

**16.810 (16.682)**

**Engineering Design and Rapid Prototyping**

# **16.810 Finite Element Method**

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Instructor(s)

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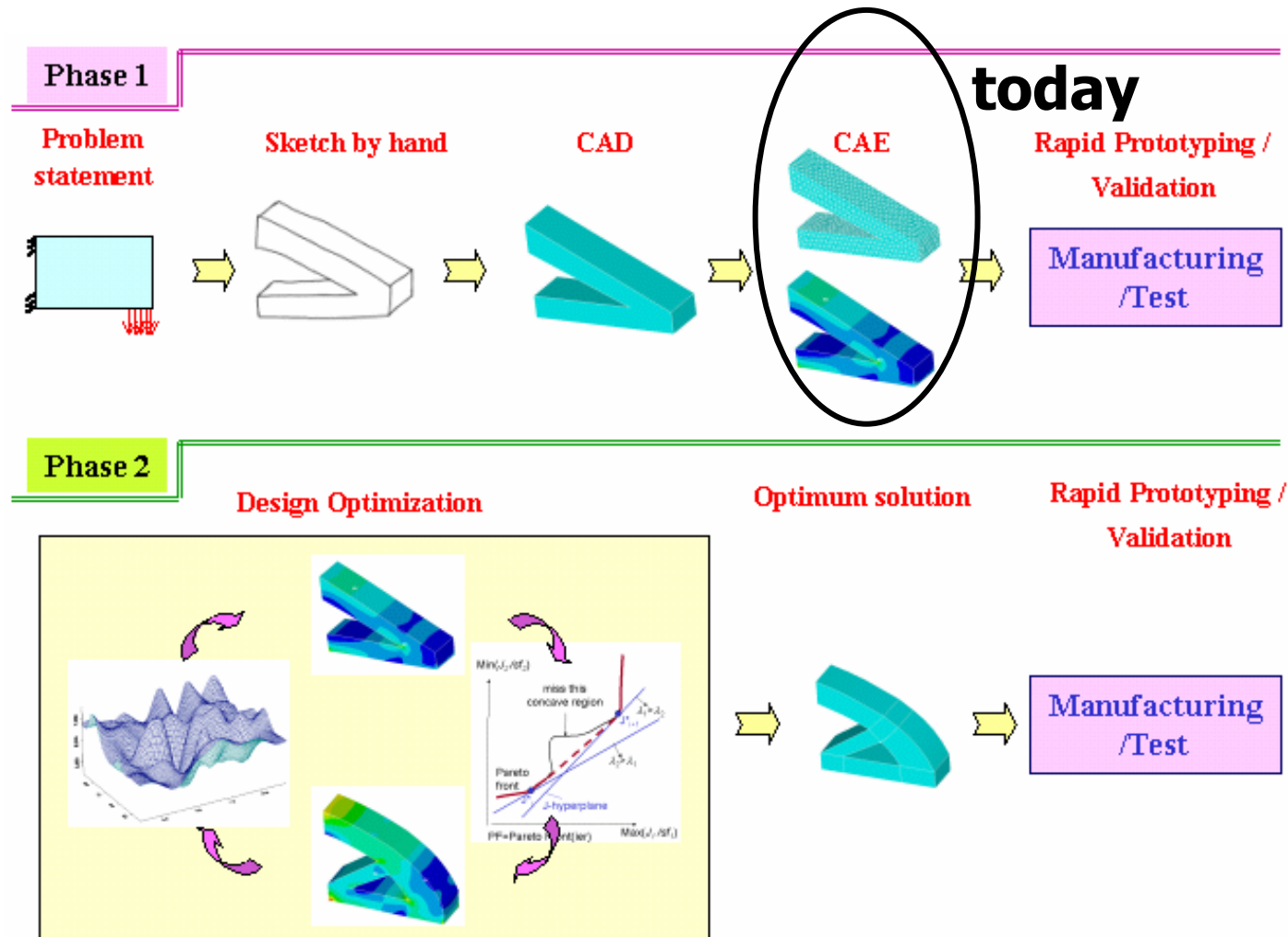
January 12, 2004

