Simulating Natural Ventilation Using Boussinesq Approximations (ANSYS Fluent)

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Buoyancy Ventilation In Building

- 1. Common in building indoor ventilation;
- 2. Small magnitude comparing to wind driven ventilation;
- 3. Difficult to identify in a mixed ventilation mode.

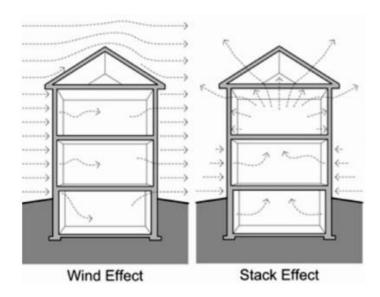
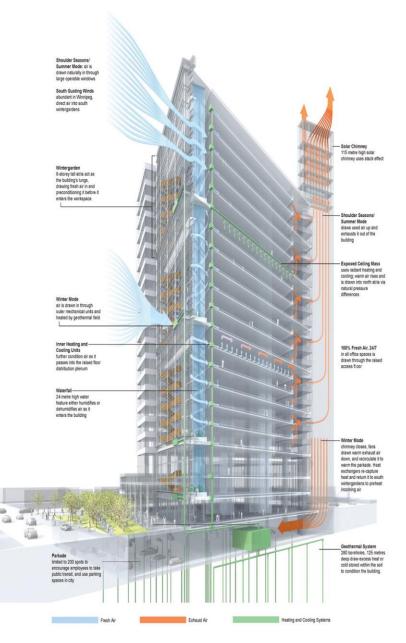


Image from: https://energy.ces.ncsu.edu/stack-effect-defined/



Manitoba Hydro Place, Canada

Image from: https://www.hydro.mb.ca/corporate/history/mh_place_design_and_construction/

Boussinesq Approximation

General NS Equations

$$\rho \left(\frac{\partial \mathbf{u}}{\partial t} + \mathbf{u} \cdot \nabla \mathbf{u} \right) = -\nabla p + \nabla \cdot \left(\mu (\nabla \mathbf{u} + (\nabla \mathbf{u})^T) - \frac{2}{3} \mu (\nabla \cdot \mathbf{u}) \mathbf{I} \right) + \rho \mathbf{g}$$

$$\frac{1}{\rho} \frac{D\rho}{Dt} + \nabla \cdot \mathbf{u} = 0$$

Boussinesq Approximation: "Density variation is only important in the buoyancy term, and can be neglected in the rest of the equation" (reference: https://www.comsol.com/multiphysics/boussinesq-approximation)

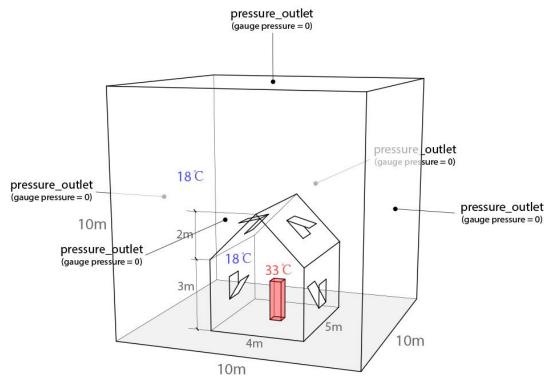
$$\rho_0 \left(\frac{\partial \mathbf{u}}{\partial t} + \mathbf{u} \cdot \nabla \mathbf{u} \right) = -\nabla p + \mu \nabla^2 \mathbf{u} + \rho \mathbf{g} \qquad (**\rho 0 \text{ is constant})$$

$$\longrightarrow \qquad \rho_0 \left(\frac{\partial \mathbf{u}}{\partial t} + \mathbf{u} \cdot \nabla \mathbf{u} \right) = -\nabla p + \mu \nabla^2 \mathbf{u} + (\rho_0 + \Delta \rho) \mathbf{g} \qquad \Delta \rho = \rho - \rho_0$$

$$(\rho-\rho_0)\mathbf{g}=-\rho_0\beta(T-T_0)\mathbf{g}$$
 For ideal gas: $\beta=1/T_0$

$$\rho_0 \left(\frac{\partial \mathbf{u}}{\partial t} + \mathbf{u} \cdot \nabla \mathbf{u} \right) = -\nabla p + \mu \nabla^2 \mathbf{u} + + \rho_0 \mathbf{g} - \rho_0 (T - T_0) / T_0 \mathbf{g}$$

Case Set-up



Geometry:

3D house with hopper windows and roof hatches, one human present in the center of the space.

