m1:

$$m_1 \ddot{x}_1 = k_2(x_2 - x_1) - k_1 x_1$$

$$m_1 \ddot{x}_1 = -(k_1 + k_2)x_1 + k_2 x_2$$

Angular momentum about point O:

$$\dot{z}_0 = \frac{dh_0}{dt} + \tilde{u}_0 \times \tilde{P}$$

$$\dot{z}_0 = -m_3 g L \sin \theta \hat{z}$$

$$\tilde{u}_0 = \dot{x}_2 \hat{x}$$

$$\tilde{u}_3 = (\dot{x}_2 + L \dot{\theta} \cos \theta) \hat{x} + (L \dot{\theta} \sin \theta) \hat{y}$$

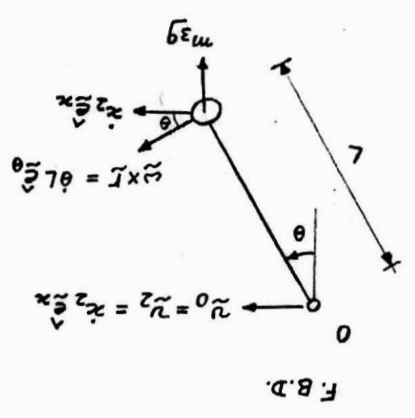
$$\tilde{h}_0 = \tilde{r} \times \tilde{P} = m_3 (\dot{x}_2 + L \dot{\theta} \cos \theta) \hat{x} + (L \dot{\theta} \sin \theta) \hat{y}$$

$$= m_3 (\dot{L}^2 \dot{\theta} + \dot{x}_2 L \cos \theta) \hat{z}$$

$$\tilde{u}_0 \times \tilde{P} = \tilde{u}_0 \times m_3 \tilde{u}_3 = m_3 L \dot{x}_2 \dot{\theta} \sin \theta \hat{z}$$

$$\therefore -m_3 g L \sin \theta = m_3 (\dot{L}^2 \dot{\theta} + \dot{x}_2 L \cos \theta) + m_3 L \dot{x}_2 \dot{\theta} \sin \theta$$

$$L \ddot{\theta} + \dot{x}_2 \dot{\cos} \theta + g \sin \theta = 0$$



F.B.D.

## Problem 4

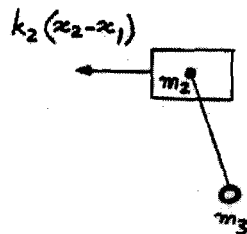
Linear momentum in  $x$  direction for  $m_2$  &  $m_3$ :

$$\underline{a}_2 = \ddot{x}_2 \hat{e}_x$$

$$\underline{a}_3|_x = \ddot{x}_2 + L\ddot{\theta} \cos\theta - L\dot{\theta}^2 \sin\theta$$

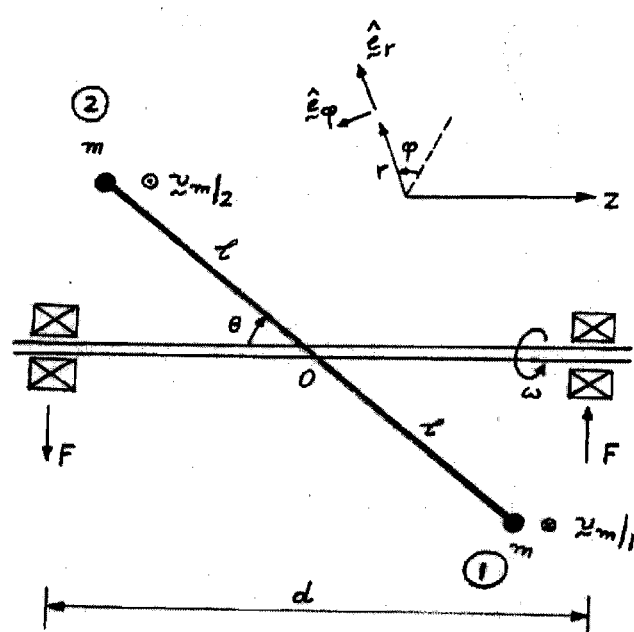
$$-k_2(x_2 - x_1) = m_2 \ddot{x}_2 + m_3 (\ddot{x}_2 + L\ddot{\theta} \cos\theta - L\dot{\theta}^2 \sin\theta)$$

F.B.D.



$$\Rightarrow (m_2 + m_3) \ddot{x}_2 + m_3 L \ddot{\theta} \cos\theta - m_3 L \dot{\theta}^2 \sin\theta + k_2(x_2 - x_1) = 0$$

## Problem 5



$$\begin{aligned} \underline{v}_{m/1} &= \underline{v}_{m/2} = \underline{\omega} \times \underline{r} \\ &= \omega l \sin \theta \hat{\underline{e}}_{\varphi} \end{aligned}$$

Apply angular momentum about point O:

$$\begin{aligned} \underline{h}_O &= \underline{r} \times \underline{P} = (l \cos \theta \hat{\underline{e}}_z + l \sin \theta \hat{\underline{e}}_r) \times m \omega l \sin \theta \hat{\underline{e}}_{\varphi} \\ &\quad + (-l \cos \theta \hat{\underline{e}}_z + l \sin \theta \hat{\underline{e}}_r) \times m \omega l \sin \theta \hat{\underline{e}}_{\varphi} \\ &= -m \omega l^2 \sin \theta \cos \theta \hat{\underline{e}}_r + m \omega l^2 \sin^2 \theta \hat{\underline{e}}_z \\ &\quad + m \omega l^2 \sin \theta \cos \theta \hat{\underline{e}}_r + m \omega l^2 \sin^2 \theta \hat{\underline{e}}_z \end{aligned}$$

$$\frac{d\underline{h}_O}{dt} = -m \omega^2 l^2 \sin \theta \cos \theta \hat{\underline{e}}_{\varphi|1} + m \omega^2 l^2 \sin \theta \cos \theta \hat{\underline{e}}_{\varphi|2}$$

(Note that  $\frac{d\hat{\underline{e}}_r}{dt} = \omega \hat{\underline{e}}_{\varphi}$ ;  $\hat{\underline{e}}_{\varphi|1}$  and  $\hat{\underline{e}}_{\varphi|2}$  are in opposite directions)

$$\tau_0^{\text{ext}} = Fd$$

$$\tau_0^{\text{ext}} = \frac{d\underline{h}_O}{dt} \Rightarrow Fd = 2m\omega^2 l^2 \sin \theta \cos \theta = m\omega^2 l^2 \sin 2\theta$$

$$\Rightarrow \underline{F = \frac{m\omega^2 l^2 \sin 2\theta}{d}} \Rightarrow \underline{F_{\text{max}} = \frac{m\omega^2 l^2}{d}} \text{ for the case of } \underline{\theta = 45^\circ}$$

Note that force  $F$  changes direction but is always in the plane of the shaft and the rods.