# 16.810 Plan for Today

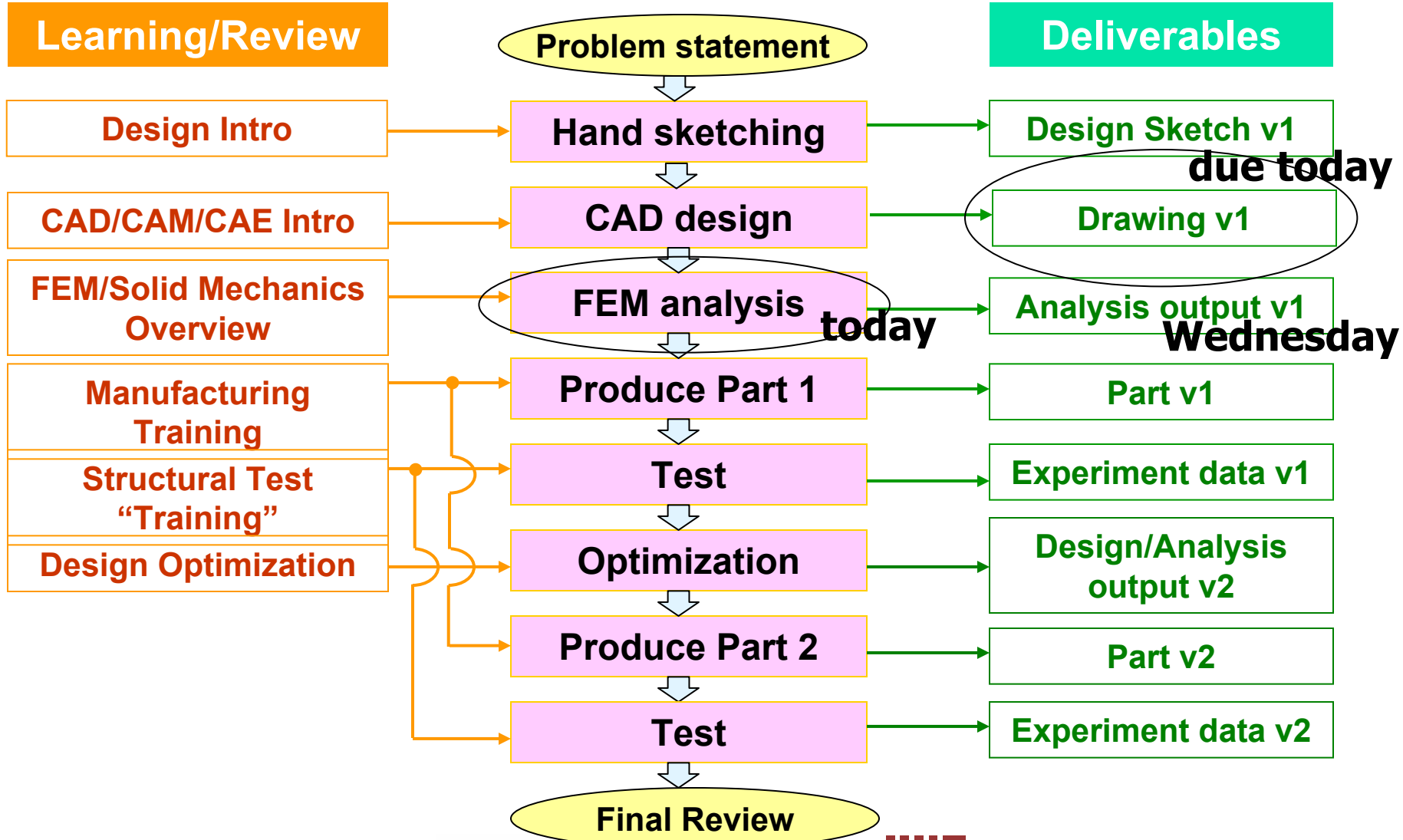
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- **FEM Lecture (ca. 50 min)**
  - **FEM fundamental concepts, analysis procedure**
  - **Errors, Mistakes, and Accuracy**
- **Cosmos Introduction (ca. 30 min)**
  - **Follow along step-by-step**
- **Conduct FEA of your part (ca. 90 min)**
  - **Work in teams of two**
  - **First conduct an analysis of your CAD design**
  - **You are free to make modifications to your original model**