Physical Environment:

Ambient temperature =18C;

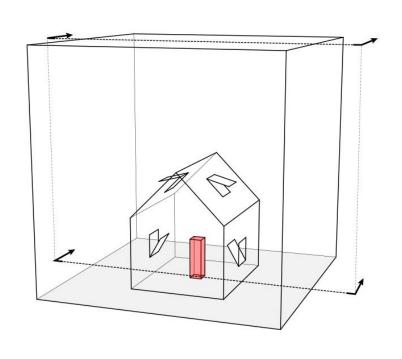
Human skin temperature = 33C.

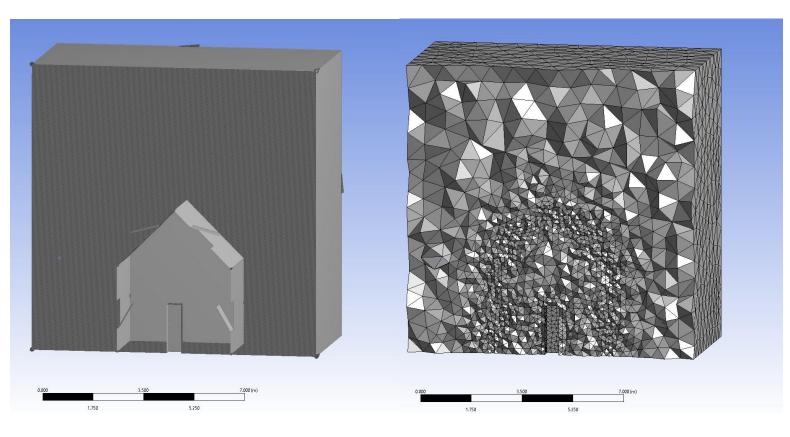
No velocity inlet (no external force triggering advection).

All open boundary are set as "pressure outlet", with pressure = 0 (back flow will happen)

Meshing

Non-structured Mesh with refined grid size adjacent to walls

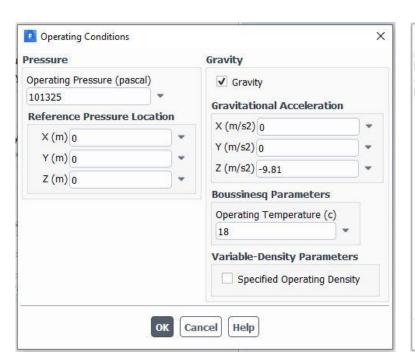




Numerical Parameter Setting

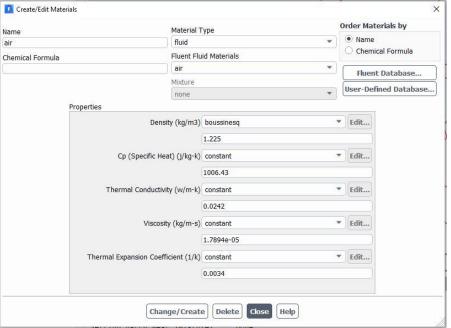
Operating Conditions:

1. Gravity 2. Operating Temperature

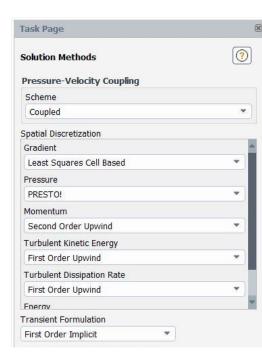


Materials air:

1. Density 2. Thermal Expansion Coefficient



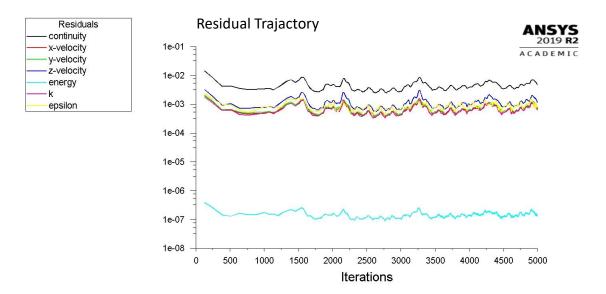
Solution Methods: Pressure: PRESTO!