# Course Concept



# Course Flow Diagram



**Finite Element Method**

**Boundary Element Method**

**Finite Difference Method**

**Finite Volume Method**

**Meshless Method**

# What is the FEM?

**FEM: Method for numerical solution of field problems.**

## Description

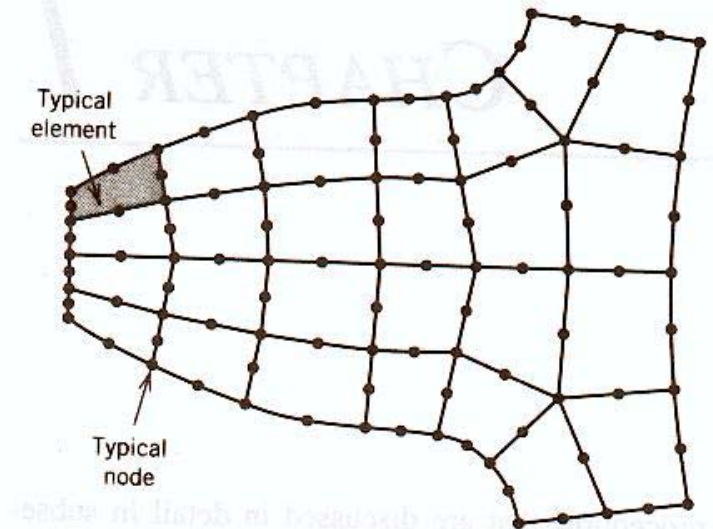
- FEM cuts a structure into several elements (pieces of the structure).
- Then reconnects elements at “nodes” as if nodes were pins or drops of glue that hold elements together.
- This process results in a set of simultaneous algebraic equations.

## Number of degrees-of-freedom (DOF)

**Continuum:** Infinite

**FEM:** Finite

(This is the origin of the name,  
Finite Element Method)



# Fundamental Concepts (1)

Many engineering phenomena can be expressed by  
“**governing equations**” and “**boundary conditions**”

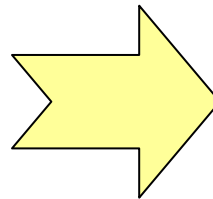
Elastic problems

Thermal problems

Fluid flow

Electrostatics

etc.



Governing Equation  
(Differential equation)

$$L(\phi) + f = 0$$

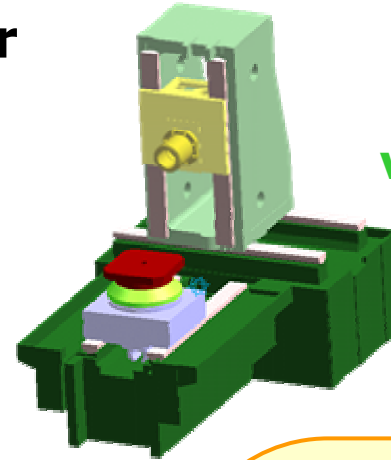


Boundary Conditions

$$B(\phi) + g = 0$$

## Example: Vertical machining center

Elastic deformation  
Thermal behavior  
etc.

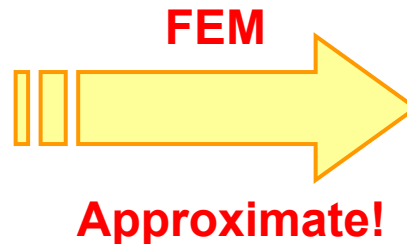


Geometry is  
very complex!

Governing  
Equation:  $L(\phi) + f = 0$

Boundary  
Conditions:  $B(\phi) + g = 0$

You know all the equations, but  
you cannot solve it by hand



A set of simultaneous  
algebraic equations

$$[\mathbf{K}] \{\mathbf{u}\} = \{\mathbf{F}\}$$



# Fundamental Concepts (3)

$$[\mathbf{K}] \{\mathbf{u}\} = \{\mathbf{F}\} \quad \Rightarrow \quad \{\mathbf{u}\} = [\mathbf{K}]^{-1} \{\mathbf{F}\}$$

Property                      Behavior                      Action

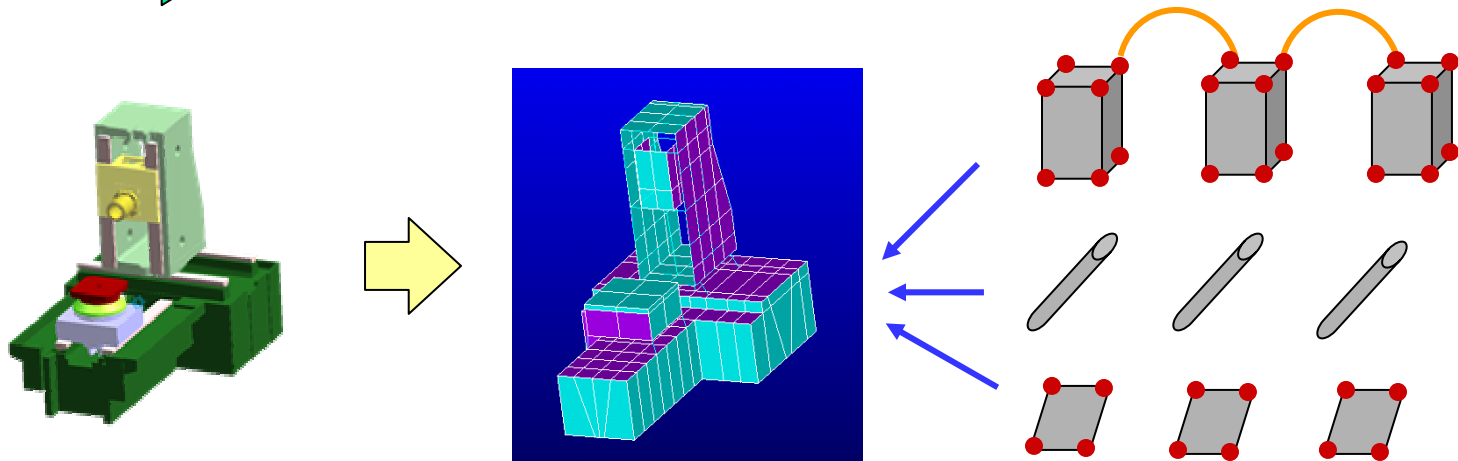
**Unknown**

	Property $[\mathbf{K}]$	Behavior $\{\mathbf{u}\}$	Action $\{\mathbf{F}\}$
<b>Elastic</b>	stiffness	displacement	force
<b>Thermal</b>	conductivity	temperature	heat source
<b>Fluid</b>	viscosity	velocity	body force
<b>Electrostatic</b>	dielectric permittivity	electric potential	charge

# Fundamental Concepts (4)

It is very difficult to make the algebraic equations for the entire domain

- ➔ Divide the domain into a number of small, simple elements
- ➔ A field quantity is interpolated by a polynomial over an element
- ➔ Adjacent elements share the DOF at connecting nodes

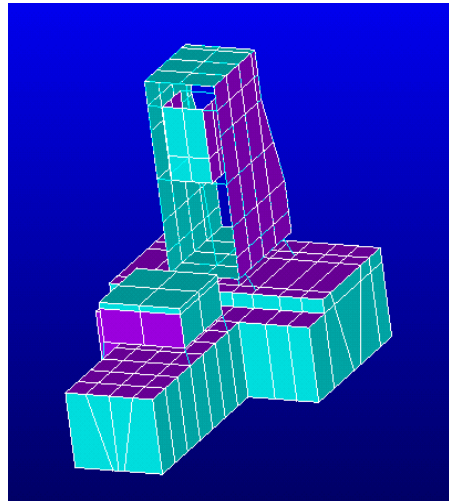


**Finite element: Small piece of structure**

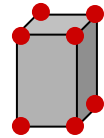
# Fundamental Concepts (5)

Obtain the algebraic equations for each element (this is easy!)

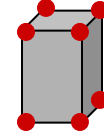
➔ Put all the element equations together



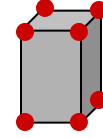
$$[\mathbf{K}^E]\{\mathbf{u}^E\} = \{\mathbf{F}^E\}$$



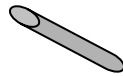
$$[\mathbf{K}^E]\{\mathbf{u}^E\} = \{\mathbf{F}^E\}$$



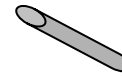
$$[\mathbf{K}^E]\{\mathbf{u}^E\} = \{\mathbf{F}^E\}$$



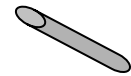
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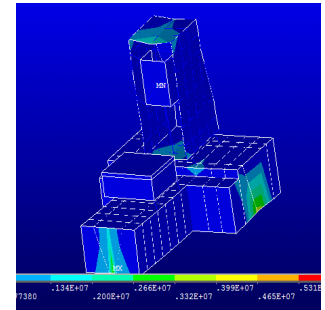
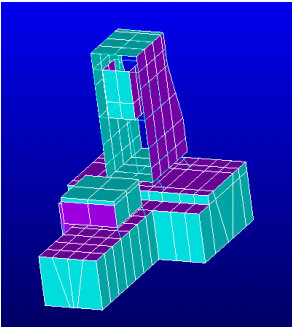
$$[\mathbf{K}^E]\{\mathbf{u}^E\} = \{\mathbf{F}^E\}$$



$$[\mathbf{K}]\{\mathbf{u}\} = \{\mathbf{F}\}$$

Solve the equations, obtaining unknown variables at nodes.