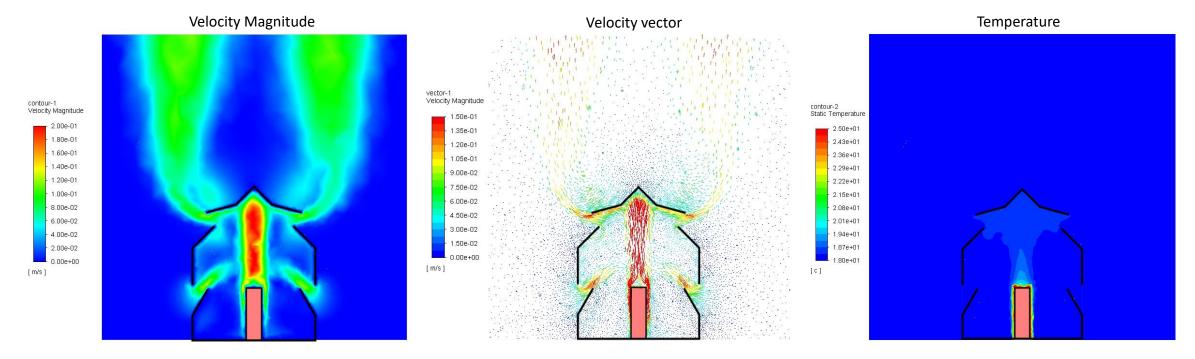


[&]quot;PRESTO! (PREssure STaggering Option) scheme uses the discrete continuity balance for a "staggered" control volume about the face to compute the "staggered" (i.e., face) pressure." (Reference: https://www.afs.enea.it/project/neptunius/docs/fluent/html/th/node371.htm)

Steady State Simulation Result

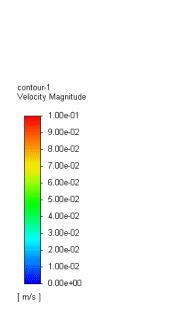
Total No. of iterations = 5000

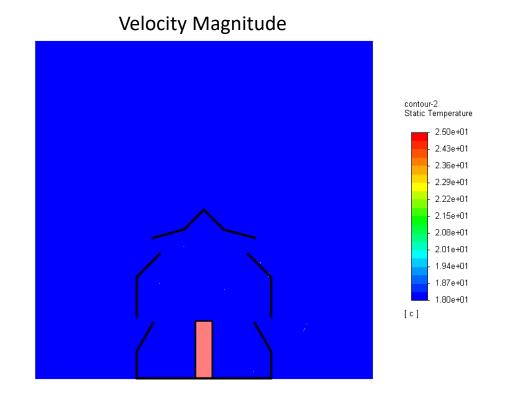




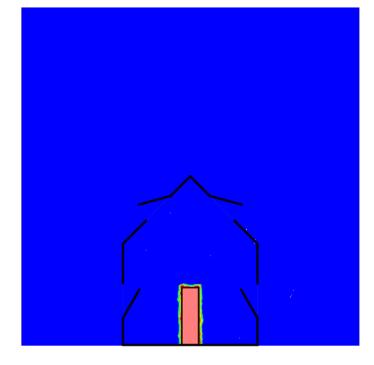
Transient Simulation Result

Time step = 0.1s; Total No. of time steps = 1000 Max CFL ~ 1 (can be greater than 1)

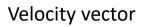


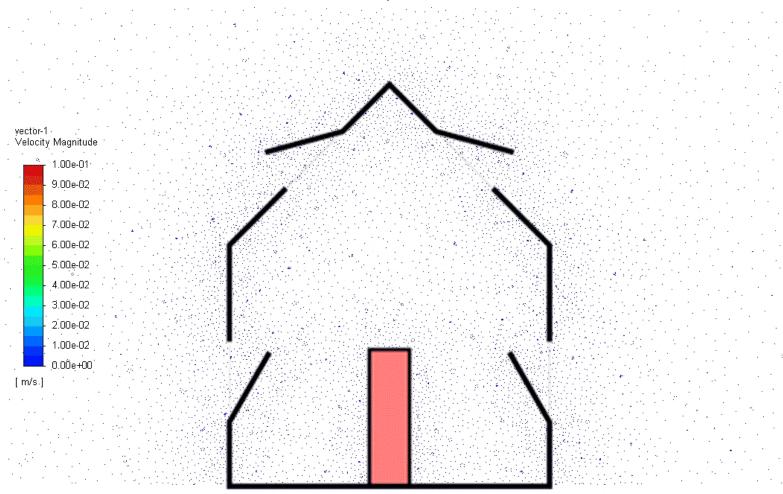






Transient Simulation Result





Conclusion & Discussion

- 1. Boussinesq approximation is robust in calculating weak buoyancy driven flow like building interior stack effect;
- 2. In this situation, pressure outlet boundary condition with gauge pressure=0 can be used in absence of a velocity/ pressure inlet (back flow will happen);
- 3. The residual for most of the variables fluctuate around 10^-3 (continuity ~ 10^-2), further improvement could be tested:
- a. Reduce dt and dx;
- b. Optimize mesh (e.g. inflation layers at walls)