$$[\mathbf{K}] \{\mathbf{u}\} = \{\mathbf{F}\} \quad \square \square \square \square \square \square \square \square \square \rightarrow \{\mathbf{u}\} = [\mathbf{K}]^{-1} \{\mathbf{F}\}$$

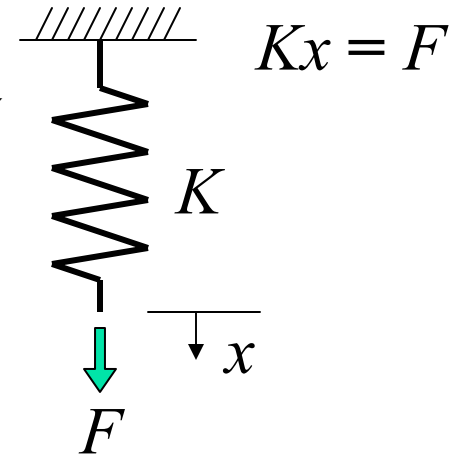


- FEM uses the concept of piecewise polynomial interpolation.
- By connecting elements together, the field quantity becomes interpolated over the entire structure in piecewise fashion.
- A set of simultaneous algebraic equations at nodes.

$$[\mathbf{K}] \{\mathbf{u}\} = \{\mathbf{F}\}$$

Property      Behavior      Action

*K*: Stiffness matrix  
*x*: Displacement  
*F*: Load



- The term **finite element** was first coined by Clough in **1960**. In the early 1960s, engineers used the method for approximate solutions of problems in stress analysis, fluid flow, heat transfer, and other areas.
- The first book on the FEM by **Zienkiewicz** and Chung was published in 1967.
- In the late 1960s and early 1970s, the FEM was applied to a wide variety of engineering problems.
- Most commercial **FEM software packages** originated in the **1970s**. (Abaqus, Adina, Ansys, etc.)
- Klaus-Jürgen Bathe in ME at MIT

## Can readily handle very complex geometry:

- The heart and power of the FEM

## Can handle a wide variety of engineering problems

- Solid mechanics
- Dynamics
- Heat problems
- Fluids
- Electrostatic problems

## Can handle complex restraints

- Indeterminate structures can be solved.

## Can handle complex loading

- Nodal load (point loads)
- Element load (pressure, thermal, inertial forces)
- Time or frequency dependent loading

A general **closed-form solution**, which would permit one to examine system response to changes in various parameters, **is not produced**.

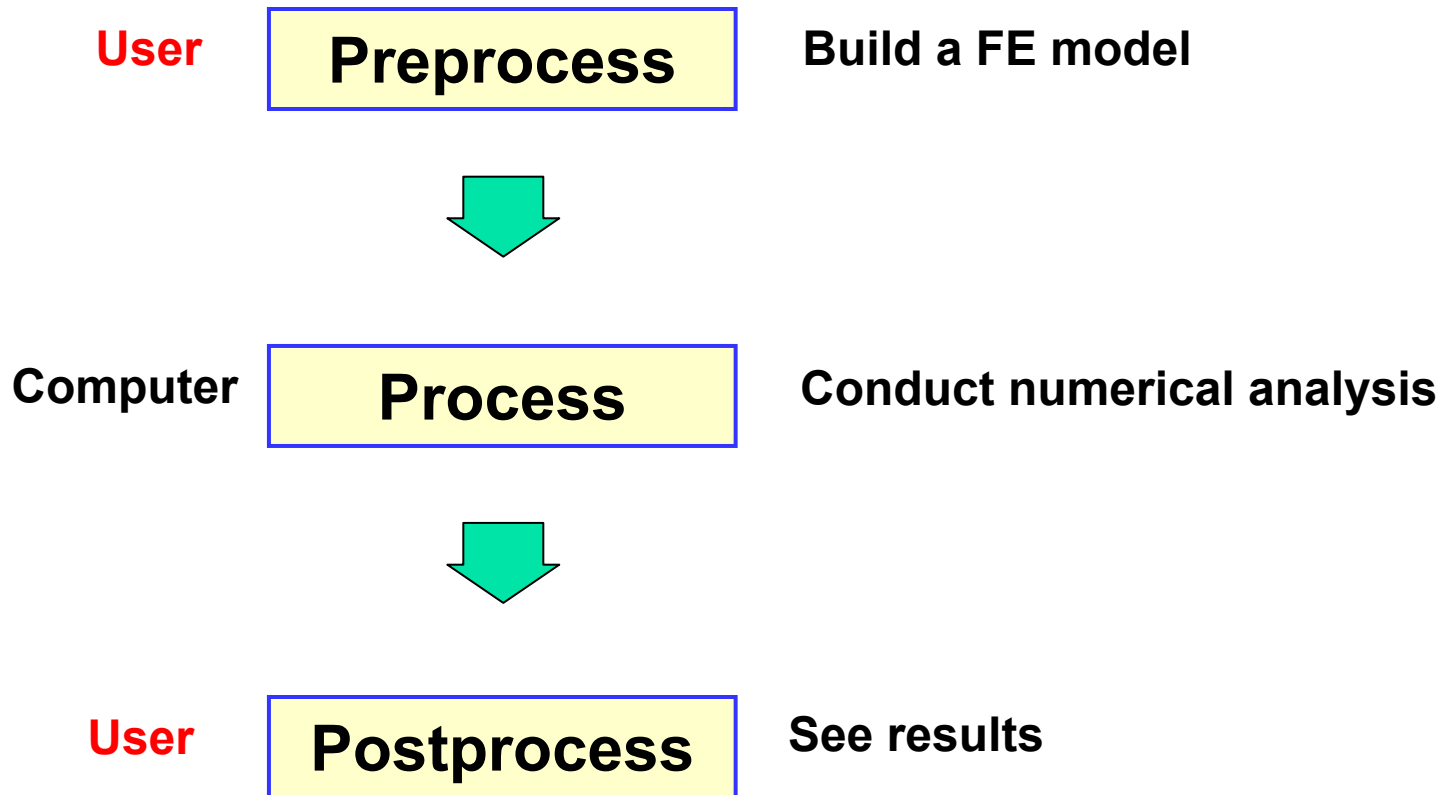
The FEM obtains only **"approximate"** solutions.

The FEM has **"inherent" errors**.

**Mistakes by users** can be fatal.

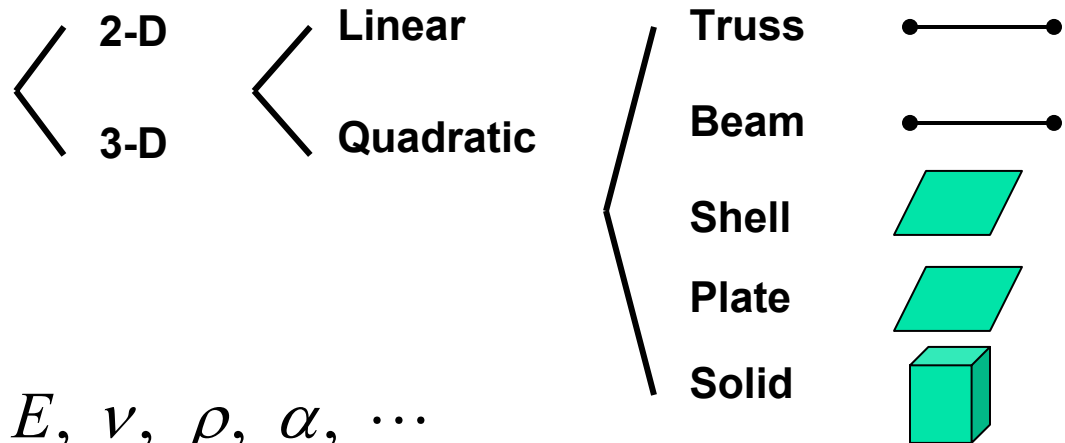


# Typical FEA Procedure by Commercial Software

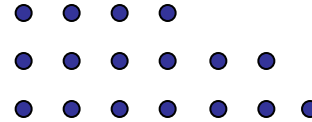


**[1] Select analysis type**

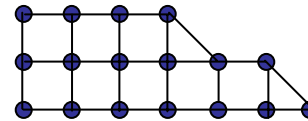
- Structural Static Analysis
- Modal Analysis
- Transient Dynamic Analysis
- Buckling Analysis
- Contact
- Steady-state Thermal Analysis
- Transient Thermal Analysis

**[2] Select element type****[3] Material properties**
 $E, \nu, \rho, \alpha, \dots$

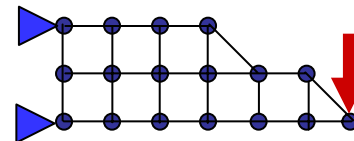
[4] Make nodes



[5] Build elements by assigning connectivity



[6] Apply boundary conditions and loads



## [7] Process

- Solve the boundary value problem



## [8] Postprocess

- See the results

Displacement

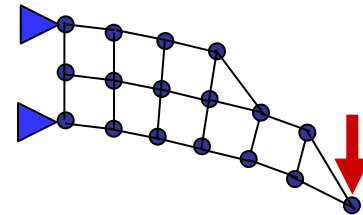
Stress

Strain

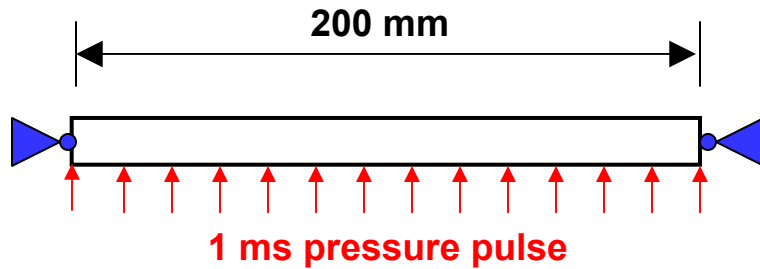
Natural frequency

Temperature

Time history



# Responsibility of the user



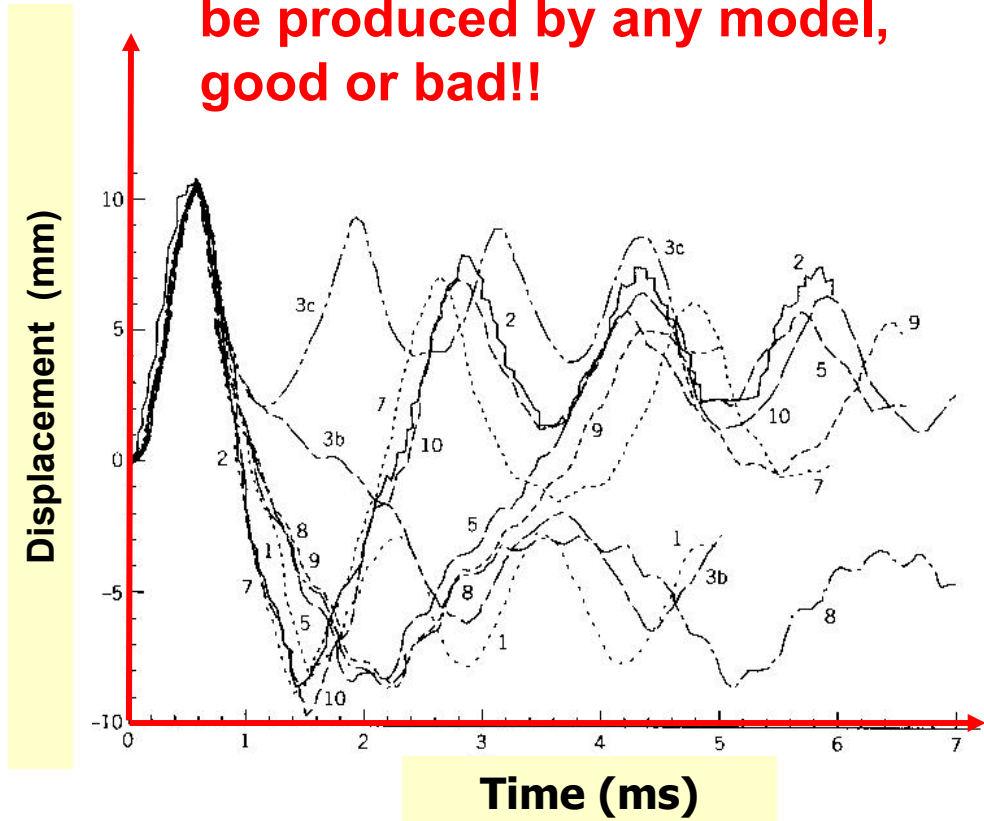
**BC:** Hinged supports

**Load:** Pressure pulse

**Unknown:** Lateral mid point displacement in the time domain

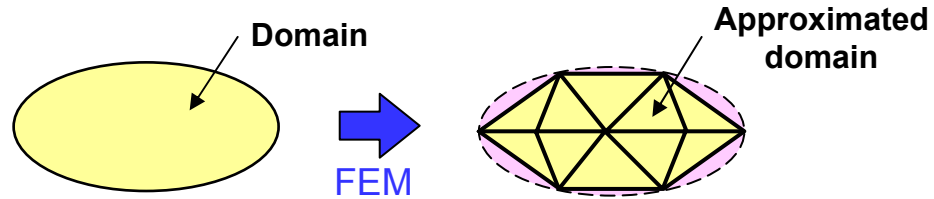
Results obtained from ten reputable FEM codes and by users regarded as expert.\*

**Fancy, colorful contours can be produced by any model, good or bad!!**

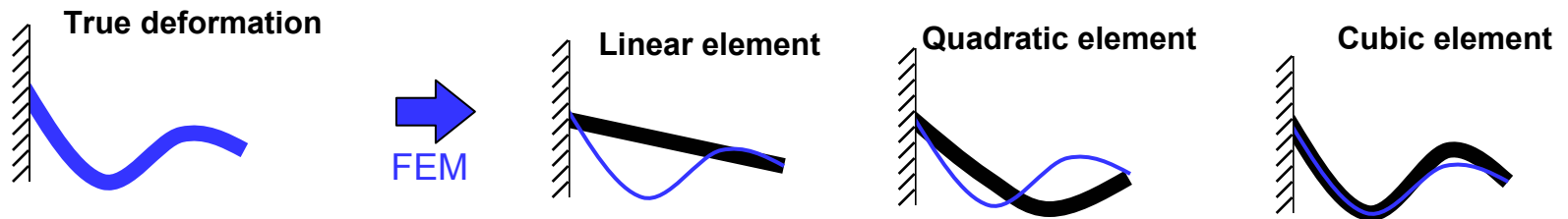


\* R. D. Cook, *Finite Element Modeling for Stress Analysis*, John Wiley & Sons, 1995

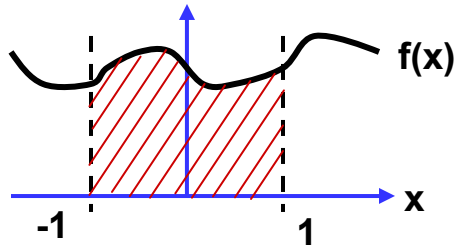
- **Geometry is simplified.**



- Field quantity is assumed to be a **polynomial** over an element. (which is not true)



- Use very **simple integration** techniques (Gauss Quadrature)



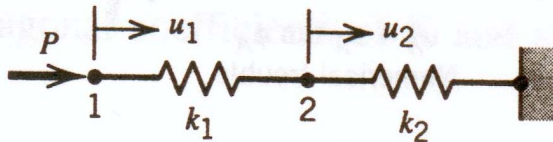
$$\text{Area: } \int_{-1}^1 f(x) dx \approx f\left(\frac{1}{\sqrt{3}}\right) + f\left(-\frac{1}{\sqrt{3}}\right)$$

- The computer carries only a **finite number of digits**.

e.g.)  $\sqrt{2} = 1.41421356$ ,  $\pi = 3.14159265$

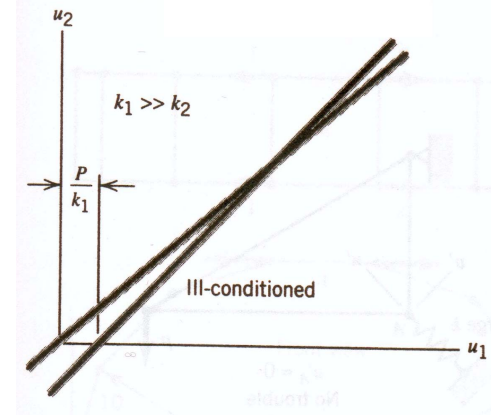
### - Numerical Difficulties

e.g.) Very large stiffness difference



$$k_1 \gg k_2, \quad k_2 \approx 0$$

$$[(k_1 + k_2) - k_2]u_2 = P \Rightarrow u_2 = \frac{P}{k_2} \approx \frac{P}{0}$$



- Elements are of the wrong type  
e.g) Shell elements are used where solid elements are needed
- Distorted elements
- Supports are insufficient to prevent all rigid-body motions
- Inconsistent units (e.g.  $E=200$  GPa, Force = 100 lbs)
- Too large stiffness differences → Numerical difficulties



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Glaucio H. Paulino, *Introduction to FEM (History, Advantages and Disadvantages)*, <http://cee.ce.uiuc.edu/paulino>

Robert Cook et al., *Concepts and Applications of Finite Element Analysis*, John Wiley & Sons, 1989

Robert Cook, *Finite Element Modeling For Stress Analysis*, John Wiley & Sons, 1995

*Introduction to Finite Element Method*, <http://210.17.155.47> (in Korean)

J. Tinsley Oden et al., *Finite Elements – An Introduction*, Prentice Hall, 1981